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**MODELING THE IMPACT RESPONSE OF BULK CUSHIONING  
MATERIALS**

**Don McDaniel**

**Army Missile Research, Development and Engineering  
Laboratory  
Redstone Arsenal, Alabama**

**9 May 1975**

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TECHNICAL REPORT RD-75-16

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OF BULK CUSHIONING MATERIALS

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US Army Missile Research, Development and Engineering Laboratory  
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REPORT DOCUMENTATION PAGE		READ INSTRUCTIONS BEFORE COMPLETING FORM
1. REPORT NUMBER RD-75-16	2. GOVT ACCESSION NO.	3. RECIPIENT'S CATALOG NUMBER <i>A-A 911 232</i>
4. TITLE (and Subtitle) MODELING THE IMPACT RESPONSE OF BULK CUSHIONING MATERIALS		5. TYPE OF REPORT & PERIOD COVERED Technical Report
		6. PERFORMING ORG. REPORT NUMBER
7. AUTHOR(s) Don McDaniel		8. CONTRACT OR GRANT NUMBER(s)
9. PERFORMING ORGANIZATION NAME AND ADDRESS Aeroballistics Directorate US Army Missile Res, Dev and Engineering Lab US Army Missile Command Redstone Arsenal, Alabama 35809		10. PROGRAM ELEMENT, PROJECT, TASK AREA & WORK UNIT NUMBERS
11. CONTROLLING OFFICE NAME AND ADDRESS		12. REPORT DATE 9 May 1975
		13. NUMBER OF PAGES
14. MONITORING AGENCY NAME & ADDRESS(if different from Controlling Office)		15. SECURITY CLASS. (of this report) UNCLASSIFIED
		15a. DECLASSIFICATION/DOWNGRADING SCHEDULE
16. DISTRIBUTION STATEMENT (of this Report) Approved for public release; distribution unlimited.		
17. DISTRIBUTION STATEMENT (of the abstract entered in Block 20, if different from Report)		
18. SUPPLEMENTARY NOTES <b>PRICES SUBJECT TO CHANGE</b>		
19. KEY WORDS (Continue on reverse side if necessary and identify by block number) Cushioning Materials Shock Mitigation Viscoelastic Properties Impact Response		
20. ABSTRACT (Continue on reverse side if necessary and identify by block number) This report deals with the utilization of bulk cushioning materials in shock mitigation systems. The current techniques used in designing bulk cushioning systems are discussed, and an improved technique is presented through the development of a mathematical model of impact response which is based on the viscoelastic properties of bulk cushioning materials. A General Model of impact response which is applicable to all types of bulk cushioning materials and predicts g-level response in terms of drop height, static stress,		

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**Block 20 Abstract continued**

thickness of cushion, and temperature is developed. A technique for determining the optimal cushioning system design is developed, and examples of the use of the technique are presented for a cross-linked polyethylene foam cushioning material.

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## Chapter I

### INTRODUCTION

A cushioning system is an interface device which isolates a protected item from the shock loads which develop in its external environment. A cushioning system is required when the protected item cannot survive the shock loads imposed by the external environment, unless these loads are attenuated by some form of cushioning system. The design of a cushioning system, therefore, involves stipulations concerning both the item to be protected and the external environment. These two stipulations of how much shock an item can survive and how much shock it is expected to experience in its environment are questions that must be answered in the design of any cushioning system, commercial or military. It is the concern for extreme external environments that is emphasized most in the military designs.

#### The Military Environment

A cushioning system designed for military use, such as a shock isolating component of a missile or missile system, is required to perform its basic functions throughout its operating life. Current military policy, based on the need for worldwide deployment, dictates that military materiel be capable of withstanding the rigors of deployment on a worldwide basis. Consequently, military materiel, including

cushioning systems, must withstand all the environments to which they are exposed and continue to operate satisfactorily if the materiel is to be considered acceptable.

The environment in which military materiel operates is frequently quite severe and may produce substantial damage. Military operations encompass all the geographic regions and the peculiar aspects of their environments. Each environment introduces hazards peculiar to geographic zones which materiel will encounter when deployed.

The tropics present a hot, humid atmosphere, subjecting materiel to what is commonly referred to as "jungle-rot". Excessive rainfall, high humidity, heat, and fungus, in addition to the prevalence of vermin, insects, and reptiles present problems relating to materiel protection and to the safety of personnel.

The arctic regions subject materiel to a very cold atmosphere. Temperatures of -65°F are common and temperatures of -85°F have been recorded in underground ammunition storage "igloos". The physical characteristics of components, in particular resilient cushioning systems, are radically altered under these conditions and their operational integrity is jeopardized.

Also, high-speed, fast-climbing aircraft introduce rapid and severe variations in temperature and pressure, and provisions must be made to withstand the effects of these changes, when required.

#### Cushioning Systems in the Military

In evaluating the environmental factors, the military has established certain extreme limits on the particular conditions that have become accepted as a basis for worldwide environments. Temperature

extremes, which produce dramatic changes in the resiliency of cushioning materials, are important in cushion system design. The worldwide temperature extremes are generally defined as -65°F and 160°F; cushioning systems in military applications are expected to perform their shock interface function under these adverse environments.

Many of the military cushioning systems use some form of bulk cushioning material (e.g. plastic foam) as the cushioning system. This provides a low-cost, lightweight cushioning system that is easily incorporated into the design; however, designers have some reservations concerning the use of bulk cushioning systems in the military. One limiting factor is the difficulty the designer has in predicting the response of bulk cushioning materials when subjected to the extreme environmental conditions encountered under military deployment. The temperature sensitivity of these cushioning materials, which causes variations in the impact response, is also a primary concern. It was shown in a recent study of thermoplastic foam cushioning systems that temperature had a significant effect on cushion system response [1]. Therefore, temperature effects must be factored into the design of cushion systems using bulk cushioning materials.

To fully utilize bulk cushioning systems, designers require sufficient information to accurately predict response variations. The current practice is to provide the designer with cushioning data for each type and thickness of cushioning material. These data are provided in the form of dynamic cushioning curves as prescribed in the current theory of cushioning design (Chapter II). For any particular shock isolation system design program, the designer is generally given a maximum allowable fragility level which the protected item is permitted

to experience. Also the particular organization involved will have an established testing policy defining appropriate impact tests. These parameters provide the basis for the design of the shock mitigation system. Given a satisfactory prediction of impact response for various candidate cushioning materials, a designer can use this prediction to select the appropriate cushioning scheme to meet the particular design requirements.

#### Research Objective

The research objective of this study is to develop a reliable impact response model that accurately predicts the dynamic cushioning performance of bulk cushioning systems. A secondary objective is to develop an optimization technique that utilizes the model in determining an optimal cushioning system design.

In answering the research objective a systematic study of background material was conducted. The current theory of cushioning design is discussed in Chapter I. The ingredients of the General Model of impact response are identified in Chapter III and the basis of the underlying structure of the developed model is based upon viscoelastic theory.

The modeling process and the analysis techniques used to formulate the General Model of impact response are also discussed in Chapter III. Chapter IV presents the two finalized models, a General Model of impact response and the Minicel Model, the model of impact response of a particular cushioning material (Minicel is a  $2 \text{ lb}/\text{ft}^3$  cross-linked polyethylene foam manufactured by Hercules, Inc).

The validity of the models are demonstrated in Chapter V through a systematic series of tests and analysis. Finally, in answer to the secondary objective of the research, an optimization technique is presented in Chapter VI that generates the optimal bulk cushion design. The optimization technique utilizes the predictive capability of the General Model of impact response to determine optimality and provide, as output, a set of dynamic cushioning curves at the optimal conditions.

## Chapter II

### CUSHIONING DESIGN THEORY

Anything that is subject to movement is subject to mechanical damage due to shock. Shock may be defined as a sudden change in direction or velocity of the motion of a body. The magnitude or intensity of shock is expressed in G's, which is defined in terms of the time rate of change of the velocity (acceleration), and is measured in feet/second/second. Mathematically,  $G's = a/g$  where  $a$  is the acceleration experienced by the body and  $g$  is the acceleration due to gravity (32.2 ft/sec/sec).

A given body in a static condition has one gravity unit of G acting on it and, therefore, exerts one G upon its support. If this body is raised and allowed to fall freely, it will accelerate in its fall, due to the force of gravity, until it collides with its support or the earth. The stop causes the body to experience a sudden deceleration that can be expressed in terms of G's. If the body experiences, upon impact, a deceleration of 20 times that of its static condition, it is said to have experienced 20 G's of deceleration. A jet pilot experiences such a condition when he pulls out of a dive. He must not exceed his G limit or he will black out, and if the aircraft is not designed and stressed to withstand high G's during such maneuvers, severe damage will occur to the aircraft and it may crash. This same situation exists in regard to fragile objects. The fragility level of an object, measured

in G's, is its ability to withstand deceleration. The purpose of a cushioning system is to reduce the shock transferred to the protected object to a degree below its fragility level, thus protecting it against physical damage.

#### Cushioning System Design

To understand how a cushioning material functions during shock transfer, one can consider what happens when a body is dropped onto a rigid surface such as a concrete floor. At impact the body is falling at a velocity ( $V = \sqrt{2 gh}$ ), where V is the velocity at impact in feet/second, g is the acceleration due to gravity of 32.2 feet/second/second, and h is the height of drop measured in feet. In a very short time after impact, this velocity is reduced to zero in a very small distance. Thus, a rapid decrease in velocity occurs, due to impact, and the body is subjected to a very high deceleration.

If the same situation is considered except that the body is dropped onto a resilient cushioning material which rests on the same rigid surface, the body has the same velocity at impact. However, the time required for the velocity to be reduced to zero is much greater than in the previous situation; the rate at which the velocity decreases is considerably less; and the distance traveled after initial contact with the cushion until the time the velocity is reduced to zero is considerably greater. Compared to the first situation, the body is subjected to lower G's. The resilient cushioning material has, therefore, attenuated the shock pulse by dissipating the kinetic energy present in the body at the time of impact over a longer time period. This has been expressed mathematically [2] as follows:

$$G = \frac{72}{t} \sqrt{h} \quad (\text{II-1})$$

where

$G$  = acceleration  $G$  - level

$t$  = shock pulse rise time in milliseconds

$h$  = drop height in inches.

Equation (II-1) shows that as the shock pulse rise time is increased, the  $G$ 's experienced by the body are proportionally decreased.

Increases in shock pulse rise time can usually be accomplished by increasing the thickness of the resilient cushioning material. For example, the  $G$ 's experienced by a body falling on a tangentially elastic cushioning material can be predicted as follows [2]

$$G = \frac{3.9 h}{T} \quad (\text{II-2})$$

where  $T$  = thickness of cushion in inches.

Equation (II-2) shows that the predicted  $G$  levels are reduced proportionally with increases in the thickness of the cushioning material. If the cushion thickness is increased, then, during impact, the excursion envelope of the body is increased proportionally and the body moves through an additional amount of cushioning material before it comes to rest. This increase in excursion directly increases the shock pulse rise time with an accompanying reduction in  $G$  levels. This reduction is consistent with the change in  $G$  levels that would be predicted by Equation (II-1) with an increase in shock pulse rise time.

#### History of Cushion System Design

Equation (II-2) is based on Mindlin's work in 1945 [3] that marked the beginning of the scientific approach to cushion design. Mendlin's paper was followed by a number of discussions [4-8] by author's who adopted his procedures. The next substantial step forward was the

development of optimum efficiency design points by Janssen [9]. Janssen utilized material properties determined by quasistatic means, in particular, stress-strain curves determined on a conventional compression tester with the speed of compression quite slow (not more than 2 inches/minute). He derived a cushion factor, "J", for several cushioning materials that was the ratio of optimum stress to optimum strain. Then G-level can be predicted on the basis of

$$G = J \frac{h}{T} \quad (\text{II-3})$$

where  $J$  = Janssen's cushion factor.

This was a significant improvement over Equation (II-2) in that the  $J$  values allowed for different performance factors for different materials. However, it soon became apparent that single curves based on the static stress-strain characteristics of a material did not describe the dynamic behavior. Several methods of presentation, all involving families of curves, were attempted. Gradually, Kerstner's approach [10] became the most popular. In this approach, a family of curves is drawn for each material thickness at each drop height, plotting peak dynamic stress (or acceleration) against the original static stress. A typical set of such curves, taken from Humbert and Hanlon [11], is shown in Figure 1.

These curves, referred to as dynamic cushioning curves, are generated for a particular type and thickness of cushion by performing drop tests using standard weight specimens that are dropped onto the cushion.

The static stress ( $\sigma_s$ ) is determined by:

$$\sigma_s = \frac{W}{A}, \quad (\text{II-4})$$

where  $\sigma_s$  is the static stress (psi),  $W$  is the specimen weight, and  $A$  is

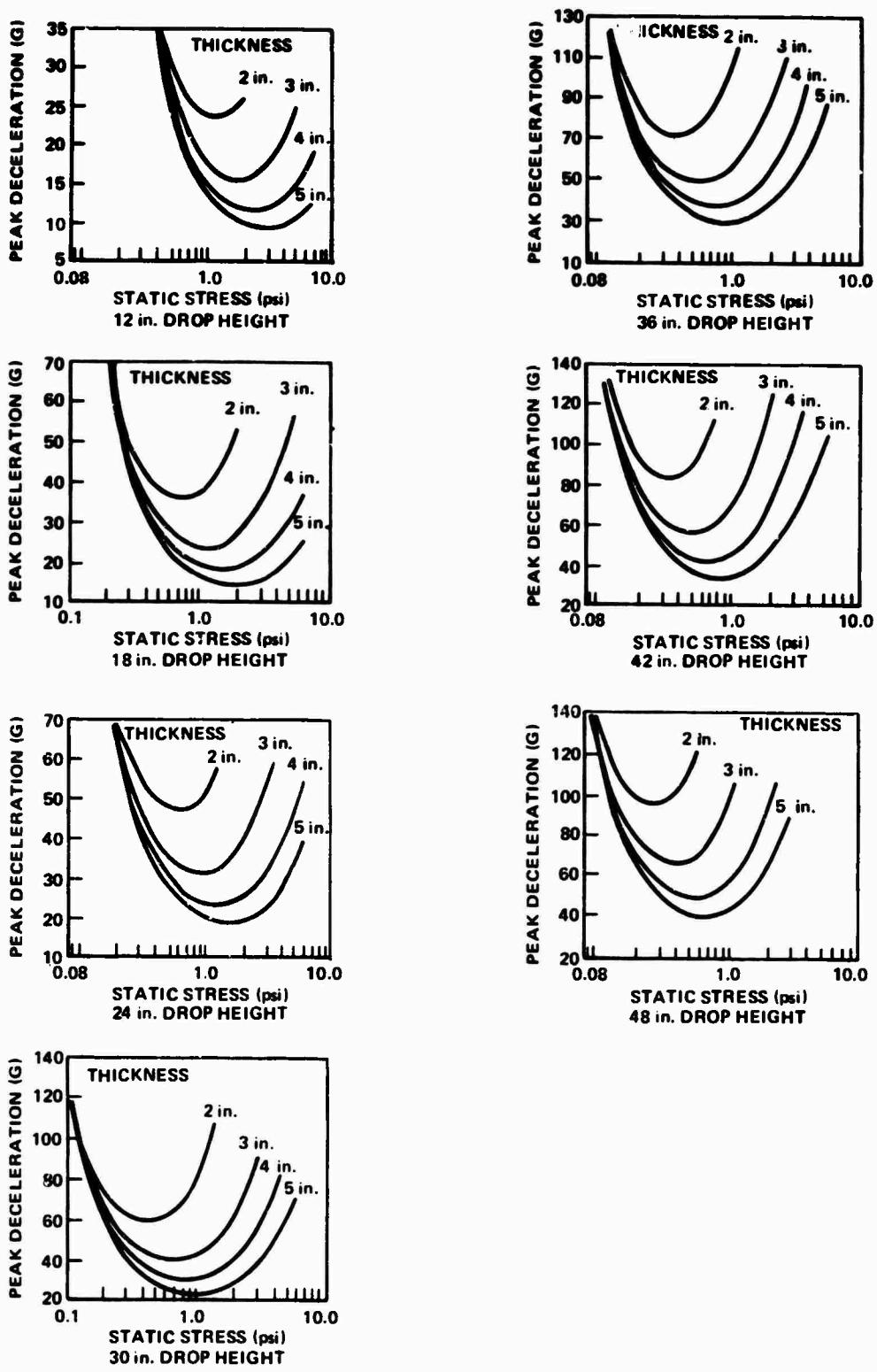


Figure 1. Peak-acceleration versus static-stress curves for polyethylene foam [11].

the footprint of the specimen in the cushion (in.<sup>2</sup>). A different curve is required for each drop height, thickness, and type of material. The curves are a good indication of the protection to be expected for a particular cushion scheme.

The current practice utilizes the Kerstner procedure in that manufacturers of cushioning products demonstrate the cushioning performance capabilities of their particular products by generating a series of dynamic cushioning curves at various drop heights and thicknesses of cushions (Figure 1). Also, Military Handbook -304, "Cushioning Design Handbook", was published in 1964 which provides families of dynamic cushioning curves for a large number of frequently utilized cushioning materials. Additional work was done by Mustin [12] who used a single value for correlating each dynamic cushioning curve in a family of curves such as Figure 1. This proved to be an over simplification in developing a general model of impact response, as will be seen in Chapter IV.

#### Reservations Concerning Current Cushioning System Design

In recent years, equipment designers have become increasingly aware of the detrimental effects of extreme temperature upon equipment performance. Consequently, there are now included in the qualification tests of equipment, some tests conducted at temperatures that are representative of the temperature extremes that the equipment is likely to encounter.

This extreme temperature testing has received substantial attention in the military, where the range of temperatures encountered is quite extreme, and the failures can produce catastrophic results. The transportation of military equipment is one area of concern and a study was made of several military containers that used bulk cushioning sys-

tems. It was found that temperature appeared to have a significant effect on impact response [1]. The cushioning systems in the containers did not perform properly due to changes in the performance of the bulk cushioning materials that were induced by extreme temperatures. Consequently, the items packaged in the containers (guided missile systems and system components) did not receive adequate protection, and the missile system reliability was compromised. This type of failure is a potential problem that can occur in very expensive equipment and produce a malfunction in weapon systems that compromises the combat power of a military organization. Also, if proper failsafe provisions are not incorporated into the protection of ordinance items, the safety of any of the personnel that handle the equipment within the logistics system is jeopardized.

Rather than incorporate additional protection into equipment, it is much more cost effective to improve the reliability of cushioning systems. This can be done if a reliable method of predicting cushioning performance can be developed.

A technique has been suggested for modifying the conventional dynamic cushioning curves in such a manner as to address the effect of temperature on the impact response of bulk cushioning systems [1]. The technique utilizes superimposed dynamic cushioning curves that are a super-positioning of the dynamic cushioning curves at temperature extremes upon the ambient dynamic cushioning curve. One such curve is presented in Figure 2. This type of curve demonstrates the effect of temperature for a selected set of conditions and provides the cushioning system designer with the capability of designing cushioning systems with a reduced likelihood of design failure under extreme temperature conditions.

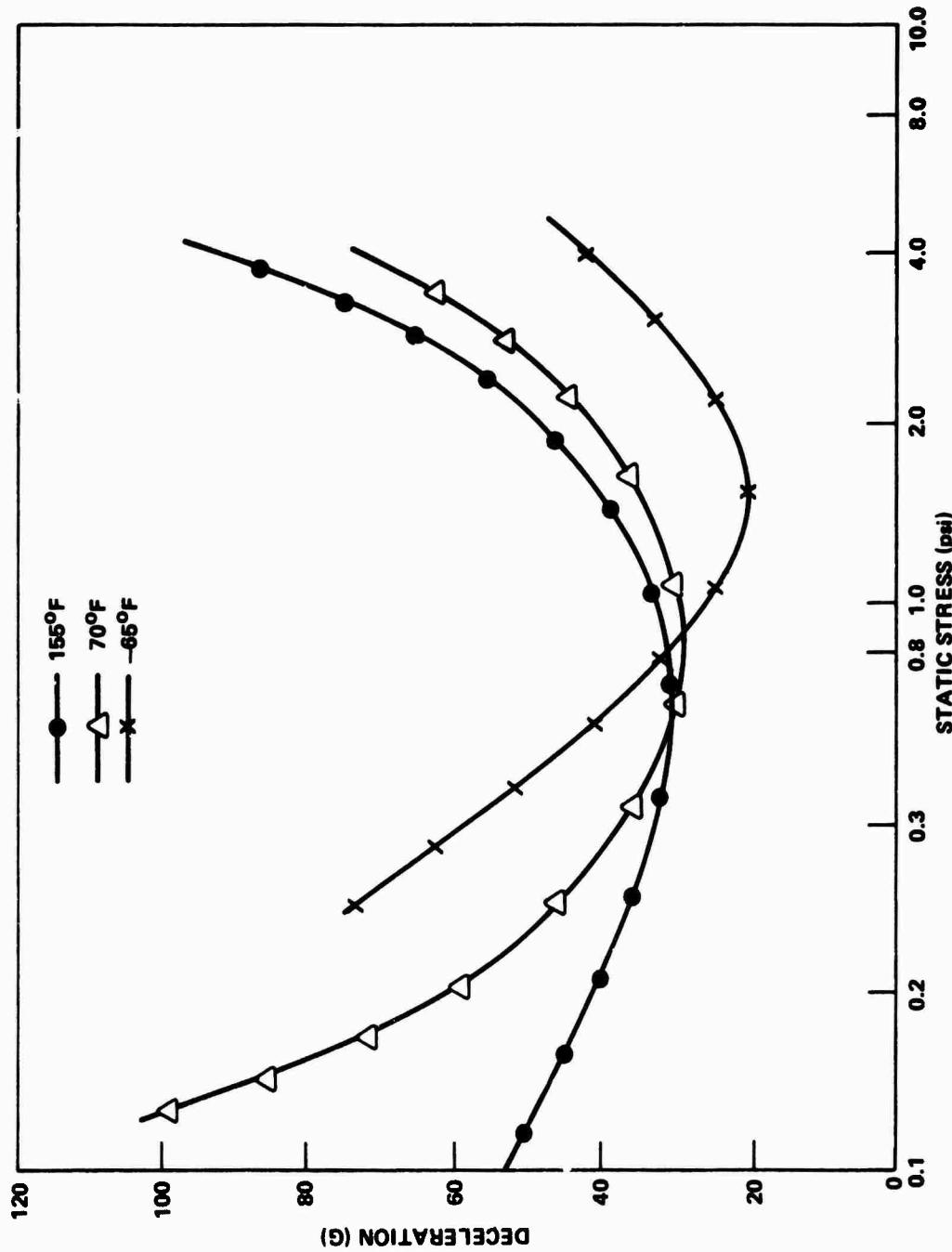


Figure 2. Superimposed dynamic cushioning curve for polyethylene foam (2 lb/ft<sup>3</sup> density, -65°, 70°, and 155°F, 30-in. drop height, 4-in. thickness).

## Chapter III

### BASIS FOR THE IMPACT RESPONSE MODEL

A valid model of impact response must incorporate all parameters that are expected to have a significant effect on impact response. Temperature has a significant effect on impact response [1], and it is postulated that viscoelastic theory can be utilized to formulate a model of impact response that incorporates temperature effects. The current design practice for predicting impact response is predicated on dynamic cushioning curves. Dynamic cushioning curves do not account for temperature effects on impact response. To improve the predictability of cushioning systems, a model of impact response must account for temperature effects. The temperature effects would be expected to be the most dramatic as the temperature tends towards the extremes; therefore, the temperature extremes encountered by a cushioning system are of particular concern in modeling impact response.

#### Temperature Extremes in the Model

Army Regulation 70-38, "Research, Development, Test and Evaluation of Materiel for Extreme Climatic Conditions," requires that the extreme external environments that are likely to be encountered, be considered in the design of Army materiel. The temperature extremes to be used in the design are defined in the AR according to the intended deployment of the materiel being designed. Figures 3 and 4 (from AR 70-38) present maps of the extreme temperature conditions to be used in the

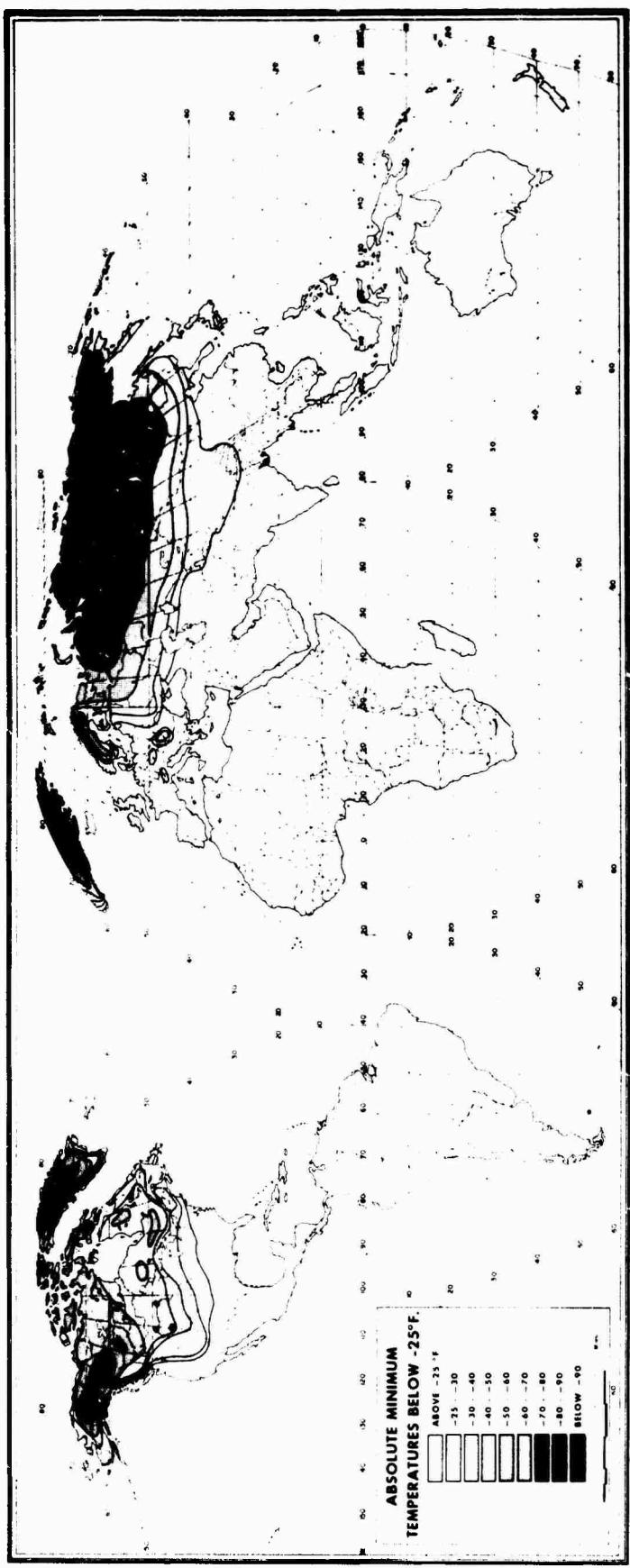


Figure 3. Distribution of absolute minimum temperatures.



Figure 4. Distribution of absolute maximum temperatures.

design of Army materiel. For example, for worldwide deployment, the operational temperature range is given as -65° to 160°F. However, for continental United States deployment only, the range is from -30° to 145°F. Army materiel is expected to function properly within these ranges and consequently the qualification testing of Army materiel is conducted at these extremes.

A research effort was initiated at the Army Missile Command (MICOM) in 1973 to develop superimposed dynamic cushioning curves that address the effect of temperature on the impact response of cushioning materials. The drop tests were conducted at MICOM and the results analyzed by the University of Alabama in Huntsville (UAH) under a supporting research contract [13]. The initial experimentation was conducted on Minicel material, a cross-linked polyethylene foam material with a 2 pound/foot<sup>3</sup> density manufactured by Hercules, Inc.

The drop test program that was conducted on the Minicel material used drop heights of 12, 18, 24, and 30 inches; temperatures of -65°, 70°, and 160°F; cushion thicknesses of 1, 2, and 3 inches; and static stress levels that varied from 0.04 to 5.0 psi. The G-level response and shock pulse duration were recorded for each of 2736 drop tests. An automated data handling system that included outlier tests and other statistical analyses was developed and used to analyze the Minicel data.

This experimental effort resulted in the generation of a data base of 2409 statistically valid data points. A family of second order polynomial equations was found to be the best predictor of impact response of the Minicel cushioning material. The data are given in Appendix A together with the families of regression polynomials for the 12 inch and 24 inch drop heights, and the correlation coefficients.

### Theoretical Basis of the Model

The construction of an impact response model for cushioning materials at varying temperatures requires the development of a functional relationship of the variables. The required relationship can be expressed mathematically as follows:

$$G = F(\sigma_s, t, \theta, h) \quad (\text{III-1})$$

where

$G$  = acceleration G-level

$\sigma_s$  = static stress in psi

$t$  = thickness of cushion inches

$\theta$  = cushion temperature in °F

$h$  = drop height in inches.

After considerable research, it was determined that the theory of viscoelasticity could be utilized to provide a theoretical basis for the model. Viscoelastic theory recognizes cushioning materials as belonging to a class of materials which have mechanical properties that are common to perfect solids and perfect liquids. Various theories have been developed over the past century for describing the behavior of perfect solids and perfect liquids. Among these, the oldest theories are the classical theory of elasticity and the theory of hydrodynamics. The classical theory of elasticity deals with the behavior of solids for which the stress is directly proportional to the strain but is independent of rate of strain. This type of solid is known as a Hookean solid (perfect elastic solid). The theory of hydrodynamics describes the behavior of perfect viscous liquids for which, in accordance with Newton's viscosity law, the stress is directly proportional to the rate of strain, rather than the strain itself. In certain instances, solids

and liquids may have their stress related to strain, rate of strain, and higher time derivatives of strain. Behavior of such materials is termed viscoelastic when the stress is linearly proportional to strain. Materials whose behavior is viscoelastic display solid-like and liquid-like characteristics [14].

Mathematical models have been formulated for viscoelastic materials that have validity over both short and long time periods; for example, Mustin [12] gives the creep and relaxation functions (long term behavior) for a number of simple mathematical models made up of simple spring elements that are assumed linear and massless, and of dashpots in which a piston is moving through a liquid that obeys Newton's law of viscosity (velocity is proportional to strain). The short term stress law and the long term creep and relaxation functions are given in Table I. Creep

TABLE I. SOME CREEP AND RELAXATION FUNCTIONS

Function	Linear	Dashpot	Voigt Solid	Maxwell Solid
Pictorial representation				
Stress law	$\sigma = E\epsilon$	$\sigma = c \frac{d\epsilon}{dt}$	$\sigma = E\epsilon + c \frac{d\epsilon}{dt}$	$\frac{\sigma}{c} + \frac{1}{E} \frac{d\sigma}{dt} = \frac{d\epsilon}{dt}$
Creep function	$\frac{1}{E} \cdot H(t)$	$\frac{t}{c} \cdot H(t)$	$\frac{1}{E} (1 - e^{-Et/c}) \cdot H(t)$	$\left(\frac{1}{E} + \frac{t}{c}\right) \cdot H(t)$
Relaxation function, R(t)	$E \cdot H(t)$	$c \cdot \delta(t)$	$E \cdot H(t) + c \delta(t)$	$(Ee^{-Et/c}) \cdot H(t)$

Notes:

- $E$  = spring constant
- $\sigma$  = stress
- $\epsilon$  = strain
- $c$  = damping coefficient
- $H(t)$  = unit step function
- $\delta(t)$  = Dirac delta function

functions are shown in Figure 5 while Figure 6 illustrates the relaxation behavior. The appearance of the relaxation functions indicates that the stress is infinite at the instant that strain occurs. This is due to the dashpot element which, unlike a spring, cannot give a finite instantaneous strain response to a finite instantaneous force change. Therefore, an infinite force is required to produce a finite instantaneous strain.

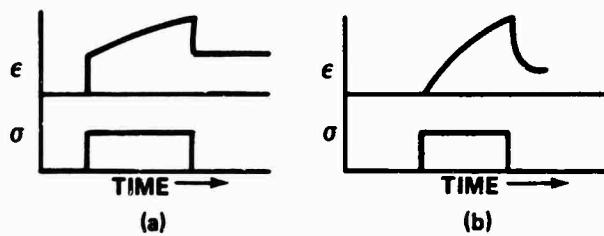


Figure 5. Behavior of some simple creep functions:  
(a) Maxwell solid, and (b) Voigt solid.

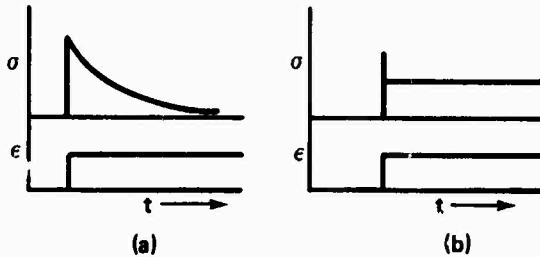


Figure 6. Behavior of some simple relaxation functions ( $R(t)$ ):  
(a) Maxwell solid, and (b) Voigt solid.

These relationships are expressed compositely in constitutive equations which, in viscoelastic theory, describe the response of materials to mechanical excitation. The constitutive equation in conjunction with the energy, momentum, and continuity equations permits the prediction of complicated responses to complicated excitations, like the flow of liquids in irregularly shaped channels or the deflections of beams of varying cross sections under complicated loading conditions.

There are two approaches to obtaining the constitutive equation of a material: it can be obtained either experimentally with the help of some simple, well-defined tests (like the stress-strain curve) or the response of the material to some strain or stress history can be calculated with the help of a model describing its structure. Hooke's law is an example for the first group, the phenomenological equations. It is based solely on experimental observation. The best known example in the latter category of structural theories is the kinetic theory of rubber elasticity predicting the elastic response of vulcanized elastomers from their structure.

While the phenomenological constitutive equations describe the results of experimentally obtained data and are usually applied to predict more complicated behavior, the structural ones offer an insight into structure-property relations [15].

Since temperature effects are required in the constitutive equation that models impact response, one must consider that portion of viscoelastic theory which accounts for the behavior of materials at varying temperatures. Most of this theory arises from the consideration of the behavior of the molecule. The molecule may be visualized as a long curled elastic chain which may have cross-linking with other molecular chains. Since molecular activity is a function of temperature, strain response (the summation of individual molecule responses) is also a function of temperature. On this basis, in the glassy zone (the zone below the temperature at which the polymer structure starts to become brittle), molecular "curling and uncurling" cannot occur rapidly enough to follow the stress. In the rubbery zone (the zone

above the temperature at which the polymer structure becomes rubbery), curling and uncurling are in phase with the stress which is not conducive to energy dissipation.

Let the long term elasticity of a material at a given temperature,  $\theta_1$ , be plotted against the natural logarithm of time. Suppose, now, that the curve shifts horizontally to the right as the temperature is lowered. This situation is illustrated in Figure 7 for three temperatures.

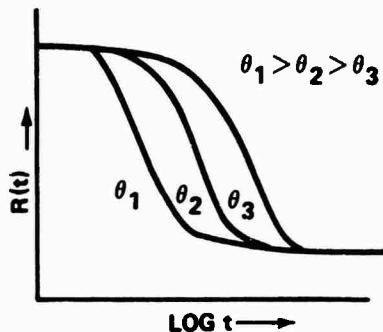


Figure 7. Shift in elasticity characteristics as a function of time and temperature.

The behavior sketched in Figure 7 is typical of many materials utilized as cushions. Materials which behave in this manner are called thermorheologically simple. Amorphous polymers behave in this manner while polymers with crystalline structures do not. Most cushioning materials may be considered amorphous. Teflon and plasticized polyvinyl chloride are not amorphous, but they are rarely used as cushions [16].

#### Viscoelastic Considerations

A constitutive equation that relates the maximum G-level experienced by a body when subjected to a deceleration impact into a cushioning material is required for the impact response model stated in Equation (III-1). A partial basis is found in a paper on the viscoelastic

properties of thermorheologically simple cushioning materials by Cost [17]. Cost considers a viscoelastic Kelvin body which is identical to the Voight model shown in Table I. The stress laws are identical with Figure 7 except for notation, in that Cost uses  $K$  and  $\eta$  for  $E$  and  $c$ , respectively.

Consider a body of mass  $M$  falling under the influence of gravity from a height  $h$  and impacting a cushion material of area  $A_c$  and thickness  $T$ . The static stress  $\sigma_s$  is defined as the weight of the body divided by the area of the cushion  $A_c$ .

Assume the cushion material behaves as a viscoelastic Kelvin body whose stress strain relation can be expressed as

$$\sigma = K\epsilon + \eta\dot{\epsilon} \quad (\text{III-2})$$

where  $\eta$  and  $K$  are material properties. When Equation (III-2) is applied to the problem under consideration,  $\sigma$  becomes the static stress in a bulk cushion,  $\sigma_s$ , and  $\epsilon$ , the unit strain, can be expressed in terms of the displacement of the upper surface of the cushion as  $\epsilon = x/T$ . If the principles of Newtonian mechanics are applied to the falling body after time of impact where force equals mass times acceleration, then the equation of motion for the body can be expressed as

$$M\ddot{x} = -\sigma_s A_c \quad (\text{III-3})$$

Upon substitution of Equation (III-2) into Equation (III-3)

$$M\ddot{x} = -\frac{A_c K}{T} x - \frac{A_c \eta}{T} \dot{x} \quad (\text{III-4})$$

where  $x$  is measured relative to the initial location of the upper surface of the cushion and is considered positive downward. Rearranging the equation of motion gives

$$\ddot{x} + \frac{A_c \eta}{MT} \dot{x} + \frac{A_c K}{MT} x = 0 \quad . \quad (\text{II -5})$$

Equation (III-5) can be cast in the following canonical form:

$$\ddot{x} + 2\beta_c \omega_n \dot{x} + \omega_n^2 x = 0 \quad (\text{III-6})$$

provided  $\beta_c$  and  $\omega_n$  are defined as follows:

$$\begin{aligned} \beta_c &= \frac{\eta A_c}{2\sigma_s T \omega_n} = \frac{\eta A_c}{2MT \omega_n} \\ \omega_n &= \sqrt{\frac{gK}{\sigma_s T}} = \sqrt{\frac{A_c K}{MT}} \quad . \end{aligned} \quad (\text{III-7})$$

The parameters  $\beta_c$  and  $\omega_n$  have physical significance as being the damping factor and natural frequency of the undamped system, respectively. For the case of "underdamped" motion, the solution of Equation (III-6) can be expressed as

$$x(t) = e^{-\beta_c \omega_n t} [B_1 \cos \mu t + B_2 \sin \mu t] \quad (\text{III-8})$$

where

$$\mu = \omega_n \sqrt{1 - \beta_c^2} \quad (\text{III-9})$$

and  $B_1$  and  $B_2$  are arbitrary constants which can be determined from the initial conditions. Applying the initial conditions

$$x(0) = 0 \quad ,$$

$$\dot{x}(0) = \sqrt{2gh} \quad (\text{III-10})$$

to evaluate the two constants  $B_1$  and  $B_2$  gives

$$B_1 = 0 \quad ,$$

$$B_2 = \frac{\sqrt{2gh}}{\omega_n \sqrt{1 - \beta_c^2}} \quad . \quad (\text{III-11})$$

Therefore, the acceleration history can be expressed as

$$\ddot{x}(t) = \frac{\omega_n \sqrt{2gh}}{\sqrt{1 - \beta_c^2}} e^{-\beta_c \omega_n t} \left[ (2\beta_c^2 - 1) \sin \omega_n t - 2\beta_c \sqrt{1 - \beta_c^2} \cos \omega_n t \right]. \quad (\text{III-12})$$

In this research, only the maximum value of the acceleration is of interest. Consequently, it is necessary to separate the solution into oscillatory and decaying components. To accomplish this, a trigonometric identity must be introduced into Equation (III-12) to obtain the following expression:

$$\ddot{x}(t) = -\frac{\omega_n \sqrt{2gh}}{\sqrt{1 - \beta_c^2}} e^{-\beta_c \omega_n t} \cos \left[ \omega_n \sqrt{1 - \beta_c^2} t + \gamma \right] \quad (\text{III-13})$$

where

$$\gamma = \tan^{-1} \left[ \frac{2\beta_c^2 - 1}{2\beta_c \sqrt{1 - \beta_c^2}} \right]. \quad (\text{III-14})$$

Equation (III-13) expresses the acceleration of the falling mass as a function of time. It is observed that the character of the response is that of a damped sinusoid with the exponential and trigonometric components of the solution governing the transient behavior. Since the magnitude of the exponential or the trigonometric term can achieve a maximum value of one, the coefficient of these terms governs the absolute magnitude of the acceleration. Thus, in regard to the form of the equation, it can be seen that

$$\ddot{x}_{\max} \approx \frac{\omega_n \sqrt{2gh}}{\sqrt{1 - \beta_c^2}}. \quad (\text{III-15})$$

Recalling the expansion

$$(1 - z^2)^{-1/2} = 1 + \frac{1}{2} z^2 + \frac{3}{8} z^4 + \dots , \quad (\text{III-16})$$

the peak acceleration can be expressed as

$$\ddot{x}_{\max} \approx \omega_n \sqrt{2gh} \left( 1 + \frac{1}{2} \beta_c^2 \right) \quad (\text{III-17})$$

where only terms up through the second order in  $\beta_c$  have been retained.

Returning to the primary variables  $\sigma_s$ , T, h, and  $\theta$ , and Equation (III-7) gives

$$\ddot{x}_{\max} \approx g \left( \frac{hK}{\sigma_s T} \right)^{1/2} \left[ 1 + \frac{1}{8} \frac{\eta^2 g}{\sigma_s T K} \right] . \quad (\text{III-18})$$

Inserting arbitrary constants instead of the specific numerical coefficients gives

$$\ddot{x}_{\max} \approx C_1 \frac{h^{1/2}}{\sigma_s^{1/2} T^{1/2}} + C_2 \frac{h^{1/2}}{\sigma_s^{3/2} T^{3/2}} \eta^2 \quad (\text{III-19})$$

where  $\eta$  is a material property dependent on temperature.

Assuming thermorheologically simple behavior for the viscoelastic material,  $\eta$  can be expressed as

$$\eta(\theta) = \eta_0 a(\theta) \quad (\text{III-20})$$

where  $a(\theta)$  is the "shift factor". Furthermore,  $a(\theta)$  has been shown by experience [16] to be expressible as

$$a(\theta) = C_1' e^{-C_2' (\theta - \theta_0)} = C_1'' e^{-C_2' \theta} . \quad (\text{III-21})$$

Substituting into Equation (III-20) gives

$$\eta(\theta) = \eta_0 C_1'' e^{-C_2' \theta} \quad (\text{III-22})$$

and

$$\eta^2(\theta) = C_3 e^{-C_4 \theta} \quad (\text{III-23})$$

which can be expanded in a Taylor series to give

$$\eta^2(\theta) = c_3 \left[ 1 - c_4 \theta + c_5 \theta^2 \right] \quad (\text{III-24})$$

where only terms up through the second order have been retained. Upon substitution of this expression into Equation (III-19), we get

$$\begin{aligned} \ddot{x}_{\max} \approx & K_1 \frac{h^{1/2}}{\sigma_s^{1/2} T^{1/2}} + K_2 \frac{h^{1/2}}{\sigma_s^{3/2} T^{3/2}} + K_3 \frac{h^{1/2}}{\sigma_s^{3/2} T^{3/2}} \theta \\ & + K_4 \frac{h^{1/2}}{\sigma_s^{3/2} T^{3/2}} \theta^2 \end{aligned} \quad (\text{III-25})$$

as an expression relating the variables  $\sigma_s$ ,  $T$ ,  $h$ , and  $\theta$  to the peak acceleration.

If additional terms are retained, a more general expression will result. Such a general expression is

$$\ddot{x}_{\max} \approx c_0 \left( \frac{h}{\sigma_s T} \right)^{1/2} + \sum_{i=1}^N \frac{c_i h^{1/2}}{(\sigma_s T)^{i+1/2}} \left[ \sum_{j=1}^M K_{ij} \theta^j \right] \quad (\text{III-26})$$

where the  $c_i$  ( $i = 0, 1, 2, \dots, N$ ) and  $K_{ij}$  ( $j = 1, 2, \dots, M$ ) are constants to be determined by curve fitting procedures.

Expressions of the form given in Equations (III-25) and (III-26) appear likely candidates for empirically curve-fitting cushioning system experimental data. This concludes Cost's derivation.

#### Verification

Once a candidate relationship was selected, it was verified. A statistical comparison of the G-level predicted by the function to the data base listed in Appendix A was used to verify the model.

This was accomplished by modifying the Stepwise Regression Procedure given by Draper and Smith [18]. In the Draper and Smith version,

variables are entered and removed from the regression at each stage on the basis of F-tests performed on the coefficients of the independent variables. This procedure was modified in this research in that the variables were entered into the regression equation and remain there as long as they make a contribution to the overall correlation. This modification was considered most appropriate during the preliminary phases of the modeling. It precludes the removal of a variable from the regression equation at one stage because other variables can explain most of the variation, and then find in a later stage that the variable was needed but was not available. In the Draper and Smith version, the establishment of critical F values can inadvertently cause the removal of a desirable variable.

It is considered most important in the early stages of model development that all variables that can possibly be retained remain in solution. Once they are discarded as irrelevant, it is difficult to reincorporate them and there is a substantial risk that they will be lost. However, the modification can cause the stages in the solution to carry superfluous variables through several iterations and ultimately retain them in the final solution. This does require some small amount of additional computer time. However, these unnecessary variables are easily identified and can be discarded during the fine tuning of the final solution with little risk to the validity of the model. If variables are discarded prematurely, which could happen in the Draper and Smith version of the regression analysis, the validity of the model may be compromised.

A program listing of the Stepwise Regression Procedure that was utilized, called MLRD, is given in Appendix B. One form of output from the

program is printer plots in the form of dynamic cushioning curves. These are a nesting of various thicknesses of a particular cushioning at a particular drop height and temperature. The curves are plotted using numbers corresponding to the thickness of cushion. The dynamic cushioning curves that were selected from the UAH study are displayed in Appendix C using the MLRD format. The initial validation exercise for the modeling effort, then, consisted primarily of displaying the curve shape and nesting of the model being validated to determine if the model generated dynamic cushioning curves that compared favorably with those in the UAH study.

#### Theoretical Validity

The required form of the model as given in Equation (III-1) can be identified in viscoelastic theory as a phenomenological constitutive equation. The model proposed by Cost was selected as the most viable candidate, where  $\ddot{x}_{max}$  is the maximum G-level experienced by a body during impact into a cushion, and  $C_0, C_1, K_1, K_2$ , etc. are regression constants. This equation was used as the basic underlying structure of the model because it provides a functional relationship of all the variables required in Equation (III-1) and has a theoretical basis in viscoelastic theory. There was one transformation of variables made prior to using the equation. It can be reasoned that when  $\theta$  is a significant aspect of the model, then for  $\theta^j$ , where an exponent j is added, the polarity of  $\theta^j$  will oscillate. To avoid this difficulty,  $\theta$  was transformed to  ${}^\circ R$ , so the extreme temperatures encountered in the data are always positive.

### Validity of Initial Trials

A program code of Equation (III-26) was run on MLRD and the MLRD plot (Figure 8) shows that  $\ddot{x}$  was a progressively decreasing value as  $\sigma_s$  increased. This was anticipated to an extent by Cost during his finite element analysis [17]. He generated a plot of peak acceleration versus static stress in the region where stress is proportional to strain (Figure 9).

This deficiency in a constitutive equation is not at all uncommon as Meinecke [15] suggests that even though the behavior of actual materials is usually different from that predicted by various classical theories, for engineering purposes it may be worthwhile to approximate the actual behavior to the idealized behavior. But for cases where it is not possible to approximate the actual behavior of the material to the idealized behavior without sacrificing the accuracy in prediction, it is very essential to consider the anomalies from the idealized character. The departure of actual material from its idealized character may be due to various reasons. For instance, the deviation from the idealized character may be such that the stress, instead of being directly proportional to the strain, is related to it in a complicated manner. Solids display this behavior when they are stressed beyond the so called elastic range or when the deformation becomes so large as to introduce nonlinearity between stress and displacements. Likewise, liquids for which the stress is nonlinearly related to rate of strain are termed non-Newtonian liquids. Equation (III-26) is a good representation of the linearity of impact response but a nonlinearity must be introduced as an initial correction for shape to get a U shaped curve.

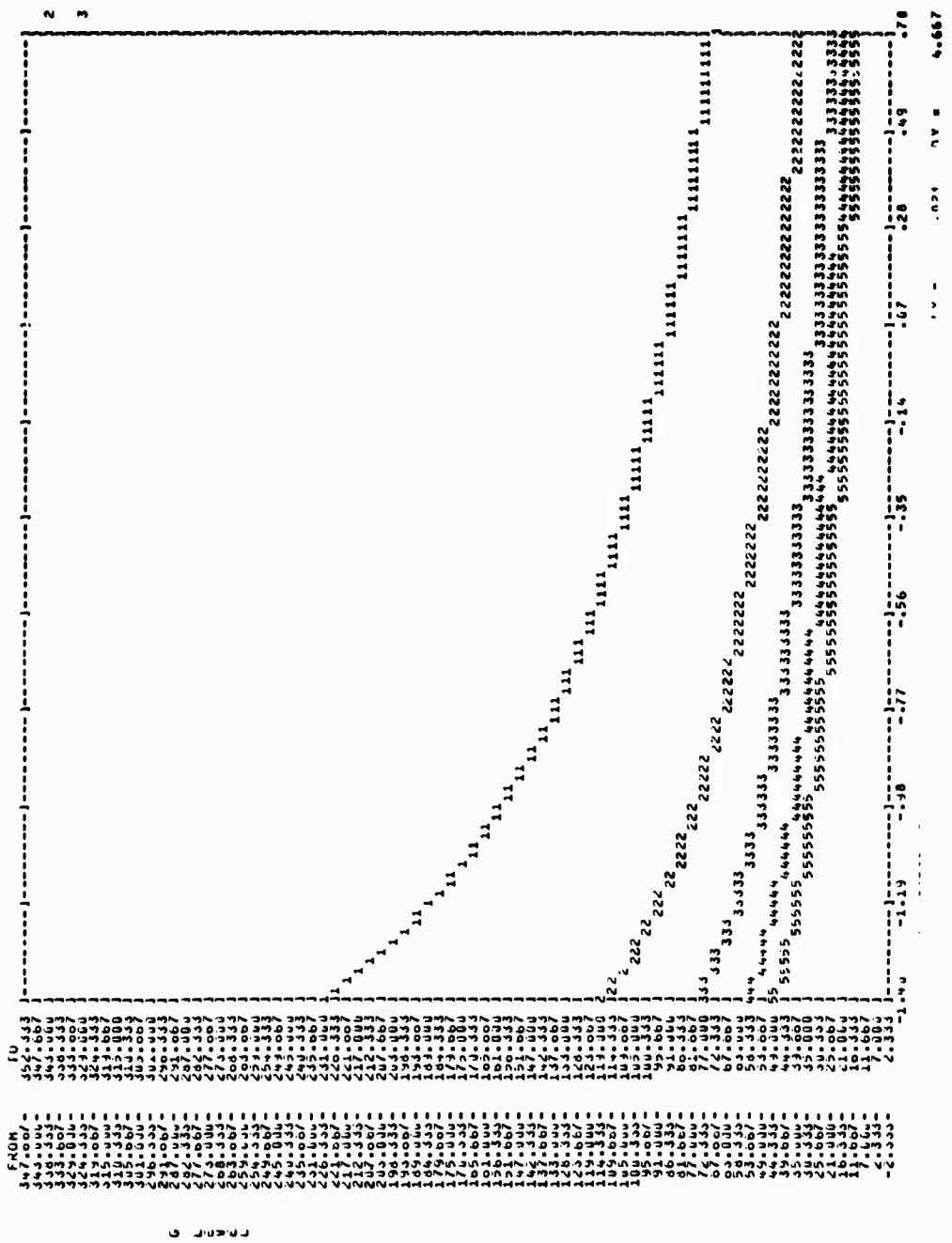


Figure 8. Preliminary MLRD plots of dynamic cushioning curves of the response model at 70°F and 12-inch drop height.

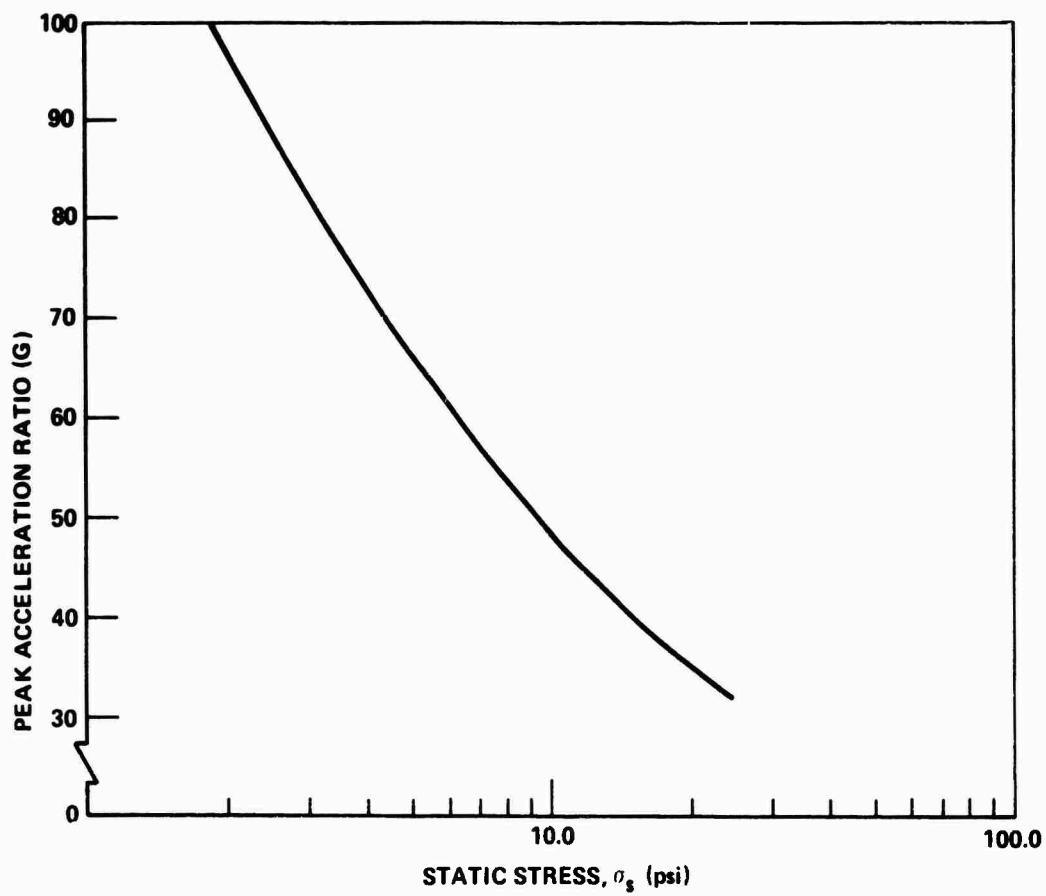


Figure 9. Finite element solution results for peak acceleration.

#### Initial Correction for Shape

All known dynamic cushioning curves exhibit a characteristic U shape. Consider a compression spring approximately 6 inches in diameter and a free height of 6 inches, similar to those used in the front suspension of an automobile. On this spring a rigid steel plate that serves as an impacting surface is placed. Then, a safety pin, an automobile, and a locomotive are dropped, in turn, onto the spring from approximately 6 inches.

Now the results of the drop test are examined. The compression spring is much too stiff for the small weight of the safety pin, so the

spring does not deflect to store the energy of the fall. This rapid deceleration of the pin causes it to experience high G-levels. Because the compression spring was taken from an automobile, the spring deflects with the fall of the automobile. Energy is stored by the spring and the load on the automobile is proportional to the stiffness of the spring and its deflection. For the locomotive, due to the extremely heavy mass, the spring deflects until the coils of the springs have compressed completely. Up to this point the deceleration has progressed slowly with no high G-levels. However, from this point the locomotive will be decelerated to zero velocity almost immediately and will experience high G-levels.

Thus, for a given stiffness and free height of the spring, the appropriate weight dropped from a given height will result in the maximum energy storage in the spring and a minimum G-level experienced by the weight. The U shapes of dynamic cushioning curves, such as those in Figure 1, are the result of this property of bottoming of bulk cushions. The optimal conditions exist at the ogive of the curve as indicated in Figure 10. At this point the maximum energy is stored in the cushion with the accompanying minimum G level.

A simplified model of a foam material such as the one proposed by Gent and Thomas [19] (Figure 11) is helpful in visualizing the transitional aspects of why bottoming occurs. The foam consist of thin threads joined together to form a cubical lattice. Mechlin [20] explains the bottoming effect as the result of the ligaments of the cushioning material structure packing together, one against another.

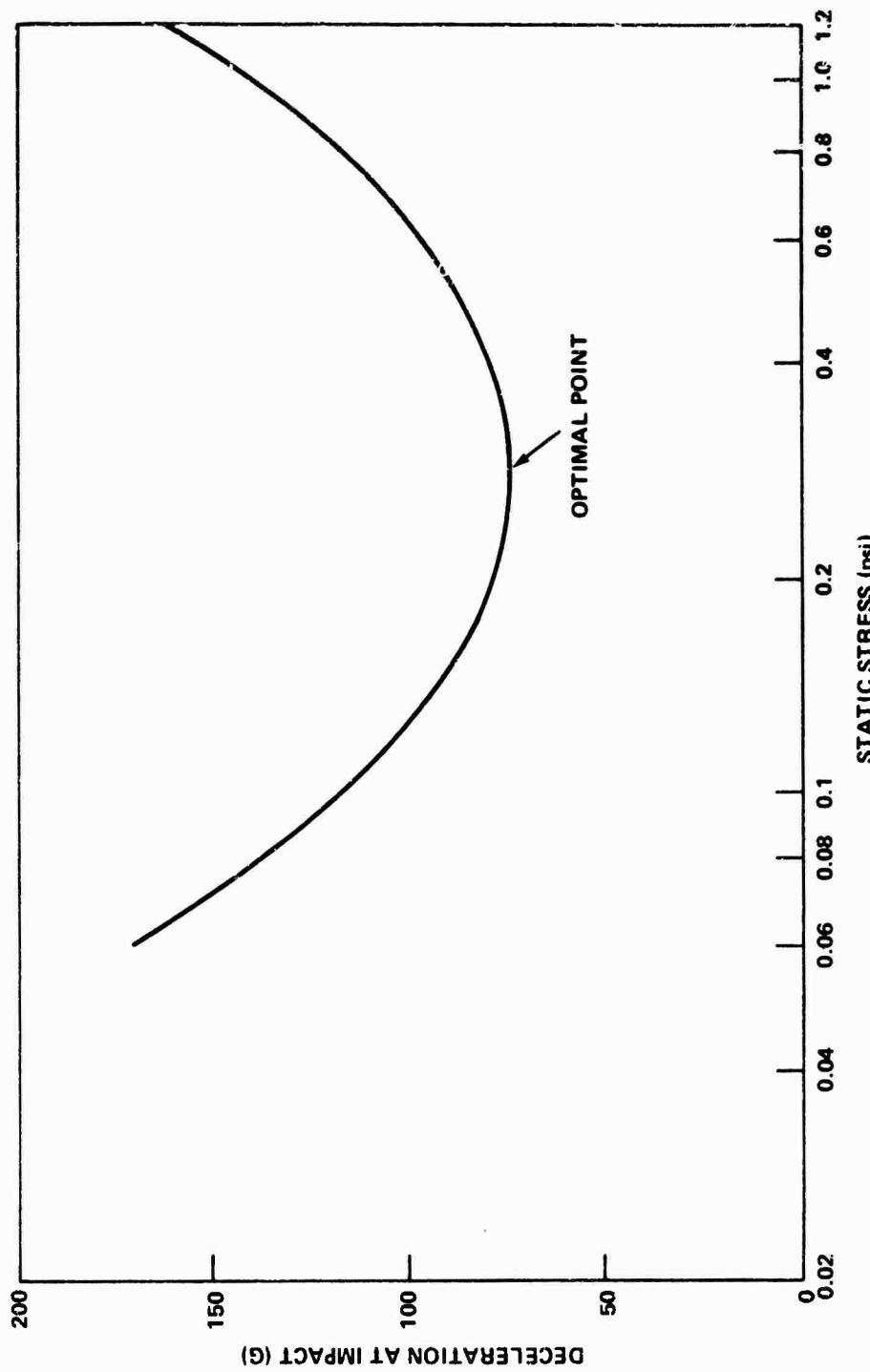


Figure 10. Typical dynamic cushioning curve (Minicel, 30-in. drop height, 70°F, 1 in. thickness).

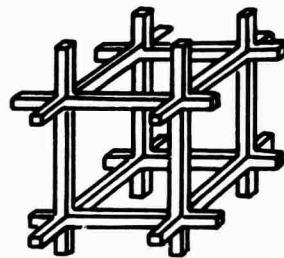


Figure 11. Structural model of a foam material [22].

This situation could be characterized as representing an entirely new material with stress-strain properties substantially different from the original cushioning material. Accordingly, the initial model, Equation (III-26), does not address the non-linearity introduced beyond the bottomed region of the curve, and in itself is not a satisfactory model of cushion response. However, it is reasonable to assume that the functional arrangement of the variables may not be rearranged through the bottomed region of the curve. This possibility was explored using a modular modeling technique and it was found that the basic arrangement of the variables has merit.

## Chapter IV

### THE IMPACT RESPONSE MODELS

#### Development of the General Model

The initial model, Equation (III-26), proved deficient in representing the nonlinear characteristics of cushion response. However an extensive literature search showed it to be the only known model that provides a direct relationship of the required variables. Consequently, a modular technique suggested by Shannon [21] was used to modify this model and construct a valid model of impact response. The relationship of each independent variable ( $\sigma_s$ , T,  $\theta$ , h) and its effect upon the dependent variable (G-level) was studied and the finalized relationship for each independent variable was then incorporated into the model.

#### Variable 1, Drop Height (h)

The effect of drop height upon G-level that is given in Equation (II-1) is based upon the relationship,

$$V = \sqrt{2gh} . \quad (IV-1)$$

V is the velocity at impact and is related as the square root of drop height (h). Mindlin [3], Janssen [9], and others utilized this relationship of G-level versus drop height, and the derivation by Cost in Equation (III-26) is on this same basis. It was determined that drop height should be incorporated into the model as  $h^{1/2}$ .

### Variable 2, Static Stress ( $\sigma_s$ )

In the UAH study [12], many relationships of G's versus static stress were investigated and it was found that the best agreement was obtained in a second order polynomial of the natural log of stress. Several polynomials of various orders of static stress were tested in this research and a similar conclusion was reached, namely, that a second order polynomial was superior and that the desired U shaped dynamic cushioning curves of G's versus static stress would result.

The initial MLRD Computer runs were made using the following relationship:

$$G = F(\sigma_s, h) \quad . \quad (\text{IV-2})$$

The variables were input with  $h^{1/2}$  and a second order polynomial of  $\sigma_s$ . The best fit was obtained using the following functional relationship:

$$G = C_0 + C_1 h^{1/2} + C_2 h^{1/2} \ln \sigma_s + C_3 h^{1/2} (\ln \sigma_s)^2 \quad . \quad (\text{IV-3})$$

These polynomials generated U shaped curves similar to those in the UAH study (compare Figure C-2 and Figure 12).

### Variable 3, Thickness of Cushion (T)

Janssen [9], Mindlin [3], and others suggested that G-level was an inverse relationship to thickness as given in Equation (II-2). This seems intuitively correct and many attempts were made using this relationship. T was introduced into these as a negative exponential such as  $T^{-1/2}$ ,  $T^{-3/2}$ ,  $T^{-5/2}$ ,  $T^{-7/2}$ , etc. Computer runs showed good correlation with thickness input as a negative exponential of the type given. Figure 12 shows the typical nesting effect obtained.

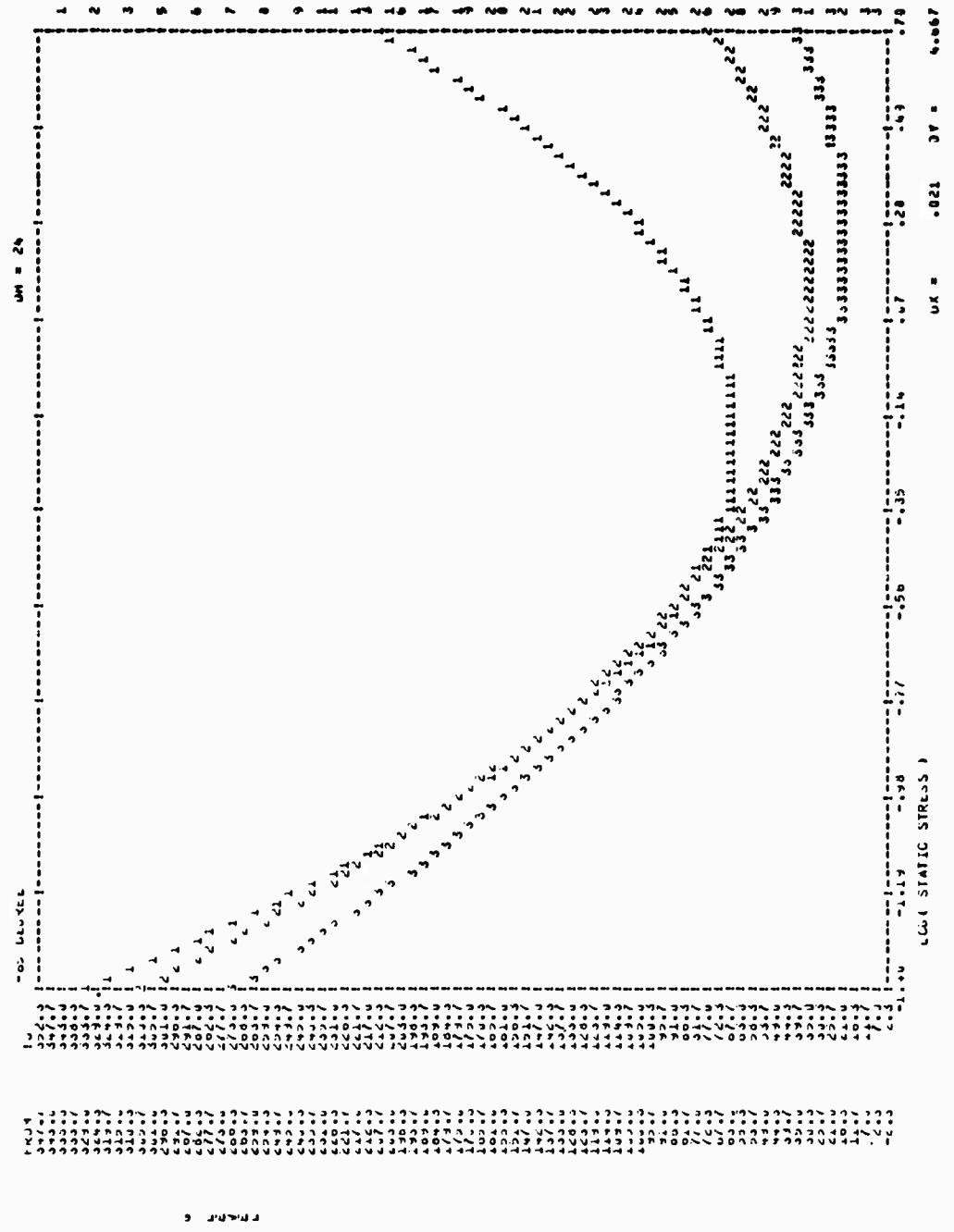


Figure 12. MLRD plots of dynamic cushioning curves of  $G = C_0 + C_1 h^{1/2} \ln \sigma_s + C_2 h^{1/2} (\ln \sigma_s)^2$  at  $-65^\circ\text{F}$  and  $160^\circ\text{F}$  and 24-inch drop height.

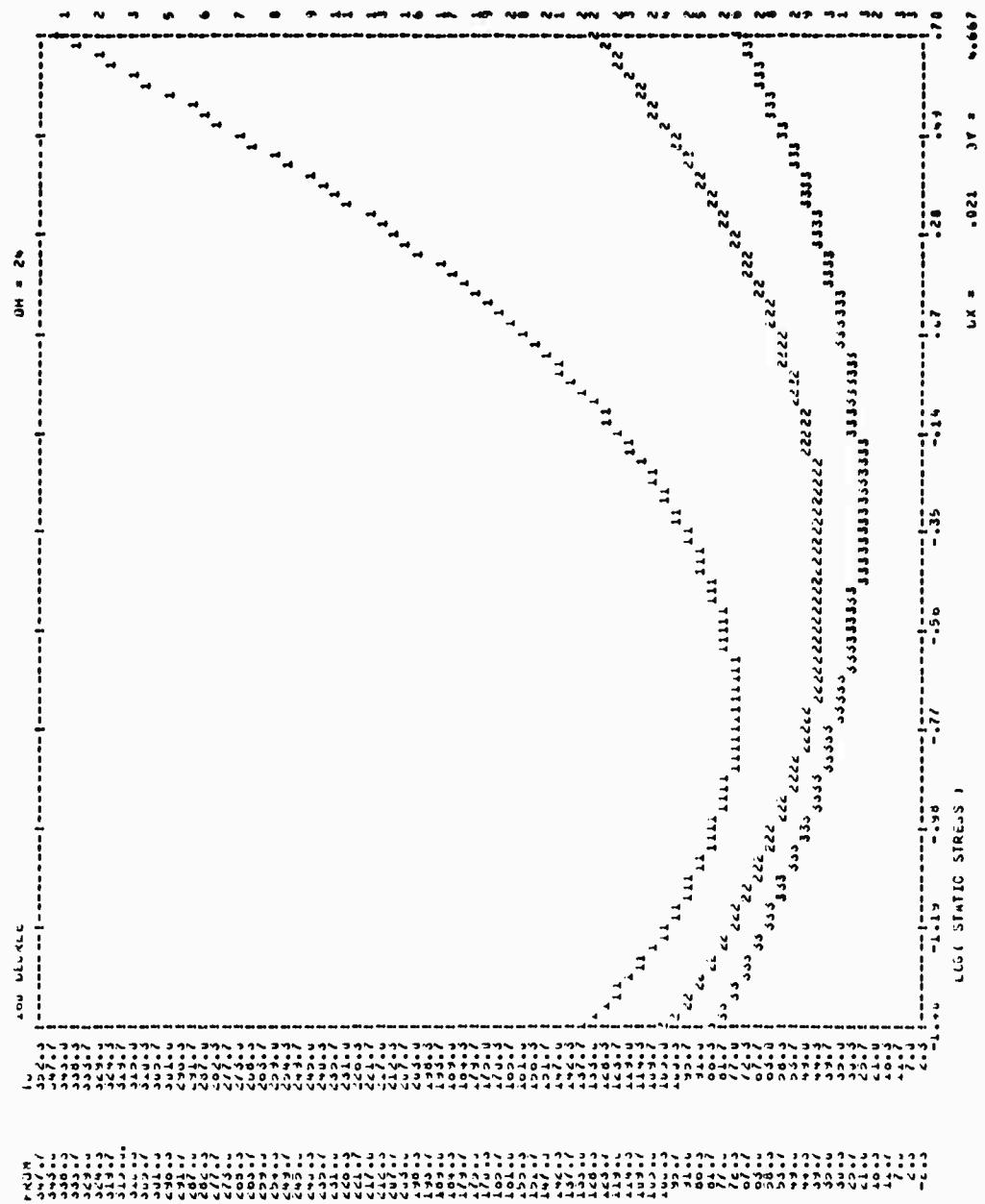


Figure 12. Concluded.

#### Variable 4, Temperature ( $\theta$ )

Temperature effects were expected to be the most difficult to incorporate and this turned out to be the case. It can be reasoned that the phase shift effect of temperature discussed in viscoelastic theory for thermorheologically simple materials produces the multiplier effect of the exponentials shown by Cost. Models were tried with several orders of temperature and the polynomial with  $\theta^j$ , where  $j = 1, 2, 3, \dots, n$  were the most satisfactory. Figure 13 is a typical plot using  $\theta^j$  (where  $j = 1, 2, 3$ ) that shows the shifting obtained from a cold temperature of  $-65^\circ\text{F}$  (C) through ambient (A) to hot  $160^\circ\text{F}$  (H).

#### The General Model

Many relationships were tried and rejected. Each time, the basic underlying structure of the variables was rejustified and new formulations were attempted. The process was repeated until a valid General Model of impact response was developed. The General Model is given as follows:

$$G = C_0 + \sum_{\ell=0}^S h^{\ell/2} \sum_{k=0}^R \frac{1}{T^{(1/2+k)}} \sum_{j=1}^N \theta^j \sum_{i=0}^M c_{ijk\ell} (\ln \sigma_s)^i . \quad (\text{IV-4})$$

This General Model incorporates each of the variables in the manner prescribed in the modular modeling effort. The curves produced by this model are all U shaped and can be displayed using the MLRD plot routine. A series of plots from the General Model for various temperatures and drop heights are given in Appendix D for the 18 and 30 inch drop heights at  $-65^\circ$ ,  $70^\circ$ , and  $160^\circ\text{F}$ . A comparison of these curves

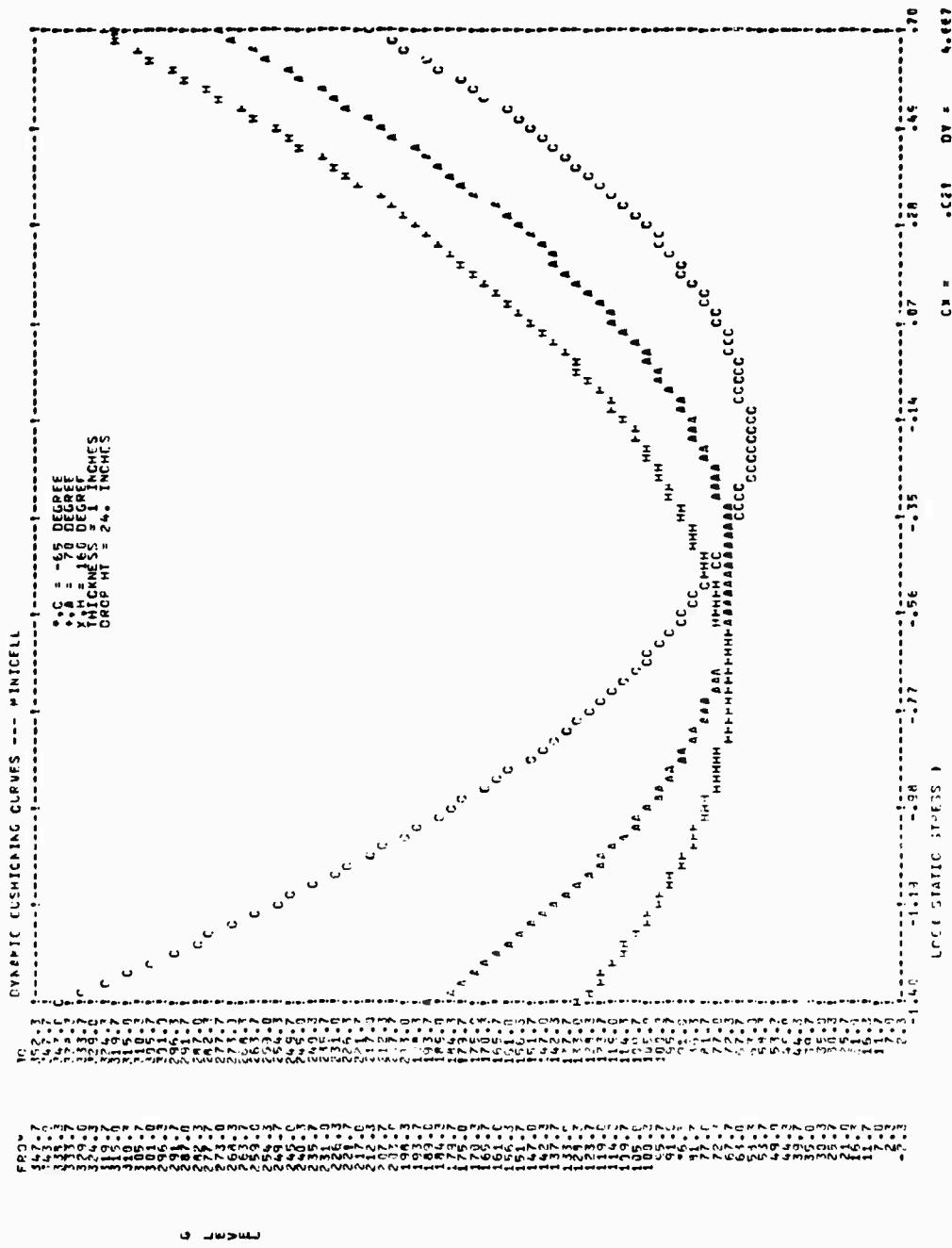


Figure 13. MLRD plots of superimposed dynamic cushioning curves derived from a model using  $\sigma_j$ .

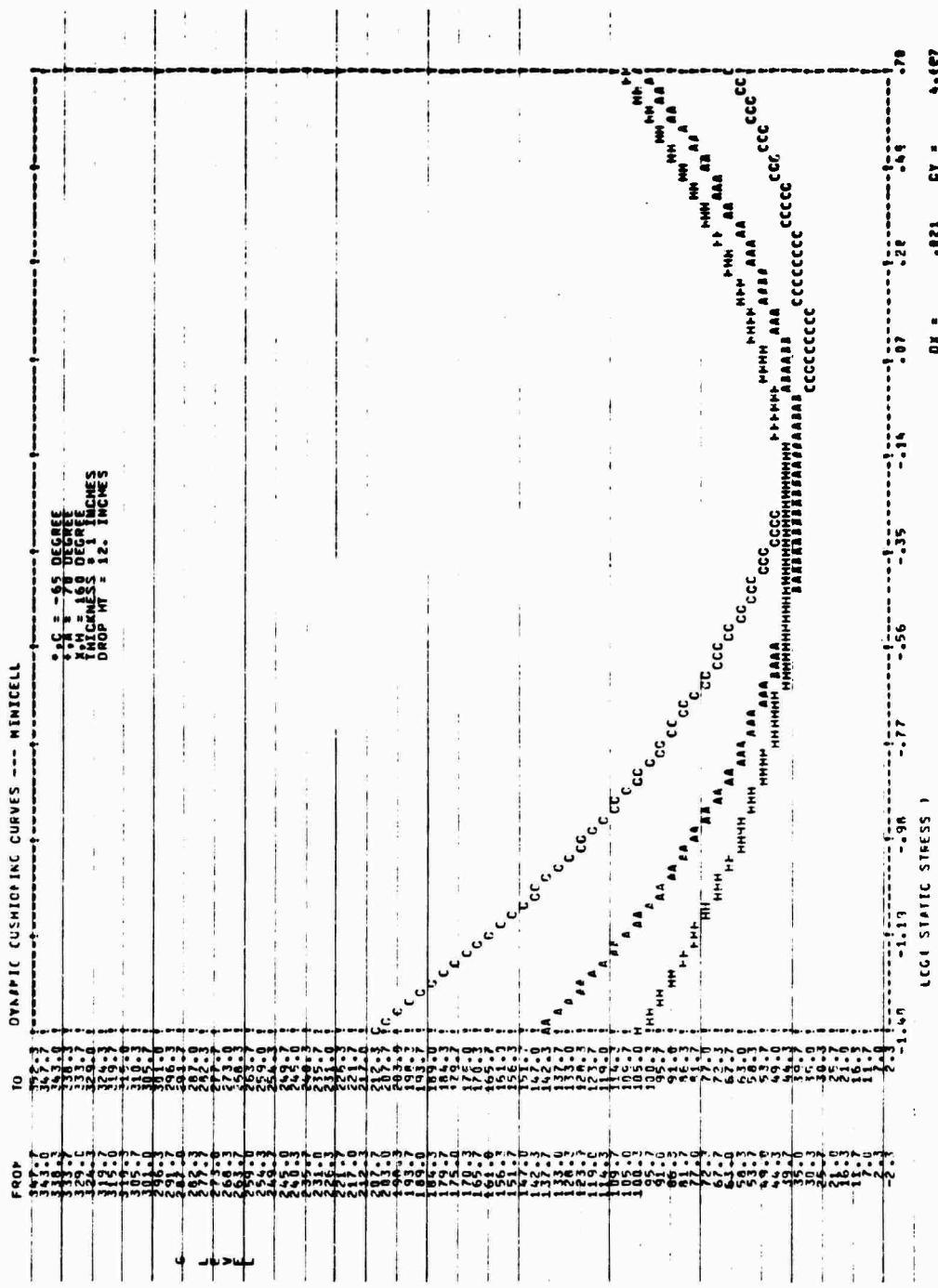


Figure 13. Concluded.

demonstrates the following:

- A) The curves of G-level versus static stress are U shaped, consistent with cushioning theory
- B) G-level decreases as drop height decreases, consistent with Equation (II-1)
- C) The curves are shifted laterally with temperature, consistent with the thermorehologically simple assumption
- D) G-level decreases as thickness increases, consistent with Equation (II-2)
- E) Thickness curves are nested similar to Humbert and Handlon [11], Figure 1.

#### Precision of the General Model

The precision of this General Model, Equation (IV-4), has to do with how well it can be made to represent a particular type of cushioning material. This can be determined through comparison with an experimental data base similar to that in Appendix A. Sensitivity analysis can be performed on the model by adjusting the upper limits of the summations, M, N, R, and S to determine if it is possible to obtain the desired precision.

A sensitivity analysis was performed using the Minicel data of Appendix A. S is set at 1, R is set at 1, N is set at 3, M is set at 2, and the Minicel Model takes the following special form:

$$G = C_0 + \sum_{\ell=0}^1 h^{\ell/2} \sum_{k=0}^1 \frac{1}{T^{(1/2+k)}} \sum_{j=1}^3 \theta^j \sum_{i=0}^2 C_{ijkl} (\ln \sigma_s)^i . \quad (IV-5)$$

This is expanded to a 36 term polynomial as follows:

$$\begin{aligned}
 G = & c_0 + \frac{\theta}{T^{1/2}} \left[ c_{0100} + c_{1100} \ln \sigma_s + c_{2100} (\ln \sigma_s)^2 \right] \\
 & + \frac{\theta^2}{T^{1/2}} \left[ c_{0200} + c_{1200} \ln \sigma_s + c_{2200} (\ln \sigma_s)^2 \right] \\
 & + \frac{\theta^3}{T^{1/2}} \left[ c_{0300} + c_{1300} \ln \sigma_s + c_{2300} (\ln \sigma_s)^2 \right] \\
 & + \frac{h^{1/2} \theta}{T^{3/2}} \left[ c_{0111} + c_{1111} \ln \sigma_s + c_{2111} (\ln \sigma_s)^2 \right] \\
 & + \frac{h^{1/2} \theta^2}{T^{3/2}} \left[ c_{0211} + c_{1211} \ln \sigma_s + c_{2211} (\ln \sigma_s)^2 \right] \\
 & + \frac{h^{1/2} \theta^3}{T^{3/2}} \left[ c_{0311} + c_{1311} \ln \sigma_s + c_{2311} (\ln \sigma_s)^2 \right] \\
 & + \frac{h^{1/2} \theta}{T^{1/2}} \left[ c_{0101} + c_{1101} \ln \sigma_s + c_{2101} (\ln \sigma_s)^2 \right] \\
 & + \frac{h^{1/2} \theta^2}{T^{1/2}} \left[ c_{0201} + c_{1201} \ln \sigma_s + c_{2201} (\ln \sigma_s)^2 \right] \\
 & + \frac{h^{1/2} \theta^3}{T^{1/2}} \left[ c_{0301} + c_{1301} \ln \sigma_s + c_{2301} (\ln \sigma_s)^2 \right] \\
 & + \frac{\theta}{T^{3/2}} \left[ c_{0110} + c_{1110} \ln \sigma_s + c_{2110} (\ln \sigma_s)^2 \right] \\
 & + \frac{\theta^2}{T^{3/2}} \left[ c_{0210} + c_{1210} \ln \sigma_s + c_{2210} (\ln \sigma_s)^2 \right] \\
 & + \frac{\theta^3}{T^{3/2}} \left[ c_{0310} + c_{1310} \ln \sigma_s + c_{2310} (\ln \sigma_s)^2 \right] .
 \end{aligned}$$

(IV-5a)

### Minicel Model

To finalize the Minicel Model, the MLRD program was used with Equation (IV-5), and successive analysis of variance tables were constructed as each additional variable was entered into solution. These ANOVA tables were used to determine if the incoming variable made a significant contribution to the overall correlation. A Duncan [22] F test is utilized with this format; it was found that the 25th variable that entered did not make a significant contribution to the G-level response prediction. The test hypothesis can be stated as follows:

$H_0$  : the entering variable has no effect upon the G-level response prediction.

$H_1$  : the entering variable has a significant effect upon the G-level response prediction.

The test is

$$F_{\text{calc}} = \frac{\text{MS}_{\text{due to}}}{\text{MS}_{\text{about}}} \quad (\text{IV-6})$$

and the null hypothesis  $H_0$  can be rejected when  $F_{\text{calc}} > F_{\text{table}}$ . Utilizing an  $\alpha$  level of 0.05, the final ANOVA's and F tests (Tables II and III) show that the 24th variable makes a significant contribution but the 25th does not. One further test is made to verify that all the regression coefficients in the final Minicel Model are significant. This test is conducted using a "t" statistic as follows:

$$t_n = \frac{C_n}{S_n} \quad (\text{IV-7})$$

where

$t_n$  is the t statistic for the nth term

TABLE II. ANOVA TABLE AND F TESTS FOR ENTERING THE 24TH VARIABLE INTO THE MINICEL REGRESSION EQUATION

Source	S.S.	d.f.	M.S.	F <sub>calc</sub>	F <sub>.05</sub>	Decision
Due to all 24 variables	5,026,531.3	24	209,438.8	1830.76	1.52	SIGN
Due to first 23 variables	(5,024,189.8)	(23)	(218,443)	1834.9	1.52	SIGN
Due to addition of 24th variable to first 23 variables	(?341.5)	(1)	(2341.5)	20.6	3.84	SIGN
About regression on all 24 variables (Residual)	172,858.2	1511	114.4			
About regression on the first 23 variables	175,199.7	1512	115.9			
Total	5,199,389.5	1535				

TABLE III. ANOVA TABLE AND F TESTS FOR ENTERING THE 25TH VARIABLE INTO THE MINICEL REGRESSION EQUATION

Source	S.S.	d.f.	M.S.	F <sub>calc</sub>	F <sub>.05</sub>	Decision
Due to all 25 variables	5,026,771.0	25	201,070.8	1758.9	1.52	SIGN
Due to first 24 variables	(5,026,531.3)	(24)	209,438.8	1830.8	1.52	SIGN
Due to addition of 25th variable	(239.70)	(1)	239.70	2.0	3.84	NOT SIGN
About regression on all 25 variables	172,618.5	1510	114.39	2.0		
About regression on the first 24 variables	172,858.2	1511				
Total	5,199,389.5	1535				

$C_n$  is the coefficient of the nth term

$S_n$  is the standard error of the nth term.

The test hypothesis is as follows:

$H_0$  : the nth coefficient is the same as zero

$H_1$  : the nth coefficient is significantly different from zero.

In conducting the test, the null hypothesis can be rejected when  $t_n > t_{table}$ . With an  $\alpha$  level of 0.05, the test of all the coefficients in the Minicel Model, as given in Table IV, are found to be significantly different from zero.

The resultant Minicel Model is a 25 term regression polynomial.

The Minicel Model has a 0.983 correlation coefficient which compares favorably with the UAH data in Appendix A.

The reliability of the correlation coefficient can be tested statistically using a t statistic defined as follows [23]:

$$t_n = r \sqrt{\frac{n - 2}{1 - r^2}}$$

where

$r$  = the correlation coefficient

$n$  = the number of samples used to derive the regression equation

$t_n$  = the resulting number of standard errors of  $r$  in the interval between the computed  $r$  and  $r = 0$ .

The test hypothesis is as follows:

$H_0$  :  $r = 0$

$H_1$  :  $r > 0$  .

TABLE IV. TEST OF THE SIGNIFICANCE OF THE REGRESSION COEFFICIENTS OF THE MINICEL MODEL

Var	Coefficient Subscript Eq IV-5a	Coefficient	Stand. Error	Coef/S <sub>e</sub>	F	Decision
0	0	-.83931602E+01	.437855E+00	8.0893	65.4	SIGN
1	2100	-.35419457E+01	.817803E+00	-18.7316	350.8	SIGN
2	1111	-.15318724E+02	.163007E+00	20.4911	419.8	SIGN
3	2111	-.33401870E+01	.402348E+01	51.5707	2659.5	SIGN
4	0101	-.20749326E+03	.107843E+01	-46.6972	2180.6	SIGN
5	11C1	-.50350553E+02	.105041E+00	13.6561	186.4	SIGN
6	2101	-.14344456E+01	.100147E+01	-6.6860	44.7	SIGN
7	1200	-.66958791E+01	.129727E+01	-42.1322	1775.1	SIGN
8	0201	-.54656687E+02	.372076E+00	31.2337	975.5	SIGN
9	1201	-.11621323E+02	.237025E+00	-5.4916	30.1	SIGN
10	0300	-.13016393E+01	.140395E+00	14.8081	219.2	SIGN
11	1300	-.20789886E+01	.996818E-02	-22.7366	516.9	SIGN
12	2300	-.22664200E+00	.6333909E-01	-6.3323	40.1	SIGN
13	0311	-.40141035E+00	.303853E-01	20.1325	405.3	SIGN
14	1311	-.61173036E+00	.453820E-02	-21.0037	441.1	SIGN
15	2311	-.95319017E+01	.126069E+00	31.2708	977.8	SIGN
16	0301	-.394222841E+01	.360074E-01	-24.0050	576.2	SIGN
17	1301	-.86603770E+00	.141474E+02	-16.5242	273.0	SIGN
18	0110	-.23377506E+03	.395093E+01	7.1637	51.3	SIGN
19	1110	-.28303458E+02	.386583E+01	12.8507	165.1	SIGN
20	0210	-.49678750E+02	.213922E+01	12.3051	151.4	SIGN
21	1210	-.26323240E+02	.310653E+00	-19.5325	381.5	SIGN
22	2210	-.60678372E+01	.286608E+00	-21.4651	460.7	SIGN
23	1310	-.61520847E+01	.509818E-01	21.0916	444.8	SIGN
24	2310					

In conducting the test, the null hypothesis can be rejected when  $t_n > t_{table}$ . This test of the reliability of the correlation coefficient was conducted on the Minicel Model. The null hypothesis can be rejected since  $t_n = 291.2 > t_{table}$ . The Minicel Model can be written as follows:

$$\begin{aligned}
 G = & -8.39 + \frac{3.54 \theta (\ln S)^2}{T^{1/2}} - \frac{15.31 \theta h^{1/2} \ln S}{T^{3/2}} \\
 & + \frac{3.34 \theta h^{1/2} (\ln S)^2}{T^{3/2}} + \frac{207.49 \theta h^{1/2}}{T^{1/2}} - \frac{50.35 \theta h^{1/2} \ln S}{T^{1/2}} \\
 & + \frac{1.43 \theta h^{1/2} (\ln S)^2}{T^{1/2}} - \frac{6.70 \theta^2 \ln S}{T^{1/2}} - \frac{54.66 \theta^2 h^{1/2}}{T^{1/2}} \\
 & + \frac{11.62 \theta^2 h^{1/2} \ln S}{T^{1/2}} - \frac{1.30 \theta^3}{T^{1/2}} + \frac{2.08 \theta^3 \ln S}{T^{1/2}} \\
 & - \frac{0.23 \theta^3 (\ln S)^2}{T^{1/2}} - \frac{0.40 \theta^3 h^{1/2}}{T^{3/2}} + \frac{0.61 \theta^3 h^{1/2} \ln S}{T^{3/2}} \\
 & - \frac{0.09 \theta^3 h^{1/2} (\ln S)^2}{T^{3/2}} - \frac{0.87 \theta^3 h^{1/2} \ln S}{T^{1/2}} - \frac{233.77 \theta}{T^{3/2}} \\
 & + \frac{28.30 \theta \ln S}{T^{3/2}} + \frac{49.68 \theta^2}{T^{3/2}} + \frac{26.32 \theta^2 \ln S}{T^{3/2}} - \frac{6.07 \theta^2 (\ln S)^2}{T^{3/2}} \\
 & - \frac{6.15 \theta^3 \ln S}{T^{3/2}} + \frac{1.07 \theta^3 (\ln S)^2}{T^{3/2}} + \frac{3.94 \theta^3 h^{1/2}}{T^{1/2}}
 \end{aligned} \tag{IV-8}$$

where  $\theta = \frac{^{\circ}\text{F} + 460}{100}$  and  $S$  = static stress in psi  $\times 100$ .

This model can be used to predict impact response for Minicel cushioning systems. The model is expected to be 95% reliable when used within the ranges of the independent variables which are as follows:

$h$  = drop height from 12 through 30 inches

$\sigma_s$  = static stress range from 0.03 to 5.0 psi

$\theta$  = temperature from -65° to 160°F

T = thickness of cushion from 1 through 3 inches.

The model will predict with good accuracy at all levels of the independent variables within these ranges. Also it was found that the Minicel Model does a reasonable job of predicting results beyond the ranges stipulated for the independent variables as can be seen when the results of tests of 4 and 5 inch thick Minicel samples are compared in Chapter VI.

#### Adjustments in Precision

The special form of the General Model as given in Equation (IV-5) was used in the validation of the model for the cross-linked polyethylene foam Minicel material and gave a correlation of 0.983. If additional precision had been required the values of S, R, M, and N could have been increased which may be necessary with other materials but gave only a minimal improvement in precision here. It was apparent, however, that increases in M, which incorporates  $\sigma_s$  with exponentials over 2 is in general not worthwhile. Also, increases in N above 3 that incorporate  $\theta$  at exponentials of  $\theta$  over 3 are of marginal value in improved precision. The changes in R and S that affect the exponentials of thickness and drop height should be explored first if additional precision is required in a model of a particular material.

Once a special form of the General Model is found that represents a particular cushioning material, a measure of its validity can be

demonstrated utilizing the same statistical procedures as demonstrated for the Minicel Model. Improvements in the precision of the model will be reflected in increases in the "t" statistic associated with the test of the correlation coefficient. Values comparable to those of the Minicel Model are desirable.

## Chapter V

### VALIDATION

The model building process proceeded through many iterations of development and verification and culminated in the General Model of impact response stated mathematically as

$$G = c_0 + \sum_{\ell=0}^S h^{\ell/2} \sum_{k=0}^R \frac{1}{T^{(1/2+K)}} \sum_{j=1}^N \theta^j \sum_{i=0}^M c_{ijkl} (\ln \sigma_s)^i . \quad (V-1)$$

This gives the basic underlying structure for a model of impact response for bulk cushioning materials. The question of how good a model has been developed is answered in the validation process. Naylor and Finger [24] suggest three stages of validation:

- 1) Verification of internal structure
- 2) Empirical testing
- 3) Verification as a predictor.

#### Verification of Internal Structure

The validating process began when the first model of impact response was developed. At that time the ingredients of a model were selected and their relationships postulated on the basis of prior knowledge and existing theory. The basis of the General Model was founded in the theory of viscoelasticity, and the individual parameters ( $\theta$ ,  $\sigma_s$ ,  $T$ ,  $h$ ,  $G$ ) were arranged in the model in a manner that is consistent with theory and intuition. The General Model that resulted from

the modeling effort incorporates the following characteristics:

- 1) The dynamic cushioning curves of G-level versus static stress generated by the model are U shaped. This is consistent with cushioning theory.
- 2) The predicted G-levels increase with increased drop height. The higher drop heights incorporate more energy into the system which would increase the energy levels experienced by the falling body.
- 3) The predicted G-levels increase with reduced cushion thickness. The G's experienced by a falling body is dependent on shock pulse duration, Equation (II-1), and a thinner cushion would allow less shock pulse time and an accompanying increase in G-levels.
- 4) Temperature effects induce lateral shifts in the dynamic cushioning curves. This is consistent with the phase shift function concept of viscoelastic theory. It is also intuitively consistent in that reduced temperatures would be expected to stiffen the cushioning material and require an increased stress level for comparable shock attenuation.
- 5) The dynamic cushioning curves generated as output from the General Model form a series of curves that are nested within progressive values of thickness and drop height for all possible temperature conditions and static stress conditions. It can be concluded that the internal structure of the General Model is intuitively correct and the output of the General Model is consistent with expectation.

#### Empirical Testing

The General Model of Equation (V-1) is hypothesized as the model of impact response that is applicable as the basic underlying structure

of a model for any one of the many cushioning materials. An impact response model for any one particular cushioning material can be constructed by establishing the summation levels M, N, R, and S, and the values of the regression coefficients in the General Model. This can be done through a testing program that generates a data base similar to Appendix A. Then an analysis program is required that provides a least squares fit of the data base. The test program required can be similar to the one conducted in the UAH study. Once the test program is completed and the data base established, an analysis must be performed to develop the model. The stepwise regression analysis given in Appendix B or a similar analysis routine can be utilized.

When this procedure was followed in constructing the Minicel Model, Equation (IV-8), the General Model was used as the basic underlying structure and the statistical tests performed in verifying the Minicel Model serve to validate the General Model. The ANOVA Table for the Minicel Model (Table II), showing an  $F_{\text{calc}}$  of 1830.76 and a correlation coefficient of 0.983, is indicative of the fit of the Minicel Model to the UAH results. In addition to the high correlation of the model with experimental data, the Minicel Model also demonstrates the characteristic U shaped dynamic cushioning curves which were one of the more important aspects of the impact response model considered in the model development. The dynamic cushioning curves produced from the Minicel Model were plotted using the MLRD printer plot routine for many conditions of thickness, drop height, and temperature. The classic U shape was evident in all the plots, six of which are given in Appendix E.

An additional measure of validity of the model can be seen when the best fitting polynomials in the UAH study are compared to the Minicel Model. The Minicel Model plots in Appendix E have the independent variables at the same levels as the plots of the UAH curves in Appendix C.

#### Verification as a Predictor

The final test of validity of the General Model is to assess the ability of the model to predict impact response. It was previously demonstrated that the Minicel Model does an excellent job of predicting impact response when compared to the actual data. However, it must be remembered that the Minicel Model is dependent upon these data when formulating its predictions. It is a limited dependency in that it is using all 2709 data points in predicting a particular dynamic cushioning curve, and in fact only a small portion of the data base, approximately 50 data points, apply directly to the particular conditions with the independent variables at the appropriate levels. The UAH best fitting polynomials, however, are derived using only those data points where the independent variables are at the appropriate levels.

To fully verify the model as a predictor, a data base independent of that used to generate the model must be utilized. Three such data bases are available: a data base of 1, 2, and 3 inch thickness Minicel material at -65°, 70°, and 160°F and at 27 inch drop height, and a data base of 4 and 5 inch Minicel material. The 27 inch drop height data are given in Appendix F and the 4 and 5 inch data in Appendix G. The 27 inch data are contained within the range of the independent variables and was not used in formulating the Minicel Model. However, the 4 and

5 inch data are beyond the data extremes of the developed model. Also the 4 and 5 inch samples themselves were not homogeneous. The samples of the 4 inch material were two-piece cushions which were 2 inches thick. The 5 inch material was made using a 2 inch and a 3 inch piece. This stacking is representative of how cushioning is actually used in shock isolation systems requiring thicker sections than the maximum manufactured thickness of 3 inches. Whether the Minicel Model can do an adequate job of predicting the impact performance of these stacked samples is questionable. The model's ability to predict adequately under these circumstances would indicate that the stacking did not significantly perturb the cushioning performance from that encountered in the 1, 2, and 3 inch continuous samples that form the basis of the model.

To determine statistically how well the model fits a set of independent data, Box and Draper [25] suggest it is appropriate to investigate the bias and variance of the predictor. Two statistical tests can be utilized for this purpose. One test, a test of means, determines whether there is a bias in the predicted values of the model when compared to actual data values. The other test, a test of variances, determines if the variations of the predicted values inherent in the model are comparable with the variations in experimental values.

#### Test of Means

In all the data bases, including the UAH data, there are three replications of the same conditions of the independent variables. These three replications can be considered a cell. Then, the mean of the G-levels of the three data values in a cell ( $G_i$ ,  $i = 1, 2, 3$ ) can be compared with the predicted G-level from the model for that cell, and

it is reasonable to expect the sum of the differences to vanish. Any difference that cannot be reasonably attributed to sampling error must be considered a bias that is introduced because the model values are not good predictors.

The first step in this test is to formulate a difference between the data values in a cell and the model prediction for that cell. This can be expressed mathematically as follows:

$$\Delta_j = (GM_j - \overline{GD}_j) \quad (V-2)$$

where

$j$  = number of a cell with fixed values of  $\epsilon$ ,  $\sigma_s$ ,  $T$ ,  $h$

$\Delta_j$  = cell difference for cell  $j$

$GM_j$  = the G-level predicted by the model for cell  $j$

$\overline{GD}_j$  = the mean value of G-level for the three data values in cell  $j$

$$\overline{GD}_j = \frac{\sum_{i=1}^3 G_i}{3}$$

Then  $S$ , the standard deviation of all the cells, can be defined as follows:

$$S = \sqrt{\frac{\sum_{j=1}^N (\Delta_j - \bar{G})^2}{N - 1}} \quad (V-3)$$

where

$N$  = the number of cells in the data base

$$\sum_{j=1}^N \Delta_j$$

$\bar{G}$  = the grand mean of all the cell differences,  $\bar{G} = \frac{\sum_{j=1}^N \Delta_j}{N}$ .

The hypothesis to be tested is based on the expected value of the differences which can be written  $E(\Delta_j)$  and stated as follows:

$$H_0: E(\Delta_j) = 0$$

$$H_1: E(\Delta_j) \neq 0 .$$

A two tailed "t" test is used where the t statistic is given as

$$t_m = \frac{\bar{G} \sqrt{N}}{S} \quad (V-4)$$

where  $t_m$  = the test statistic for the model.

The test compares  $t_m$  with  $t_{table}$  and the null hypothesis can be rejected when  $t_m > t_{table}$ . Rejection of the null hypothesis implies that  $E(\Delta_j)$  is sufficiently greater than zero as to be unexplainable as sampling error and therefore the model predictions would appear not to be representative of the data.

#### Test of Variances

The other test of goodness of fit of the Minicel Model with actual data determines how the variations in the prediction, using the model, compare with the variations in the data. In each data base, the data points in each cell are the replications of the experiment for each set of conditions of stress level, drop height, and temperature. The sum of these variations for each cell can be written as follows:

$$T_j = \sum_{i=1}^R (G_i - \bar{G}_j)^2 \quad (V-5)$$

where

$i$  = the number of the sample in the cell being considered

$T_j$  = the sum of squares of the variation for cell  $j$

$R$  = the number of replications in each cell

$G_i$  = G-level value of the data point being considered

$\bar{GD}_j$  = mean value of the data values in the cell being considered from Equation (V-2).

Then  $\sigma_d^2$  can be defined as the data within-cells variance which is found by summing the variations within all the cells:

$$\sigma_d^2 = \frac{\sum_{j=1}^N T_j}{N \times df} \quad (V-6)$$

where  $df$  = degrees of freedom in each cell.

The hypothesis to be tested is whether the variance of the data base is equal to the variance in the model predictions.

The test hypothesis can be expressed as follows:

$$H_0 : \sigma_d^2 = \sigma_m^2$$

$$H_1 : \sigma_d^2 \neq \sigma_m^2$$

where

$\sigma_d^2$  = the variance in the data as given in Equation (V-6)

$\sigma_m^2$  = the variance in the predicted values from the model.

An "F" value is used to test the ratio of variances and the F statistic is defined as follows:

$$F_{\text{calc}} = \frac{\frac{\sigma_m^2}{2}}{\frac{\sigma_d^2}{N-2}} \quad (V-7)$$

The model variance is computed during the regression procedure as the Residual Mean Square in the analysis of variance of the

model being tested. The test compares  $F_{\text{calc}}$  with  $F_{\text{table}}$ .  $F_{\text{calc}}$  is computed from Equation (V-7) and  $F_{\text{table}}$  is set using  $\alpha = 0.05$  and is tabulated according to the degrees of freedom in the numerator (the degrees of freedom in the data base being considered) and the degrees of freedom of the denominator (the degrees of freedom of the model being tested).

Rejection of the null hypothesis indicates that there is a difference between the model variance,  $\sigma_d^2$ , and the data variance,  $\sigma_m^2$ , that cannot be attributed to sampling error. This implies that there is a significant difference in the variance of the model when compared with the actual data and that the model is not representative of the data.

#### Prediction Test Results

The test of means and variances were conducted on the Minicel Model, Equation (IV-8), using the 27-inch drop height data in Appendix F and the 4 and 5 inch data samples of Appendix G. The results are given in Table V and show the following:

1) 27-inch drop height data - The 27-inch drop height data are within the range of the independent variables used in the Minicel Model and the null hypothesis cannot be rejected in the test of means or test of variances. This appears to indicate that there is no significant difference between the means and variances of the data and the values predicted by the model. The model appears to be a statistically valid predictor of these data.

2) 4 and 5 inch data - The test results for the 4 and 5 inch data show that the Minicel Model gives good predictions of the

TABLE V. TESTS OF MINICEL MODEL AS A PREDICTOR OF 4 AND 5 INCH THICKNESS AND 27 INCH DROP HEIGHT

	Minicel Material ( $\alpha = 0.05$ )		
	27 inch Drop Height	4 inch Thickness	5 inch Thickness
<u>Test of Means</u>			
Number of cells in the data base (N)	99	156	156
Standard deviation of the cells (S)	20.14	8.63	8.56
$t_m$	1.84	-1.45	-2.61
$t_{table}$	$\pm 1.96$	$\pm 1.96$	$\pm 1.96$
Decision on $H_0 : E(\Delta_j) = 0$	cannot reject	cannot reject	reject
<u>Test of Variances</u>			
Variance of the data, $\sigma_d^2$	111.63	26.41	105.35
Variance of the model, $\sigma_m^2$ (Residual Mean Square, Table IV)	114.39	114.39	114.39
$F_{calc}$	1.02	4.33	1.08
$F_{table}$	$F_{1511,198} = 1.17$	$F_{1511,312} = 1.15$	$F_{1511,312} = 1.15$
Decision on $H_0 : \sigma_d^2 = \sigma_m^2$	cannot reject	reject	cannot reject

data means at the 4 inch thickness. The test of means on the 4 inch showed no significant difference but the test on the 5 inch showed a significant difference which is not surprising because the model is projecting 2 inches beyond its range at the 5 inch thickness. The test of variances shows that the model variance at the 4 inch thickness is significantly larger, but this is not true at the 5 inch level. The model dispersion is comparable to the dispersion of the 5 inch data.

There were reservations as to whether the model could predict the 4 and 5 inch materials since the samples were stacked and not homogeneous and since 4 and 5 inches are beyond the ranges of the independent variables used in the Minicel Model. The results of the tests of means and variances show that the model was not completely acceptable at either the 4 inch or 5 inch thickness. However, the tests indicate that model means were comparable with the data at the 4 inch level and the variances were comparable at the 5 inch level.

## Chapter VI

### OPTIMIZATION

The procedure which was designed in this research to perform cushion system optimization can best be described as a constrained sequential search technique. The technique uses the knowledge that all dynamic cushioning curves are U shaped, and searches for limits of acceptable G-level values along these curves. The procedure is a direct search technique in that the mathematical model of a material such as the Minicel Model, Equation (IV-8), is used directly as the objective function in the optimization procedure.

The first step in the optimization exercise is to identify the parameters in an optimal cushioning system design; then, the objective or goal of the optimization procedure must be identified. The constraints on the procedure and the objective function are formulated and finally the outputs from the optimization routine itself are identified.

#### Formatting for Optimal Cushion System Design

An optimal cushioning system is a system that provides the necessary shock isolation to the protected item at a minimum cost. Because the cost of a cushioning system is dependent upon the amount of cushioning material, the optimal cushioning system will employ the minimum thickness of cushion needed. The optimal point on a dynamic cushioning

curve such as that shown in Figure 10 is the minimum G's possible which provides the maximum protection for that particular thickness of cushioning. Therefore, it is the identification of this optimal point that must be determined in selecting the optimal cushion system.

A different dynamic cushioning curve is required to depict cushion performance for each drop height, thickness, and temperature of the cushion; the optimal point is different for each curve. When one considers the many types of cushioning materials and a different curve for each condition, it can be seen that a very large library of curves is required to present even the most likely conditions of drop height, temperature, and thickness of a few candidate cushioning materials.

A model of impact response for a cushioning material, e.g. the Minicel Model, computes the impact response directly from the values of the independent variables. This precludes the need for a library of dynamic cushioning curves in predicting impact response. It is also possible, using the cushioning models for various materials, such as the Minicel Model, to determine an optimal cushioning design for each material, if such a design exists. It is also important to provide the designers with the maximum amount of useful design information in the output.

#### Specification of the Cushion System Constraints

The development of a valid mathematical model of impact response for a particular cushioning material provides a vehicle for the application of optimization techniques. This model can be used as an objective function and explored for an optimal cushioning system. This can be done by defining cushioning system design requirements in terms of the external environment and the amount of exposure the protected item can withstand.

The specification of the external environment must contain a definition of those aspects of the environment that have an impact upon cushion system design. This would include a quantitative measure of the magnitude of the maximum shock pulse to which the system will be exposed, which is usually specified in terms of drop height. Also since temperature has a significant effect on impact response, the specification of the external environment should include the range of temperatures to which the system will be exposed ( $\theta_{\min}$ ,  $\theta_{\max}$ ).

The specification of the survivability of the protected item is given in terms of its ability to withstand shock. The maximum shock pulse, in G's, that the item can withstand is given as the fragility level of the item ( $GL_{\max}$ ).

These considerations concerning the G-levels and temperatures are incorporated into the model as input values that are utilized to constrain the optimal search to feasible solutions.

#### Construction of the Objective Function

Consider the parameters of the general cushion design model that can be expressed mathematically in functional notation as

$$G = F(\sigma_s, T, \theta, h) . \quad (VI-1)$$

For the purpose of optimal design,  $GL$  is identified as the fragility level of the protected item. The temperature parameter,  $\theta$ , must consider the range of temperatures of the external environment  $\theta_{\min}$  to  $\theta_{\max}$ . The superimposed dynamic cushioning curve format, shown in Figure 2, can be used for this purpose wherein the curves of extremes of the temperature range are superimposed upon the ambient curve.

One parameter in the model, drop height ( $h$ ), can be determined based on published testing standards. Other parameters, such as the fragility level of the item to be protected ( $GL_{max}$ ) and the temperature range ( $\theta_{min}, \theta_{max}$ ), are usually specified by overall system requirements. For example, if the cushioning system is to be designed for a military container for a missile, the fragility level of the protected item ( $GL_{max}$ ) will be specified as part of the hardware specifications in the missile system design criteria. The temperature range ( $\theta_{min}, \theta_{max}$ ) is defined in the missile system requirements and the drop height is specified in various Military Standards depending on total container weight. These parameters,  $GL_{max}, \theta_{min}, \theta_{max}$ , and  $h$ , are the exogenous variables in the optimization model. Equation (VI-1) can now be written as follows:

$$GL_{max} = F[\sigma_s, T, (\theta_{min}, \theta_{max}), h] \quad (VI-2)$$

where  $GL_{max}$ ,  $(\theta_{min}, \theta_{max})$ , and  $h$  are inputs to the equation which are determined by the cushion system design requirements. The optimization technique will search out the optimal design from this expression, and the optimal design solution will be expressed in terms of  $\sigma_s$  and  $T$ .

The optimal design searches upon the functional relationship expressed in Equation (VI-2) and with the inputs introduced, reduces to the selection of the minimum thickness of cushion that will perform satisfactorily at a static stress condition that is determined in the search. Classical search techniques would establish Equation (VI-2) as the model to use in the search procedure and it would be rearranged with  $T$  as the dependent variable and the objective function as follows:

$$T_{min} = F[\sigma_s, GL_{max}, \theta, h] \quad . \quad (VI-3)$$

The search would be subject to certain constraints such as

$[\theta_{\min} < \theta < \theta_{\max}]$  and  $[GL \leq GL_{\max}]$ , and a search made for  $T_{\min}$  using one of many search techniques.

However, it is not apparent how Equation (VI-2) can be rearranged mathematically into the simple form of Equation (VI-3) where  $T$  is solved for directly, nor is it necessarily desirable to do so. It is important to recognize that when the exogenous variables are introduced into the objective function, a three-dimensional search on  $\sigma_s$  and  $T$  is all that is required. Since the general shape of the curve of  $G$  versus  $\sigma_s$  is known, i.e., it is a classical U shaped dynamic cushioning curve, Equation (VI-2) can be searched directly with little difficulty and with reasonable efficiency.

Once Equation (VI-1) is rewritten as Equation (VI-2) with the values of  $h$ ,  $(\theta_{\min}, \theta_{\max})$ , and  $GL_{\max}$  introduced, we have an objective function that can be written as follows:

$$GL_{\max} = F(\sigma_s, T, (\theta_{\min}, \theta_{\max}), h)$$

where  $GL_{\max}$ ,  $\theta_{\min}$ ,  $\theta_{\max}$ , and  $h$  are input constraints. This objective function is then searched using the direct search routine.

#### The Direct Search Routine

The direct search routine (CUSHION OPT) is a three stage process. The initial stage involves the selection of a material type. It is anticipated that a data base will eventually be available that contains valid impact response models for many of the different types of bulk cushioning materials. The model of each type of material in the data

base can be investigated by the direct search routine to determine the feasibility of meeting the design objective function. The optimal design conditions of minimum thickness and acceptable static stress range will be output for each type of cushion in the data base if an optimal design exists.

The initial step is to select the type of material. Then, a search is initiated on a minimum thickness cushion (1 inch) at the drop height and minimum temperature ( $\theta_{\min}$ ) stipulated in the design requirements. These values are input into the objective function and the search is conducted across the static stress spectrum at the temperature extreme,  $\theta_{\min}$  and then  $\theta_{\max}$ , to determine the feasibility of meeting the stipulated  $GL_{\max}$ . It is also necessary that the acceptable static stress range at  $GL_{\max}$  be greater than 0.2 psi to permit design flexibility and preclude creep problems within the design. (This 0.2 psi value can be changed at the discretion of the designer.).

If the search of the acceptable static stress range on the minimum cushion thickness is successful, the answer is output as a feasible solution. If it is not successful, the thickness is incremented 1/2 inch and the search repeated. A flow chart of the CUSHION OPT search routine is given in Figure 14 and the program code is given in Appendix H.

### Results

The CUSHION OPT Program output is given in the form of superimposed dynamic cushioning curves such as the typical one given in Figure 15. In Figure 15, the design objective function had the temperature range of -65° to 160°F and a 30-inch drop height. The fragility level was

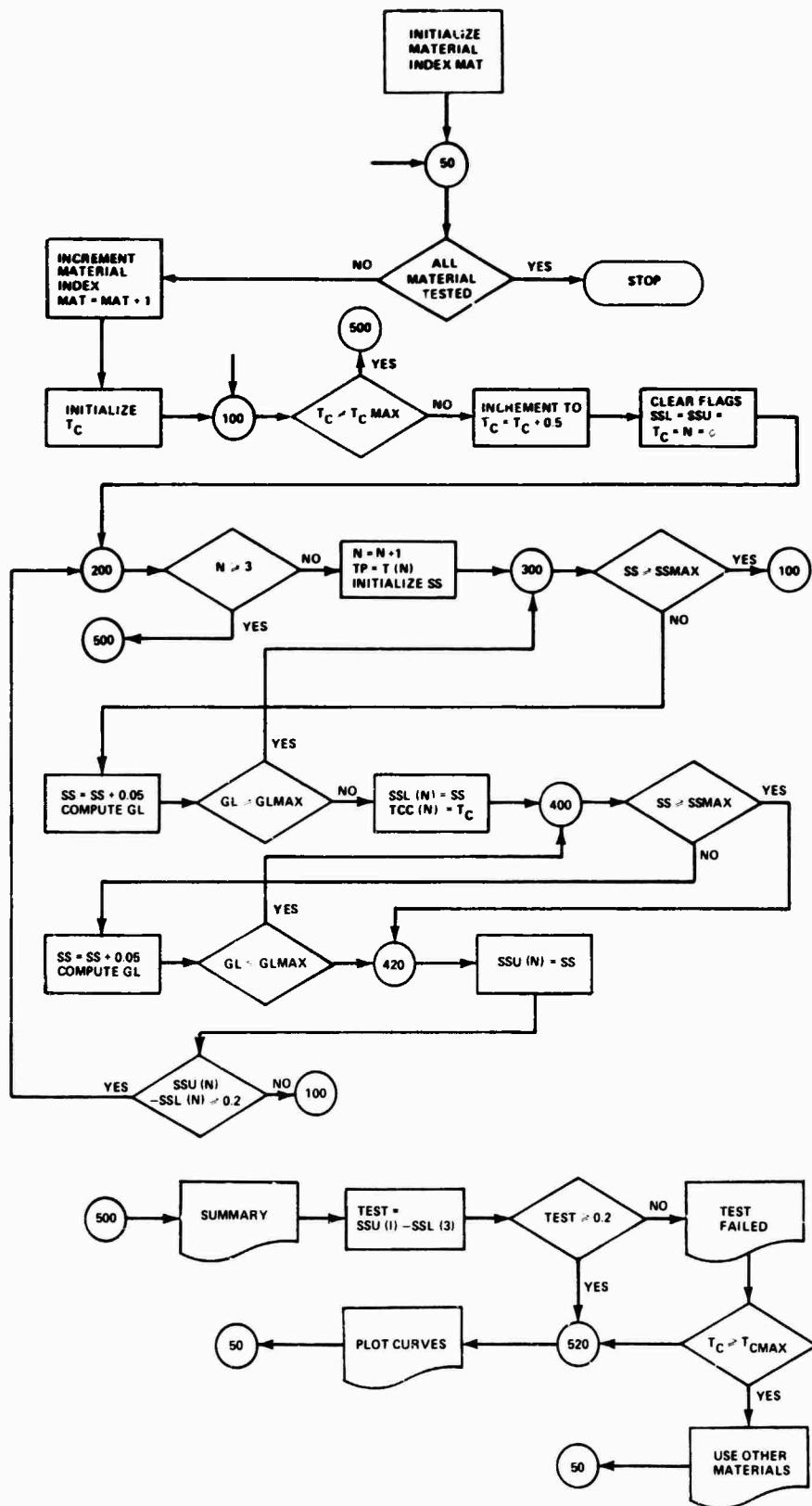


Figure 14. Flow chart of optimization direct search routine (CUSHION OPT).

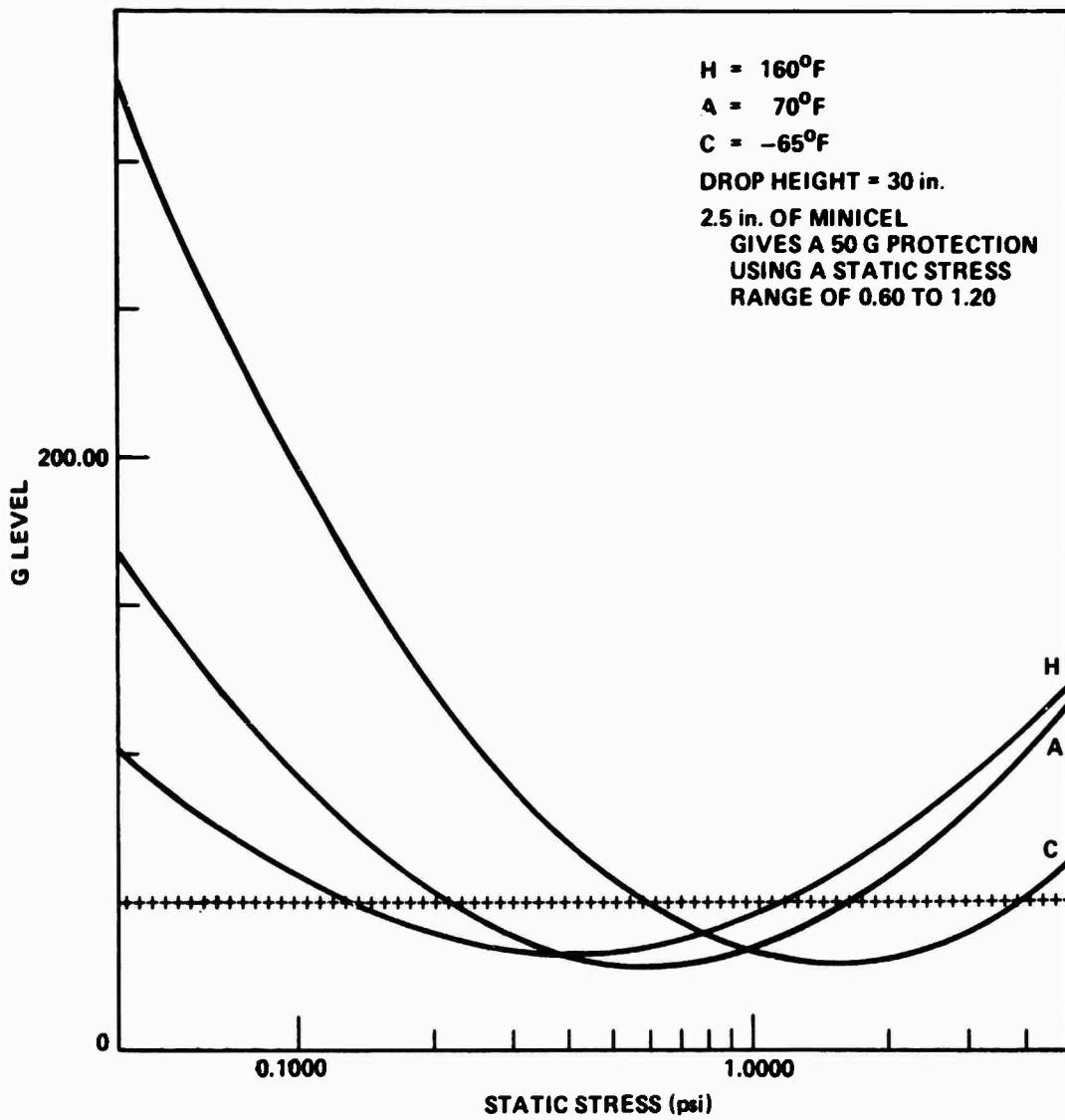


Figure 15. Optimal design output, 2-1/2 in. Minicel.

50 G's which is indicated with a dotted horizontal line. The optimal thickness is determined to be 2.5 inches and the feasible static stress range is found to be 0.60 to 1.20 psi. One of the inherent advantages of using math models is seen here in the capability of determining G-level response at thicknesses other than those in the data base. In this instance a 2.5-inch thickness of cushion was optimal; tests were not conducted at this thickness in the data base. This advantage can also be seen in Figure 16 which is another output of the optimization program where the temperature range has the non-data base values of -20° and 120°F. The optimal thickness is 2.0 inches and the stress range is 0.55 to 0.85 psi to give 50 G protection. Comparison of Figure 15 with Figure 16 demonstrates the effect of temperature on optimal cushion design. When the temperature range was relaxed from the extremes of -65° through 160°F to -20° through 120°F and all other exogenous variables kept the same, the thickness of cushion required for 50 G protection dropped from 2.5 inches to 2 inches. Another comparison can be made between Figure 15 and Figure 17, which demonstrates the increase in thickness of cushion required when the fragility level of the protected item is lowered from 50 G's to 40 G's. One other comparison can be made between Figure 15 and Figure 18, which demonstrates how a reduced drop height requirement reduces the thickness of cushion required.

One additional advantage in using the superimposed dynamic cushioning curve format for the output format is that the designer is presented with a convenient tool for minimizing the creep tendency of the materials by maintaining as low a stress condition as possible and yet be able to ascertain the response of the stress level selected.

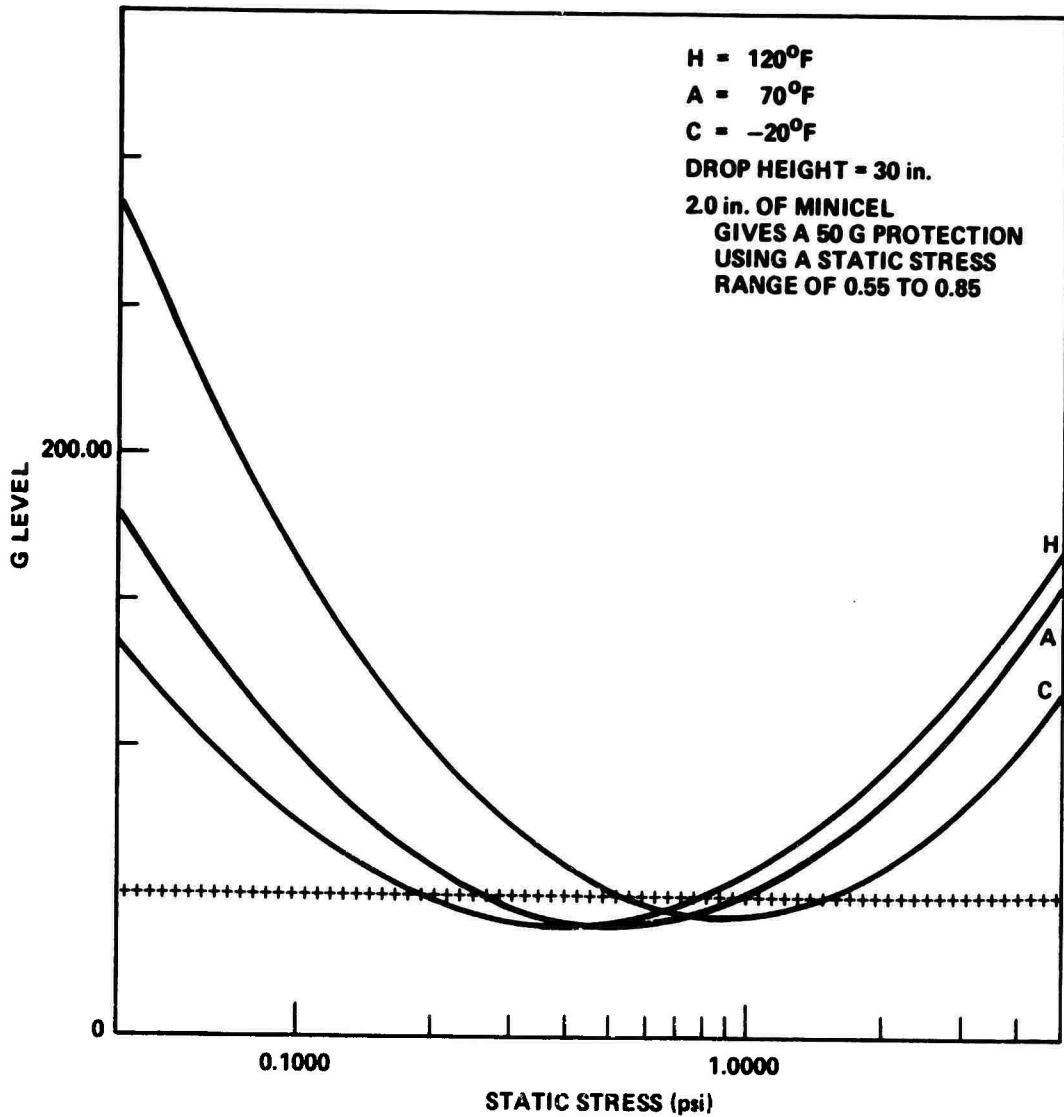


Figure 16. Optimal design output, 2 in. Minicel.

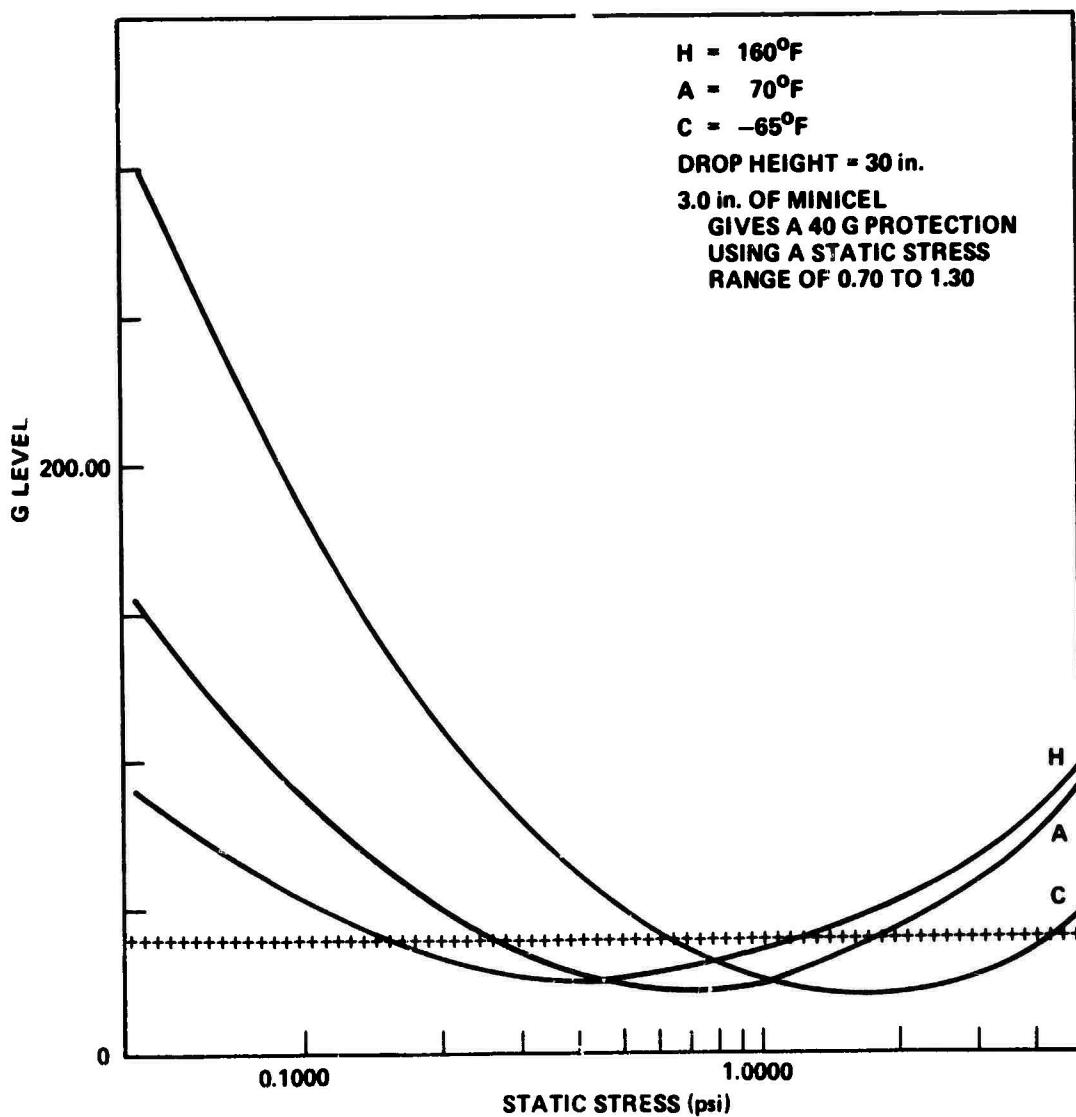


Figure 17. Optimal design output, 3 in. Minicel.

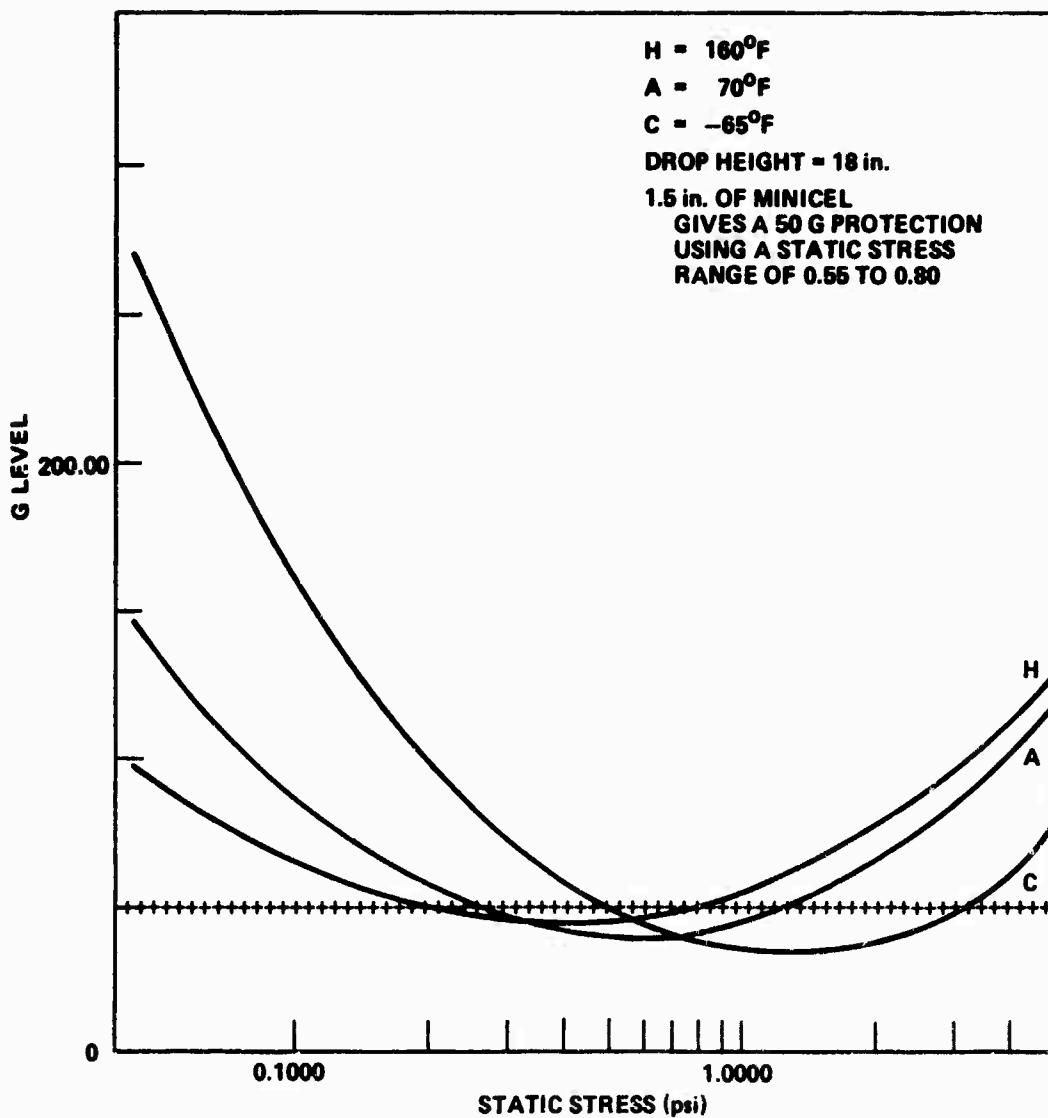


Figure 18. Optimal design output, 1-1/2 in. Minicel.

## Chapter VII

### CONCLUSIONS AND RECOMMENDATIONS

There has been limited use made of bulk cushioning in shock isolation systems for commercial and military equipment. One reservation that designers have had regarding bulk cushioning is an inability to predict cushioning performance at temperature extremes. Consequently, many designs incorporate mechanical suspension systems such as elastomeric mounts or springs and dash pots and other types of shock isolation systems that are bulky, very heavy, require substantial structural interfaces, and are very expensive.

One result of this research is the development of a valid model of bulk cushioning response that provides a basis for improving the predictability of temperature performance for bulk cushioning materials. Resultant increases in the use of bulk cushioning for shock mitigation systems will generate dollar savings in the accompanying reduced procurement and development costs.

#### Conclusions

The developed General Model of the impact response of bulk cushioning materials is

$$G = C_0 + \sum_{\ell=0}^S h^{\ell/2} \sum_{k=0}^R \frac{1}{T^{(1/2+K)}} \sum_{j=1}^N \theta^j \sum_{i=1}^M c_{ijkl} (\ln \sigma_s)^i . \quad (VII-1)$$

The model is predicated on viscoelastic theory and incorporates the effect of temperature, stress, drop height, and thickness of cushion upon the impact response of a cushioning system. This General Model provides the basic underlying structure of impact response of any one of the many bulk cushioning materials used for shock isolation. A sensitivity analysis can be run on the values of S, R, N, and M to obtain the precision desired for an impact model of a particular cushioning material.

Models that are predicated on the basic underlying structure of the General Model are better predictors of impact response than the dynamic cushioning curves currently being utilized, because the effect of temperature has been incorporated into the model. The Minicel Model is one such model that was constructed for the Hercules, Inc. 2 lb/ $\text{ft}^3$  Minicel material using the General Model as the basic underlying structure. The Minicel Model is a 25-term polynomial given in Equation (IV-8). The correlation of the Minicel Model with actual data demonstrates the validity of the models and their value as predictors of impact response. The Minicel Model showed high correlation with the actual data within the ranges of the variables and also showed promise as a predictor beyond the variable ranges.

The development of a valid model of impact response of bulk cushioning materials defines the relationships and interrelationships of the variables in response to impact. Using this basis, it was possible to employ a search technique (CUSHION OPT) to determine optimal cushion design and display the findings in terms of superimposed dynamic cushioning curves. The functional form of the model provides the advantage of determining impact response at non-tested levels of the

variables with confidence and precludes the need for a library of dynamic cushioning curves for all the different combinations of conditions.

Once a valid model of a particular cushioning material has been developed, it can be incorporated into the CUSHION OPT optimization program. This program accepts the design requirements for a shock isolation system and computes and provides, in the form of superimposed dynamic cushioning curves, the optimal design for each cushioning material in the data base, if one exists. The outputted superimposed dynamic cushioning curves give the pertinent information needed in the optimal design of a shock isolation system.

#### Recommendations

It is recommended that the model of impact response of bulk cushioning materials, Equation (VII-1), be used as the basic underlying structural design for cushioning systems. This model is considerably better than any previous basis of design. The optimization program used in conjunction with the model can be used to provide accurate predictions of shock mitigation system performance in a time saving manner and in a useful format. It is reasonable to expect that in the design of shock isolation systems, considerable savings can be realized in design time and cost savings by using these more accurate response predictions prior to prototype fabrication and test.

Further, it is recommended that the optimal design program, CUSHION OPT, be used to generate the optimal design of bulk cushioning systems. Cushioning system designers that have a large scale digital computer capability should be encouraged to utilize this procedure. As models

of additional cushioning materials become available, CUSHION OPT is provisioned to permit updating to incorporate additional design alternatives using the new materials. To insure maximum utilization of the improved predictive capability of the General Model, it is further recommended that superimposed dynamic cushioning curves of the most likely conditions of the independent variables ( $\epsilon$ ,  $\sigma_s$ ,  $h$ ,  $T$ ) be published and made available to cushioning system designers who do not have access to a computer facility. It is recognized that one of the major advantages inherent in the CUSHION OPT program, that of obtaining the exact design constraints needed, becomes inoperative and therefore, manual interpolation will be necessary when using published curves. However, the difficulties associated with using and maintaining a library of superimposed dynamic cushioning curves are warranted when the savings accrued through the use of the improved predictions are considered.

Additional drop test programs, similar to the one conducted on Minicel, should be conducted on the bulk cushioning materials. A model of impact response similar to the Minicel Model, Equation (IV-8), using the General Model as the basic underlying structure should be constructed for each new material. This model can then be entered into the CUSHION OPT program to provide an additional optimal design alternative in terms of this new material. It is further recommended that the drop test programs on additional materials be conducted using levels of the independent variables ( $\epsilon$ ,  $\sigma_s$ ,  $h$ ,  $T$ ) that bracket the values of the variables that might occur in cushion system design. This increase in the range of the variables would insure that the model is predicting values within the range of the model. Particular attention should be

given to insuring that the range of thickness is sufficient to obtain G-levels as low as 10 G's, which is not uncommon in some of the more fragile optical and electronic hardware. Also, since only the lower portions of dynamic cushioning curves are used in optimizing cushion design, the test programs can be abbreviated in the understressed and overstressed regions.

It is also recommended that the General Model of impact response be considered as a basis for future research. The mathematical formulation of impact response should provide a vehicle to be utilized for the rigorous analysis of impact response. For example, one particularly lucrative area is that the model be used as a phenomenological constitutive equation in advancing the viscoelastic theory of bulk cushioning materials.

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**Appendix A**

**UAH DATA, 12, 18, 24 AND 30 INCH**

MINICEL - 12 in. Drop Height

STRESS LEVELS (PSI)

		Temperature (°F)																																											
		2.4					3.0					3.6					4.0					4.4					4.6																		
		-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	Thickness																
1	35	59	72	45	75	78	48	89	94	52	81	86*	57	79*	110	61	107*	94	59	106	124*	1	38	57	70	46	73	84	43	77*	82*	61	81	101	65	90	85*	71	96	106	61	100	114	1"	
2	32	65*	75	50	73	79	48	86	92	56	80	102	68	95	104	70	95	100	69	89*	109	2"	3	18	24	27	18	25	28	16	27	35	18	27	37	16	31	39	24	36	41	24	40	43	3"
1	16	25	30	12	22	32*	18	28	31	15	29	35	12	24*	36	21	37	39	23	37	45	1	16	24	25	12	23	25	15	29	31	19	30	33	15	34	41	23	36	41	26	39	40	2"	
2	16	24	25	12	23	25	15	29	31	19	30	33	15	34	41	23	36	41	21	37	45	3	16	17	18	11	15	17	7	14	18	9	18	20	10	19	20	11	16	25	3"				
1	18	14	16	12	15	17	10	18	18	9	15	17	12	15	18	10	20	23	13	20	49*	1	12	13	15	10	14	17	9	13	20	12	9*	15	11	15	18	9	17	21	16	25			
2	16	16	17	8	11	11	10	15	17	7	15	17	7	14	18	9	18	20	10	19	20	3	16	16	17	8	11	11	10	15	17	7	15	17	7	14	18	9	18	20	11	16			

## MINICEL - 18 in. Drop Height

## STRESS LEVELS (PSI)

	0.04	0.10	0.20	0.40	0.80	1.0	1.6	2.0
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	Temperatures (°F)																							
	-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	Thickness								
1	248	187	113	182	91	77	107	66	65	62	57	70	44	59	81	46	67	84	56	93	107	59	100	115
2	252	200	120	179	95	72	113	61	66	57	61	66	45	66	79	42	72	85	58	95	117	68	96	118
3	250	200	119	183	90	74	103	63	63	56	54	58	45	69	78	45	67	79	54	91	100	68	99	120
1	259*	166*	86*	166	65	52	88	48	37	50	39	35	31	33	35	26	30	56*	25	36	43	28	42	44
2	250*	216*	105*	166	65	51	87	42	39	46	33	36	31	35	43	27	30	35	24	36	40	26	37	45
3	260	128	91	157	67	53	73*	43	37	41*	35	40	33	32	37	27	30	36	26	35	44	28	38	48
1	169	112	67	118*	56	41	92*	38	38	48	29	28	28	29*	27	15	14	24	22	16	23	19	22	24
2	198	142*	73	153	54	43	76	34	31	44	26	25	25	27	22	22	17	19	20	18	23	22	19	28
3	242*	125	71	148	49	42	66	40	32	41	27	25	32	30	27	24	21	19	27	21	19	24	18	23

	2.4	3.0	3.6	4.0												
	Temperatures (°F)															
	-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	Thickness
1	64*	128	134	100	146	174	104	175	164	131	187	210	124	166*	211	1"
2	78	107*	133	99	143	163	124*	161	173	100*	189	202	100*	189	202	
3	73	126	139	98	137	165	99	168	206*	189	202					
1	27	44	48	32	52	63	38	64	68	41	73	78	41	73	78	
2	26	45	54	32	57	66	39	59	70	38	61*	84	2"			
3	29	34*	54	31	57	59	37	61	75	35	68	72				
1	15	22	19*	18	24	32	18	29	38	25	38	39	3"			
2	12	20	31	15	29	33	15	29	33	21	38	38				
3	15	21	28	12	27	34	19	30	35	21	34	37				

## MINICEL - 24 in. Drop Height

## STRESS LEVELS (PSI)

		Temperature (°F)																									
		0.04		0.08		0.10		0.20		0.40		0.60		0.80		1.0		1.4									
		-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	Thickness							
1	231*	180	114	262	122	87	200	106	90	104	77	80	66	71	89	60	83	102	66	96	122	81	123	147			
2	335	215*	121	263	127	104	195	107	87	104	73	81	63	69	86	61	82	97	60	102	108	60	101	122	150		
3	289	195	113	250*	116	104	198	106	86	97	77	85	63	68	82	61	73*	103	57	88*	120	61	86*	111	150		
1	277*	163	85	235	112	69	194*	91*	61	102	55	44	48	39	43	49	37	46	35	36	45	34	38	45	54		
2	254	202*	88	125*	101*	77	213	82	60	88	49	43	50	39	36	41	38	42	36	34	45	35	37	47	52		
3	245	149	97	270	114	75	205	81	57	93	51	46	71*	41	42	40	34	40	35	38	41	31	36	49	51		
1	220*	111*	94	233	104*	53	110*	68	50	73*	39	39	53	36	27	36	24	25	27	24	27	24	23	27			
2	262	167	86	192*	81	53	155	75*	45	87	40	32	47	33	26	37	23	27	29	25	26	24	25	22	27		
3	281	158	68*	232	82	57	169	34	44	89	42	34	46	27*	27	37	25	27	31	24	26	30	23	29	21	24	27
		1.8	2.0	2.2	2.4	2.6	2.8	3.0	3.2	3.4	3.6	3.8	4.0														
		Replicate	1	2	3	1	2	3	1	2	3	1	2	1	2	3	1	2	3	Thickness							
		1	112	176*	186	96	148	208	145	198	235	142	198	246	1	112	176*	186	96	148	208	142	198	235	141	198	246
		2	108	139	166*	92	164*	205	145	203	254	142	198	246	1	108	139	166*	92	164*	205	142	198	254	141	198	246
		3	90*	137	192	98	136	190	142	198	246	1	90*	137	192	98	136	190	142	198	246	1	90*	137	192	98	
		1	30	49	60	32	16*	64	41	81*	86	47	81	95	52	86	95	45	83	93	57	108	114	68	107	116	
		2	32	50	65	32	54	66	46	76	92	44	73	90	48	88	99	45	84	92	58	103	107	75	106	116	
		3	33	49	60	30	43	62	42	76	98	49	70	100	102*	87	100	57	78	101	44	99	102	59	100	115	
		1	21	25	30	23	26	34	34	45	22	26	45	21	42	47	21	40	49	22	46	51	30	55	63		
		2	21	27	35	22	23	32	34	48	18	28	49	24	46	49	20	36	51	20	43	54	31	58	65		
		3	20	27	31	19	25	33	23	38	50	24	25	51	27	44	51	31*	41	43	22	43	49	26	56	52*	

## MINICELL - 30 in. Drop Height

## STRESS LEVELS (PSI)

		Temperature (°F)																				
		0.04	0.08	0.1	0.2	0.3	0.4	0.6														
		-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	Thickness		
1	365*	208*	143	264	133	118	240	115	100	110	91	98	84	95	102	76	90	120	75	98	144	
2	334	195	136	271	129	113	220	100	112	112	88	96	84	93	101	85	91	116	72	106	127	
3	321	187	137	263	134	109	210	100	105	116	92	99	90	93	98	78	90	120	67	103	138	
Replication	1	372	173	91	258	99	76	244*	95	65	98	64	54	84	46	49	54	46	53	48	54	
2	369	169	96	241*	104	76	220	80*	72	106	56	55	82	49	52	57	46	47	47	45	50	
3	369	192*	106*	269	56*	82	215	88	72	88*	58	53	91	54	49	57	49	53	46	42	55	
Replication	1	330*	177*	106*	278*	90	60	184	64	56	77	52*	41	71	36	33	50	32	31	38	30	31
2	295	166	89	265	90	57	181	70	58	79	45	39	60*	37	35	52	36	32	43	29	34	
3	301	171	81	257	84	58	195*	80*	57	87*	43	41	76	33	33	46*	35	33	46	30	34	

		Temperature (°F)																				
		0.7	0.8	1.0	1.2	1.3	1.4	1.6														
		-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	Thickness		
1	71	108	150	66	113*	170*	79	136	182	105	181	212	99	203*	242	108	201	222	140	217*	287	
2	70	119	147	76	133	145	86	170*	161*	88	173	213	102	178	229	107	206	263*	111	234	264	
3	69	114	145	77	130	154	78	149	191	88	139*	211	97	182	228	125	204	233	123	232	256	
Replication	1	41	46	53	41	48	61	38	53	63	37	51	67	34	57	68	32	60	83	35	73	89
2	42	46	56	39	43	58	40	54	62	36	52	72	35	54	73	39	55	77	41	70	41*	
3	47	47	58	40	48	55	40	57	65	37	56	69	35	63	69	40	62	79	37	66	81	
Replication	1	37	30	29	36	31	33	35	32	36	27	33	36	25	34	35	27	32	37	23	35	44
2	38	28	32	30*	32	33	31	29	34	27	31	35	24	32	39	25	29	39	24	33	43	
3	36	29	31	35	30	34	32	32	34	26	28	35	27	32	37	26	32	36	26	31	43	

**MINICEL - 30 in. Drop Height  
(Continued)**

\*These values were removed by the outlier procedure during the UAH analysis.

CORRELATION COEFFICIENTS OF THE BEST FITTING  
POLYNOMIALS IN THE UAH STUDY

Temperature (°F)	Thickness (in.)	Drop Height			
		12 in.	18 in.	24 in.	30 in.
-65	1	0.97	0.98	0.98	0.97
	2	0.99	0.99	0.96	0.99
	3	0.99	0.98	0.99	0.98
70	1	0.98	0.98	0.99	0.98
	2	0.98	0.96	0.97	0.98
	3	0.97	0.98	0.97	0.97
160	1	0.98	0.98	0.97	0.98
	2	0.97	0.97	0.94	0.97
	3	0.93	0.98	0.96	0.97

NOTE: This table gives the sample correlation coefficients from an analysis of regression variance between the data and a polynomial fit of the form

$$y_i = b_0 + b_1 \ln x_i + b_2 (\ln x_i)^2.$$

**SELECTED BEST FITTING POLYNOMIALS IN THE UAH STUDY  
(Hercules Minicel, 2 lb/ft<sup>3</sup> Density)**

Thickness (in.)	Temperature (°F)	Design Curve Equation
12 in. Drop Height		
1	-65	$y = 377.74 - 142.48 \ln x + 14.78 (\ln x)^2$
	70	$y = 278.24 - 118.34 \ln x + 14.44 (\ln x)^2$
	160	$y = 197.11 - 84.21 \ln x + 11.27 (\ln x)^2$
2	-65	$y = 367.89 - 131.51 \ln x + 12.22 (\ln x)^2$
	70	$y = 201.97 - 78.34 \ln x + 8.34 (\ln x)^2$
	160	$y = 142.03 - 55.43 \ln x + 6.31 (\ln x)^2$
3	-65	$y = 329.67 - 118.13 \ln x + 10.86 (\ln x)^2$
	70	$y = 159.93 - 58.41 \ln x + 5.77 (\ln x)^2$
	160	$y = 105.43 - 36.66 \ln x + 3.73 (\ln x)^2$
24 in. Drop Height		
1	-65	$y = 691.02 - 301.76 \ln x + 36.16 (\ln x)^2$
	70	$y = 403.76 - 193.48 \ln x + 27.86 (\ln x)^2$
	160	$y = 280.51 - 141.90 \ln x + 23.96 (\ln x)^2$
2	-65	$y = 544.94 - 210.69 \ln x + 21.69 (\ln x)^2$
	70	$y = 333.50 - 150.03 \ln x + 18.71 (\ln x)^2$
	160	$y = 202.97 - 93.58 \ln x + 13.17 (\ln x)^2$
3	-65	$y = 517.16 - 194.42 \ln x + 18.94 (\ln x)^2$
	70	$y = 289.25 - 123.01 \ln x + 13.98 (\ln x)^2$
	160	$y = 170.46 - 73.58 \ln x + 9.21 (\ln x)^2$

**Appendix B**  
**STEPWISE REGRESSION PROGRAM LISTING**

74/74 OPT=1 FTN 4.2+74278 11/22/74 18.10.21.  
 C STEPHIS MULTIPLE LINEAR REGRESSION  
 C WRITTEN BY WAYNE L. JONES, REEDSTONE ARSENAL, ALABAMA  
 C BASED UPON PROCEDURES IN DRAPER'S APPLIED REGRESSION ANALYSIS  
 5 AND SHARE NUMBER 1333  
 C TAPES 11 AND 10 ARE USED AS BINARY INPUT TAPES.  
 C TAPES 9 AND 10 ARE USED AS WORK TAPES.  
 C\*\*\*\*\*  
 C A. MLP CONTROL CARD 1 FORMAT 1A15 1 \*\*\*\*\*  
 C 01-35 NPROF = NUMBER TO IDENTIFY PROBLEM.  
 C 76-10 NXV = TOTAL NUMBER OF INDEPENDENT VARIABLES IN INPUT DATA.  
 C 11-15 NV = TOTAL NUMBER OF DEPENDENT VARIABLES IN INPUT DATA.  
 C 16-20 INDEXY = INDEX OF THE DEPENDENT VARIABLE FOR THE PRCBLK.  
 C 21-25 NDATA = TOTAL NUMBER OF DATA OBSERVATIONS FOR THE PRCBLK.  
 C IF UNKNOWN- SET EQUAL MAXIMUM EFFECTED AND SET LAST  
 C DATA OBSERVATION EQUAL TC 9999999.  
 C 26-30 IDEN = NUMBER OF ALPHABETIC HEADER CARDS (SEE C ).  
 C 31-35 INTYPF = 0 FOR REGULAR RUN WITH DATA ON CARDS.  
 C 1 TO REWIND 10 AND STORE CARD DATA FOR LATER EXECLEM.  
 C 2 TO REWIND 10 AND USE DATA STORED BY A PREVIOUS EXEC.  
 C 3 TO STORE CARD DATA ON TAPE 10 WITHOUT REWINDING.  
 C 4 TO USE DATA ON TAPE 10 WITHOUT FIRST REWINDING.  
 C 5 TO USE TAPE 11 AS INPUT AFTER REWINDING.  
 C 6 TO USE TAPE 11 AS INPUT WITHOUT REMINING.  
 C 7 REWIND 11. USE AS INPUT. THEN REWIND FOR LATER USE.  
 C 36-40 NRFAPI = 0 TO USE DATA WITHOUT REARRANGING IT.  
 C 1 TO REARRANGE DATA ACCORDING TO CONTROL CARD F.  
 C 41-45 MAXSTP = MAXIMUM NUMBER OF STEPS OR ITERATIONS ALLOWED.  
 C TO BYPASS PRINTOUT OF CALCULATIONS PRIOR TO SUMMARY.  
 C 51-55 NSTAPT = NUMBER OF INDEPENDENT VARIABLES THAT YCL ISH IC STAR1  
 C SET EQUAL TO 999. NORMAL VALUE IS 0.  
 C 46-50 IFEACK = STEP AT WHICH BACK SOLUTION STARTS (ACTUAL VS. PREC.).  
 C SET EQUAL TO 0 FOR NO BACK SOLUTION.  
 C SET EQUAL TO 999 FOR BACK SOLUTION OF SUMMARY CALLY.  
 C NOTE - IF NO DATA(NXV=1) IS GREATER THAN 3000, TAPE 9  
 C IS USED TO STORE DATA THEREBY INCREASING RL TIME.  
 C 56-60 MINSUM = MIN NBR OF IND VAR IN SUMMARY O/P. NORMAL VALUE IS 1.  
 C 61-65 MAXSUM = MAX NBR OF IND VAR IN SUMMARY O/P. NORMAL VALUE IS NV.  
 C 66-70 MAXREG = MAX NBR OF IND VAR IN REGRESSION. NORMAL VALUE IS NV.  
 C\*\*\*\*\*  
 C 9. MLP CONTROL CARD 2 \*\*\*\*\*  
 C 01-35 IFWT = 0 FOR UNWEIGHTED DATA  
 C = 1 WEIGHTS ARE READ IN AS INPUT.  
 C 06-10 IFCNST = 0 IF CONSTANT TERM IS TO BE CALCULATED.  
 C = 1 TO DELETE CONSTANT TERM  
 C = -1 IF CONST TERM IS TO BE CONSIDERED AS THE COEFFICIENT  
 C OF A NEW INDEPENDENT VARIABLE X0 WHICH ALWAYS HAS THE  
 C VALUE 1. THE SIGNIFICANCE OF THE CONSTANT WILL BE  
 C INDICATED BY ITS STANDARD ERROR.  
 C 11-15 IFLIST = 0 TO LIST SUM(XI\*Xi) & OTHERWISE.  
 C 16-20 IFSURS = 0 TO LIST SUM(XI-XBAR)(XJ-XBAR). 1 OTHERWISE.  
 C 21-25 IFRFS = 0 TO LIST SUM(XI-XBAR)^2. OTHERWISE.

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74/74   OPT=1           FTN 4.2+74270   11/22/74  12:16:21.

C 26-30 IFCCEN = 3 IC LIST SIMPLE CORRELATION COEFFICIENTS. 1 CTIME=ISE.
C 31-35 ACROSS = 1 IF YOU WISH TO INCLUDE AS ACCIDENTAL INEFFECTIVE
C VARIABLES. THE SQUARES AND THE CROSS PRODUCTS OF
C THE INEFFECTIVE VARIABLES. ZERO OTHERWISE.
C   VARIABLES ARE X1(XV+1)*X1*X1
C   X(NXV+2)=X1*X2, X(NXV+3)=X1*X3, ..., X(NXV+N)=X1*XN
C   X(NXV+N1)=X2*X2, X(NXV+N2)=X2*X3, ..., ETC.
C   = N (GREATER THAN 1) FOR POLYNOMIAL CURVE FIT OF ORDER N.
C NOTE - ISTART SHOULD BE SET TO -2 FOR MCPAL RUN.
C 36-40 IFTRA = ALLOCS FOR TRANSFORMATIONS OF INPUT DATA. SEE G FCS USE.
C   0 FOR NO TRANSFORMATIONS.
C   = 1 FOR TRANSFORMATIONS.
C 41-45 NVOID = 0 TO PROCESS ALL OBSERVATIONS. = 1 TO READ LPIC 1A
C   OBSERVATIONS TO LEFT OUT OF REGRESSION. SEE CONTROL
C CARD E. = -1 TO USE PREVIOUS E CARD.
C 46-50 IFSUP = ZFRC FOR NORMAL RUN POSITIVE VALUE CALLS IN A
C   USER SUPPLIED SUBROUTINE CALLED NEROPAISUB. ALSO
C   TO CHANGE NXV (NOTE - NOFRC MUST BE SUPPLIED EVEN
C   IF IT IS JUST A RETURN). A USER SUPPLIED SUBROUTINE
C   CALLED EQUATIFSUB,DATA IS USED TO NAME THE DESIREC
C   CROSS PRODUCT AND TRANSFORMATIONS (NOTE - DEPENDENT
C   VARIABLE SHOULD BE DEFINED AS THE VARIABLE DATA(NV+1)
C   WHERE DATA IS A SET OF OBSERVATIONS GOING IN IC 1+E
C   S/R AND THE TRANSFORMED SET COMING OUT).
C 51-55 MFMT = 0 FOR REGULAR INPUT FORMAT F10.0.
C   = 1 TO READ INPUT FORMAT 1.
C 56-60 IFPNCH = -1 TO USE FORMAT FROM PREVIOUS RUN.
C 61-65 IFOATF = 0 TO OBLIGE PUNCHING OF EQUATIONAL COEFFICIENTS IN SUMMARY.
C 66-70 IFNAME = 0 TO READ NAME OF COMPUTER RUN.
C   = -1 TO USE PREVIOUS H CARO. STILL IN CORE.
C 90
C*****+
C C. ALPHABETIC HEADER CARDS. DO NOT USE IF IOEN=0
C IDE H CARDS WITH FORMAT 16(5) LAST CARC REPAEAD ON FACH PAGE.
C*****+
95 C D. CARDS FOR VARIABLES IN REGRESSION AT START AND CORRESPONDING TESTS.
C   DO NOT USE IF ISTART=0. THEREF SHOULD BE "NSTART" FIELDS 7(IFB+1,I2)
C   31-09 TEST(1) = A TEST CONDITION WHICH INDICATES WHETHER A VARIABLE
C   WILL BE OLEAFD OR ADDED TO THE REGRESSION. ITS
C   VALUE IS 1-R#2. ZERO CORRESPONDS TO A MULTIPLE CORR
C   COEFFICIENT OF 1. WHICH MAKES IT IMPOSSIBLE FOR THE
C   PROGRAM TO DELTE THAT VARIABLE FROM THE SET OF INC
C   VARIABLES. TEST=1 CORRESPONCS TO MULT CORR COEF CF 0.
C   WHICH MAKES SUCH A DELITION CERTAIN.
C 19-20 INDFX(1)= FIRST VARIABLE TO BE INCLUDED IN REGRESSION AT START.
C 11-19 TEST (2)= TEST FOR TWO VARIABLE SET.
C 19-20 INDUX(2)= SECOND VARIABLE TO BE INCLUDED IN REGRESSION AT START.
C 21-29 TEST (3)= TEST FOR THREE VARIABLE SET.
C 29-30 INDU(3)= FTC.
C*****+
110 C E. OBSERVATIONS TO BE REMOVED FROM REGRESSION. USE ONLY IF NVOIC1 .
C 01-05 NOGOC(1) = INDEX OF 1ST POINT TO BE REMOVED. (FORMAT 16(5))
C 06-10 NOGOOD(2) = ETC.
C*****+

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74/74   CPT=1          FTN 4.2+74278    11/22/74  10.10.21.

115      C F. CONTROL CARD TO REARRANGE MLR DATA.  DO NOT USE IF NREAD=0. (11415)
C 01-05 NWRCRS = NUMBER OF WORDS IN TAPE OR CARD RECORD.
C 06-10 LOLV = LOCATION OF DEPENDENT VARIABLE Y.
C 10-15 LOCX(J), J=1,NVY = LOCATIONS OF INDEPENDENT VARIABLES.
C           IF IINP>0 LAST LOCATION IS FOR WEIGHTS.
C*****.
120      C G. TRANSFORMATION CONTROLS.  DO NOT USE IF ITRANS=0. FORMAT(17(F0.0,I2)).
C       PUT TRANSFORMATIONS AND CORRESPONDING CONSTANTS IN SAME CDEF AS
C       X AND Y VARIABLES.
C       TRANSFORMATIONS 0=NONE, 1=X+C, 2=X*C, 3=X/C, 4=G/X, 5=X**C,
C       6=C*X, 7=LN(X+C), 8=LOG(X+C), 9=E**C*X, 10=E*(C/X),
C       11=SQRT(C*X), 12=COS(C*X), 13=TAN(C*X).
C 01-04 CONST = CONSTANT FOR FIRST VARIABLE
C 09-10 NBSTRA = TRANSFORMATION FOR FIRST VARIABLE
C 11-18 = CONST FOR SECOND VARIABLE. * ETC.
C 19-20 = TRA FOR SECOND VARIABLE. * ETC.
C*****.
130      C H. INPUT VARIABLE NAMES IN ORDER OF INPUT.  USE ONLY IF IFNAME=C.
C 01-10 NAME OF FIRST INPUT VARIABLE
C 11-0 = NAME OF SECOND INPUT VARIABLE. ETC.  FORMAT(7IA6,A4)  .
C*****.
135      C I. VARIABLE FORMAT FOR INPUT DATA (122A6).  USE C ONLY IF NFMT=1.
C*****.
140      C J. MLR DATA CARDS SHOULD BE PUNCHED WITH FORMAT(17F10.0), OBSERVATION
C       BY OBSERVATION IN THE FOLLOWING ORDER (IF INTYPE=0,1) X1, X2, X3,
C       X4, *** XNV, Y1, Y2, Y3, *** MNV, W1 IF IFN=1).
C       DATA CARDS FOR INTYPE=0,1,3 ONLY. BINARY TAPE INPUT FOR INTYPE COSES.
C       IF NREAD=1 ORDER OF DATA IS DETERMINED BY CONTROL CARD F.
C       OBSERVATIONS WITH BLANK DATA (-0-) ARE REJECTED FROM REGRESSION.
C*****.
145      C TO USE THE PROGRAM FOR AN ORDINARY MULTIPLE REGRESSION (IE, MC
C       ACCORDING CR CR CELTING), PUT ALL VARIABLES IN THE REGRESSION AT
C       THE OFFSET (NSTART = -NNV) AND PUT MAXSTP = 1.
C*****.
150      C*****. CARE CUPUT ( IF IFNCH IS NOT EQUAL ZER0 )
C       ONE CARD FOR EACH VARIABLE IN EQUATION
C       FORMT(15.20.8,515,F20.4,F10.7)
155      C 01-05 = I = INDEX OF INDEPENDENT VARIABLES IN EQUATION
C 06-25 = COEFF(I1) = COEFFICIENT FOR VARIABLE I
C 26-30 = NPROB = PROBLEM NUMBER
C 31-35 = NBNOM = NUMBER OF VARIABLES IN EQUATION
C 36-43 = INDEXY = INDEX OF DEPENDENT VARIABLE
C 41-45 = NOSTEP = STEP NUMBER IN WHICH THE EQUATION WAS COMPLETED
C 46-50 = IPNCH = INPUT VALUE GREATER THAN ZERO
C 51-70 = SIGPC = STANDARD ERROR OF EQUATION AS A PERCENT OF Y MEAN
C 71-80 = REGRCO = CORRELATION COEFFICIENT OF EQUATION
C*****. BASIC STATISTICS OUTPUT ****.
165      C XI = X = VALUE OF OBSERVATION FOR VARIABLE I
C SUM( XI ) = SUMMATION OF VARIABLE I
C N = NUMBER OF OBSERVATIONS
C MN = WEIGHTED NUMBER OF OBSERVATIONS
170      C

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74/74 OPT=1          FTN 4.2+74278   11/22/74  10:10:51.

C MEAN = WEIGHTED AVERAGE = SUM(XI)/SUM(XI*(MEAN)/(XN-1) )
C STANDARD DEVIATION = SQRT((SUM(XI**2)-SUM(XI)**2)/N)
C
C 175 C SUM OF VARIABLES = SUM(XI)
C      C RAW SUM OF SQUARES AND CROSS PRODUCTS = SUM(XIXXJ)
C      C SUM OF SQUARES AND CROSS PRODUCTS ABOUT THE MEAN = CORRECTED SUMS
C      C = SS(XI,J) = SUM(XI,XJ)-SUM(XI)*SUM(XJ)/N
C
C 180 C SIMPLE CORRELATION COEFFICIENTS =
C      C = R(XI,J) = SS(XI,J)/SQRT(SS(XI,I)*SS(XJ,J))
C
C***** RESIDUAL ANALYSIS + ACTUAL VS PREDICTED + PRINTOUT *****
C
C ACTUAL = Y = DEPENDENT VARIABLE
C PREDICTED = YC = COMPUTED Y USING REGRESSION EQUATION
C
C 185 C RESIDUAL = E = YC - Y
C      C NORMALIZED DEVIATE = RESIDUAL / STANDARD ERROR
C      C PERCENT DEVIATION = 100 * RESIDUAL / ACTUAL
C      C WEIGHT = INPUT WEIGHT OF OBSERVATION
C      C SSE = RESIDUAL SUM OF SQUARES
C      C CHI SQUARE = SUM(E RESIDUAL**2) / WC
C
C***** ANALYSIS PRINTOUT *****
C
C 190 C TOTAL(CORRECTED) SUM OF SQUARES = SUM OF SQUARES ABOUT THE MEAN
C      C = SUM((Y-YC-MEAN)**2) = SUM((YC-MEAN)**2)+SUM((Y-YC)**2) = SSE(Y)
C      C TOTAL(ORIGIN) SUM OF SQUARES = SUM OF SQUARES ABOUT THE ORIGIN
C      C USED INSTEAD OF SSE(Y) WHEN REGRESSION IS FORCED THRU ORIGIN.
C      C REGRESSION SUM OF SQUARES = SUM OF SQUARES DUE TO REGRESSION
C      C = EXPLAINED VARIATION = SUM((YC-MEAN)**2) = SSE(Y)
C      C PREDICTED SUM OF SQUARES = SUM OF SQUARES ABOUT THE REGRESSION
C      C = UNEXPLAINED VARIATION = SUM((Y-YC)**2) = SSE(Y)
C      C THE MEAN SQUARES COLUMN IS OBTAINED BY DIVIDING THE SUM OF SQUARES
C      C ENTRY BY ITS CORRESPONDING DEGREES OF FREEDOM.
C      C RESIDUAL MEAN SQUARE = VARIANCE ABOUT THE REGRESSION = SSE/(N-1)
C      C COEFFICIENT OF MULTIPLE DETERMINATION = FACT OF EXPLAINED VARIATION
C      C = (SS DUE TO REGRESSION)/SS(About MEAN) = CORR. FCRP. S*(R)
C      C CORRELATION COEFFICIENT = R = SQRT(CORR. FORM OF SSE(R))
C      C S.E. AS PCT. OF MEAN = 100 * S / YMEAN
C      C F TEST FOR SIGNIFICANCE OF REGRESSION PREDICTED(SSE)
C
C 195 C CONSTANT = A(0) = YMEAN-SUM(XI)*XMEAN() = CONSTANT TERM
C      C COEFFICIENT = A(I) = THE EFFECT ON Y OF A UNIT INCREASE IN XI IF FN
C      C THE OTHER VARIABLES ARE HELD CONSTANT.
C      C STANDARD ERROR = STANDARD ERROR OF REGRESSION COEFFICIENT.
C      C THE 95 PERCENT CONFIDENCE LIMITS FOR A UNIVARIATE REGRESSION
C      C COEFFICIENT ARE GIVEN BY THE SAMPLE COEFFICIENT PLUS AND MINUS
C      C T(10.025) TIMES THE ESTIMATED STANDARD OF THE COEFFICIENT.
C      C COFF/SE = USED IN HYPOTHESIS THAT COEFFICIENT = 0.
C      C COEFFICIENT DIVIDED BY ITS STANDARD ERROR TO GIVE THE NUMBER
C      C OF S.E. AWAY FROM HYPOTHEZED ZERO. SHOULD BE GREATER THAN 1
C      C VALUE TO REJECT THE HYPOTHESIS THAT THE COEFFICIENT IS NOT
C      C SIGNIFICANTLY DIFFERENT FROM ZERO.
C
C 200 C F = VALUE TO REMOVE VARIABLE FROM REGRESSION. *** NOT USED ***
C
C 205 C BETA COEFFICIENT = MEASURE OF THE NET EFFECT OF EACH VARIABLE ON Y.
C
C 210 C RSC CHANGE = DECREASE IN RSO IF THE VARIABLE IS REMOVED FROM REGRESSION

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7474 OPT=1  
 FTN 4.2+74278 11/22/74 10.1e.21.  
 C\*  
 C PARTIAL RSD = THE SQUARE OF THE PARTIAL CORRELATION COEFFICIENT OF  
 C VARIABLE K NOT IN THE REGRESSION WITH THE RESPONSE Y.  
 C = R(XYLN...)\*2 WHERE L,M,N,... ARE ALREADY IN REGRESSION.  
 C = RELATIVE AMOUNT OF IMPROVEMENT THAT IS BROUGHT ABOUT IF  
 C VARIABLE K WERE ADDED TO THE REGRESSION.  
 C NORMED SUMPSD = THE NORMALIZED SUM OF SQUARES OF RESIDUALS FOR  
 C VARIABLE K HAD IT TOO BEEN INCLUDED. USEFUL IN DRAWING  
 C ATTENTION TO NEAR-LINEAR DEPENDENCIES AMONG THE IND. VARIABLES  
 C DFLTA PSD = CHANGE IN RSD IF VARIABLE K WERE ADDED TO REGRESSION NEXT.  
 C VARIABLE WITH LARGEST DELTA IS ADDED TO REGRESSION NEXT.  
 C F = F VALUE TO ADD VARIABLE TO REGRESSION. \*\*\* NOT USED \*\*\*  
 C\*\*\*\*\* ADDING AND DELETING VARIABLES \*\*\*\*\*  
 C  
 C STEP 1 - THE VARIABLE NOT IN THE EQUATION WHICH CAUSES THE GREATEST  
 C CHANGE IN PSD IS ADDED TO THE REGRESSION.  
 C STEP 2 - THE VARIABLES IN THE EQUATION ARE THEN CHECKED TO SEE IF ONE  
 C CAN BE DELETED. THE VARIABLE WHICH CAUSES THE SMALLEST CHANGE IN  
 C PSD IS SELECTED FOR REMOVAL. IF THE EQUATION WITHOUT THIS VARIABLE  
 C PRODUCES A SSIR WHICH IS SMALLER THAN THE PREVIOUS SSIRD FOR THAT  
 C NUMBER OF VARIABLES, THE VARIABLE IS REMOVED.  
 C STEP 3 - IF A VARIABLE WAS REMOVED, REPEAT STEP 2.  
 C OTHERWISE REPEAT STEP 1 AND 2.  
 C\*\*\*\*\*  
 C  
 C IT SHOULD BE NOTED THAT THE STATISTICS FOR NON-LINEAR EQUATIONS  
 C SHOULD BE USED WITH CARE, AND SHOULD NOT BE COMPARED WITH THOSE  
 C FROM LINEAR EQUATIONS, AS THEY MAY HAVE DIFFERENT MEANINGS.  
 C FOR EXAMPLE - IF Y IS TRANSFORMED BY TAKING ITS LOGARITHM, THE  
 C SUM OF THE SQUARES OF THE ACTUAL PREDICTED VALUES BETWEEN THE CALCULATED  
 C AND THE OBSERVED Y VALUES ARE NOT MINIMIZED. FATHER THE SUM OF  
 C SQUARES OF THE LOGARITHMS OF THE RATIOS OF THESE VALUES ARE  
 C BEING MINIMIZED ( $(\log y_i - \log \hat{y}_i)^2 / \log(y_i)$ ).  
 C THEREFORE, COMPARISON OF ANY STATISTICS THAT ARE BASED ON THE  
 C SUM OF THE SQUARES OF THE Y RESIDUALS SUCH AS THE F VALUE OR  
 C CORRELATION COEFFICIENT MAY BE MISLEADING.  
 C  
 C IT SHOULD ALSO BE NOTED THAT WHEN THE CURVE IS FORCED THROUGH THE  
 C ORIGIN OR SOME OTHER SPECIFIED Y INTERCEPT, THE DEGREES OF FREEDOM  
 C ARE CHANGED AND THE CURVE NO LONGER GOES THROUGH THE MEANS OF THE  
 C VARIABLES. THEREFORE, CHANGING THE VALUES OF THE STATISTICS ARE  
 C MAKING COMPARISONS OF CURVES WITH UNSPECIFIED Y INTERCEPTS  
 C MISLEADING. ALSO, COMPARISON OF F VALUES WITH THE STANDARD F  
 C DISTRIBUTION IS NOT NECESSARILY VALID.  
 C\*\*\*\*\*  
 C  
 C MF USERS OF THIS PROGRAM ARE URGED TO REVIEW THE STANDARD TEXTS  
 C ON REGRESSION ANALYSIS FOR THE USES AND LIMITATIONS OF THIS  
 C TECHNIQUE, AND REAR IN MIND THAT THE STATISTICAL RELATIONSHIPS ARE  
 C NO BETTER THAN THE DATA THAT WAS USED TO COMPUTE THEM.  
 C\*\*\*\*\*  
 C  
 230  
 235  
 240  
 245  
 250  
 255  
 260  
 265  
 270  
 275  
 280  
 285



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PROGRAM MLP      74/74   OPT=1          FTN 4.2+74276   11/22/74  16:16:21.

      9 4H  1.9.1.328.1.729.2.539. 4H  20.1.322.1.725.2.520.    PLR 292E
      1 4H  21.0.1.721.2.518. 4H  22.0.1.32.1.73.2.508.    PLR 293E
      2 4H  23.0.1.215.1.714.2.510. 4H  24.0.1.318.1.711.2.492.    PLR 294E
      3 4H  25.1.316.1.708.2.465. 4H  26.1.311.1.706.2.475.    PLR 295E
      4 4H  27.0.1.31.1.703.2.473. 4H  28.0.1.31.1.703.2.461.    PLR 296E
      5 4H  29.1.311.1.699.2.462. 4H  30.1.310.1.691.2.477.    PLR 297E
      6 4H  43.0.1.303.1.694.2.423. 4H  50.1.298.1.676.2.433.    PLR 298E
      7 4H  60.0.1.296.1.671.2.390. 4H  80.1.292.1.664.2.372.    PLR 299E
      8 4H  500.0.1.290.1.660.2.365. 4H  200.1.287.1.655.2.345.    PLR 300E
      9 4H  500.0.1.233.1.649.2.334. 4H  INF.1.282.1.645.2.325.    PLR 301E
      WRITE(6,1000) (ADATA(j), j=1,22), (YDATA(j), j=18A)

      C
      TSTART = SECOND(TSTART)
      PSTART = TSTART
      PLR 3640
      CALL SLITF (0)
      PLR 3650
      CO 20 J=1,30
      PLR 3660
      00 20 K=1,30
      PLR 3770
      20 IMOPAC(j,k)=0
      PLR 3780
      JF = 0
      PLR 3790
      JF = 0
      PLR 3800
      READ (5,950) NPROB,NXV,NYV,INDEXY,NOAT,JOE,h,INTYPE,AREAR,PASSTF,IMLR
      1,IFRACH,INSTART,MINSUM,MAXREG,IFWT,IFCRNS,IFLIST,IFSUNS,IFAVPLR
      2,IFCOPR,INCROSS,IFTRA,ASKIP,IFSUR,IFMT,IFPACH,IFDATE,IFNAME
      IF 1 EOF(5) .NE. 0.0 , GO TO 805
      PLR 3810
      WRITE (6,810)
      PLR 3820
      WRITE (6,1010)
      PLR 3830
      1,IFRACH,INSTART,MINSUM,MAXREG,IFWT,IFCRNS,IFLIST,IFSUNS,IFAVPLR
      2,IFCOPR,INCROSS,IFTRA,ASKIP,IFSUR,IFMT,IFPACH,IFDATE,IFNAME
      DIS=BLANK
      PLR 3840
      IF ((IFLATE-EF.0) CALL DATE (01))
      PLR 3850
      NPNON=NSTART
      PLR 3860
      IF ((NF=1) .EQ. 0 ) F4T(1) = R4T
      PLR 3870
      IF ((NOAT.LE.0) NOATA=10000
      PLR 3880
      NHTT=NXY+NYY
      PLR 3890
      IF ((IFW,1,NE,0) NHTT=NNHT+1
      PLR 3900
      IF ((IFSUB,GT,0) CALL NAROFX (IFSUB,NXV)
      PLR 3910
      NSTEP=0
      PLR 3920
      NSUMRY=0
      PLR 3930
      TOL=0.00001
      PLR 3940
      NRRXY=XV*1
      PLR 3950
      NBEX = NUMBER OF INDEPENDENT VARIABLES
      PLR 3960
      NBKAY = NUMBER OF INDEPENDENT VARIABLES + DEPENDENT VARIABLE
      PLR 3970
      NBRYM = SIZE OF ARRAY = NBRYX + 1
      PLR 3980
      NBROM = NUMBER OF COEFFICIENTS FOR PRESENT EQUATION
      PLR 3990
      INDEX = INDEX OF PRESENT EQUATION
      PLR 3995
      IF ((PASSTF,ED,0) MAXSTP=99
      PLR 3996
      IF ((MAXREG,LT,5) MAXREG=5
      PLR 3997
      TO STORE DATA ON TAPE FOR USE IN ANOTHER FROLEM. SET INTYPE=1
      PLR 3998
      FOR REWIND. INTYPE=3 FOR NO REWIND
      PLR 3999
      TO USE DATA FROM A PREVIOUS PROBLEM SET INTYPE = 2 IF .
      THEREBY CAUSING TAPE TO REWIND.
      PLR 3999
      INTYPE = 4 OR 6 FOR BIN. TAPE 10 OR 11 TO BE USED AS INPUT
      PLR 3999
      ALSO PREVENTS TAPE REWIND AT START OF PRCLEP.
      PLR 3999

```

```

PROGRAM MLR      74/74   OPT=1          FTN 4.0-74278      11/22/74  10.10.21.

400      NREADS=1
        IF (INTYPE .NE. 7) GO TO 50
        NFMHS=0
        INTYPE=5
        NTAPI=10
        NREAD=0
        NWRITE=0
        IF (INTYPE .EQ. 0) GO TO 60
        IF (INTYPE .EQ. 1.0R. INTYPE .EQ. 3) NWRITE=1
        IF (INTYPE .EQ. 5.0R. INTYPE .EQ. 6) NTAPE=11
        IF (INTYPE .EQ. 1.0R. INTYPE .EQ. 2.0D. INTYPE .EQ. 5) REWIND TAPE
        IF (INTYPE .NE. 1.AND.INTYPE .NE. 3) NREAD=1

C      NUMBER OF INDEPENDENT + DEPENDENT VARIABLES
C      60 NTOTAL=NKV.NVY
        NF=NKV+NQEXY
        IF (NTOTAL .LE. 52) GO TO 70
        C      NTOTAL=NEW
        TOC MANY VARIABLES
        NTOTAL=NEW
        IF (INITIAL .LE. 52) GO TO 70
        WRITE (16,820) NTOTAL
        CALL EXIT
        70 NSTEP=6
        C      CHECK FOR CROSS-PRODUCTS OR POLYNOMIAL
        IF (NCROSS .EQ. 0) GO TO 80
        IF1 NCROSS .GT. 1 J3 10 75
        WRITE(16,1110)
        NOVAR=(NBRXY*(NBRXY+1))/2
        IF (NOVAR .LE. 51) GO TO 90
        NCROSS=0
        WRITE (16,050)
        GO TO 80
        75 IF ( NCROSS .GT. 50 ) NCROSS = 50
        WRITE(16,1120)
        IF1 NVY .EQ. 1 ) GO TO 76
        NCROSS = 0
        WRITE(16,1130)
        76 NCROSS = NCROSS + 1

C      80 NOVAR=NBRXY
        90 IF ((FCNST.LT.0) NOVAR=NOVAR+1
        NSRXY=NOVAR+1
        NBRX=NOVAR-1
        MAXVAR=MIN((MAXSUM,MAXREG,NBRX))
        REAO CONTROL CARD C
        C      IF ( IDEN .LT. 95 . LT. 94,110
        94 DO 100 J=1,16
        100 ALPHA(J)=BLANK
        95 IOEN = IABSTIDEN
        GO TO 130
        110 00 120 1=1,10EN
        READ (5*03C) (ALPHA(J),J=1,16)
        120 WRITE (16,840) (ALPHA(J),J=1,16)
        130 CONTINUE
        C      REAO CCTRL CARC 0

```

```

PROGRAM PLR      74/74   OPT=1           FTN 4.0.2+74278      11/22/74  12.10.21.

00 140 J=1,60
KSTEP(J)=1
140 TEST(J)=+2.0E+30
IF (NSTART, 170, 200, 150
150 READ (5,530) (TEST(J),INDEX(J),J=1,NBRACH)
C     PACK INDEX
00 160 J=1,NBRNM
160 CALL PACK (NBRNM,J,INDEX(J),1)
GO TO 190
CC170 NBRNM=NBRX
170 NBRNM=N:V
DO 190 J=1,NBRNM
CALL PACK (NBRNM,J,J,1)
INDEX(X(J)) = J
TEST(J) = 1.000
180 IF (NSTART,LI,-1) TEST(J)=0*0
190 WRITE (6,90) START,TEST(J),INDEX(J),J=1,NBRNM)
200 MINVAR=MAX(0,MINSUM)
NAP=9=1
IF (BACK,EC,0) GO TO 210
IF (DATA=NBRXYM,LE,30000) GO TO 210
NTAP=9=0
REWIND 9
210 00 220 I=1,NBRXYM
00 220 J=1,NBRXYM
220 AII,J=0
C     REAO CONTROL CARD E
IF (NSKIP,LE,0) GO TO 270
READ (5,950) (JV010IJ,J=1,14)
JV=0
DO 250 L=1,28,2
NOGOD0(LL)=0
NOGOD0(LL+1)=0
230 IF (JV,EQ,14) CO TO 260
JV=JV+1
IF (JV,CID(JV)) 250,230,240
?40 NOGOD0(LL)=JV010IJ
NOGOD0(LL+1)=JV010IJ
L840=L+
IF (NSKIP,NE,0) 4RITE (6,960) VOTEO,(NCGCCD(J),J=1,L840)
250 NOGOD0(LL+1)=IABS(JV010IJ)
260 CONTINUE
270 NSKIP=IABS(NSKIP)
IF (NSKIP,NE,0) 4RITE (6,960) VOTEO,(NCGCCD(J),J=1,L840)
C     READ CONTROL CARD F
LCKED=0
IF (INREAR) 290,300,240
IF (NREAR=1) READ SET OF SEARCH PARAMETERS.
C     280 MNXJ=MNV
IF (IFWT,NE,0) MNXJ=MNV+1
READ (5,950) NMROS,LOCX(J),J=1,MNXJ
290 WRITE (6,960) SEARCH,NMROS,LOCX(J),J=1,MNXJ
LOOK=1
300 CONTINUE
C     READ CONTROL CARD G
IF (IFTRA,GT,0) READ (5,930) (CONST(I),REFRA(I),I=1,NTOTAL)
PLR 3940
PLR 3950
PLR 3960
PLR 3970
PLR 3980
PLR 3990
PLR 4000
PLR 4010
PLR 4020
PLR 4030
PLR 4040
PLR 4050
PLR 4060
PLR 4070
PLR 4080
PLR 4090
PLR 4100
PLR 4110
PLR 4120
PLR 4130
PLR 4140
PLR 4150
PLR 4160
PLR 4170
PLR 4180
PLR 4190
PLR 4200
PLR 4210
PLR 4220
PLR 4230
PLR 4240
PLR 4250
PLR 4260
PLR 4270
PLR 4280
PLR 4290
PLR 4300
PLR 4310
PLR 4320
PLR 4330
PLR 4340
PLR 4350
PLR 4360
PLR 4370
PLR 4380
PLR 4390
PLR 4400
PLR 4410
PLR 4420
PLR 4430
PLR 4440
PLR 4450
PLR 4460
PLR 4470
PLR 4480
PLR 4490

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```

PROGRAM MLR      74/74   OPT=1          FTN 6.2+74270    11/22/74  10-18-21.

420 IF (POINT(1) .EQ. 99999999.1) GO TO 530
  IF (IFSUB .LT. 0) CALL EQUAT (IFSUB,POINT)
  IF (IFLIS .EQ. 0) WRITE (6,310) N, (POINT(j),j=1,NWT)
  THREW AWAY BLANK DATA (BLANK = -0.)
C   IFALK = 3

IF (IFBLK .NE. 0) GO TO 428
ON 425 J=1,NXV
  IF (POINT(j) .NE. 0.) GO TO 425
  POINT(j) = POINT(j)
  SIGN(j) = SIGN(j)
425 CONTINUE
  IF (POINT(NEW) .NE. 0.) GO TO 426
  IF (SIGN(NEW) .NE. 0.) POINT(NEW) = 427+426+428
  427 WRITE(6,1105) N
GO TO 529

C   428 JDATA = JDATA + 1
  IF (IFTRA .EQ. 0) GO TO 430
  CALL CHANGE (POINT,NRTPA,CONST,NTOTAL)
  IF (IFLIST .EQ. 0) WRITE (6,923) N, (POINT(j),j=1,NWT)
C   430 CONTINUE
  PCINT(POVARI)=POINT(ITEM)
  MHT=1.0
  IF (IFWT .NE. 0) MHT=POINT(1)
  POINT(NBRYW)=MHT
  IF (INCROSS .EQ. 0) GO TO 450
  IF (INCROSS .GT. 1) GO TO 445
  C   CROSS PRODUCTS ARE USED AS INDEPENDENT VARIABLES
  L=NBRXY
  00 440 I=2,NBRXY
  CO 440 J=1,NBRXY
  POINT(I)=PCINT(I-1)*PCINT(J-1)
440 L=L+1
  GC TO 450
C   GENERATE POWERS FOR POLYNOMIAL
  445 NO 446 J=2,NCROSS
  446 POINT(j) = POINT(j-1) * POINT(1)
  451 IF (IFCMST .LT. 0) POINT(INDVAR-1)=1.0
  IF (IFBACK .LT. 0) GO TO 470
  IF (ITAPE9 .EQ. 0) GO TO 470
  C   STORE IN STRING IF DATA POINTS * VARIABLES LESS THAN 3000
  ON 470 J=NBRXY
  JJ=NBRXY+N-1+j
  460 STPING(JJJ)=POINT(j)
  GO TO 489
C   STORE DATA ON TAPE 9 IF DATA POINTS * VARIABLES EXCEED 3000
  471 WRITE (9) (POINT(k),k=1,NOVAR),MHT
C   480 CONTINUE
  IF (INSKIP .EQ. 0) GO TO 500
  C   CHFCK TO SEE IF POINT IS TO BE DELETED REGRESEICA
  00 490 J=1,LBACK2
  IF (IN.LT.400000(j)).OR. N.GT. MC000(j+1)) GC TO 490
  NBO=LBACK1
  GO TO 520
C   490 CONTINUE

```

PROGRAM NLR      74/74      OPT=1      FTN 4.2+ /4278      11/22/74 10.10.21.

```

C 500 00 S10 I=1,NOVAR
C 500 00 SUM X(I)
C A(I,NBRXYN)=A(I,NBRXYN)+POINT(I)*NNHT
CO S10 J=1,NOVAR
C S10 SUM X(I)*X(J)
C 510 A(I,J)=A(I,J)+POINT(I)*POINT(J)*NNHT
A(NBRAYN,NBRXYN)=A(NBRAYN,NBRXYN)+NNHT
S20 CONTINUE
C *****
C 530 NOATA=JODATA-NBAD
DEF R=NOATA
DE NOATA=A(NBRAYN,NBRXYN)-1.0
IF (IIFWT*NE.0) DENOM=DE*404+1.0
IF (INAPES*EQ.0) RENIN=9
IF (INFEN5*EQ.0) RENIN=11
C
WRITE (6,1000) NPROP,ALPHA,NDATA,NOVAR,A(NBRXYN,NBRXYN),D1
K=2
IF (IIFTRA-EQ.0) K=1
WRITE (6,860) (BLANK,J=1,K)
WRITE (6,870)
IX1=60
IX2=0
IXP = 1
C
C 00 590 J=1,NOVAR
V(NBRYN)=A(J,NBRXYN)/A(NBRAYN,NBRXYN)
STORE(SORT((A(J,J))-A(J,NBRXYN))*Y(NEAN))/DENOM)
TRA(1)=BLANK
TRA(2)=BLANK
V(1,J)=VNAME(1,J)
V(2,J) = VNAME(2,J)
L=J
K=2
IF (IIFGT,NVY) GO TO 560
IF (IIFTRA-EQ.0) GO TO 580
I=NBRIPAI()
IF (FILE.0) GO TO 580
K=3
TRA(3)=CONST(L)
550 TRA(1)=ATRA(1,I)
TRA(2)=ATRA(2,I)
GO TO 560
560 I=17
V(1,J) = 6WCONST.
V(2,J) = 5W TERM
IF (IFCNSLT,L,0,AND,J-EQ.NOVAR-1) GO TO 550
L=NWVINDEXX
V(1,J) = VNAME(1,L)
V(2,J) = VNAME(2,L)
TR(4,J-EQ.NCVAR) GO TO 570
IF (ACROSS .GE. 2) GO TO 575
C CROSS PRODUCTS
IX1=IX1+1
675
680

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PROGRAM PLR      76/74   OPT=1           FTN 4.2+74270    11/22/74  18:10:21.

685      IF (IX1.LT.NAV) GO TO 570
         IX2=IX2+1
         IX1=IX2
570      WRITE(6,800) J,YMEAN,SDEV,IX2,IX1
         V11,J1 = CROSS(IX1)
         V12,J1 = CROSS(IX2)
         GO TO 590
C      POLYNOMIALS
C      575 V11,J1 = V11,J1
         EXP = EXP + 1
         ENCODE(6,1140,V12,J1) EXP
580      IF(J.EC,NOVAR,GJ TO 590
         WRITE(6,900) V11,J1,V12,J1,J, YMEAN, SDEV, (TRAIL),J=1,8)
         590 CONTINUE
         J = ACVAR
         MP ITRE(6,900) V11,J1,V12,J1, YMFAN, STOEV, (TRAIL),J=1,K)
         IF (LIFSUMS.EQ.0) CALL PRSUM
         IF (IFCAST.NE.0) GO TO 600
         CALL RESID
         DEFNM=DEFRM-1,0
600      CALL CCPFL
         CALL SLITET(1,LIGHT)
         IF (LIGHT.EQ.1) GO TO 10
         NDATA=JOATA
         NSXP=NBAO
         IF (MAXSXP.EQ.999) WRITE(6,1060)
         *****
C      605 LPATH = 1
         KPATH = 1
         IF (LABFROM.GT.0) GO TO 710
610      IF (AINOVAR,NOVAR.GT.0) GO TO 620
         WRITE(6,1030) AINOVAR,NOVAR)
         GO TO 750
C      615 JP = 0
         NRPV = NERNDH
         NRNDH = NRNDH - 2
         DO 617 J=1,NRPV
         CALL PACK1(NRPV, J, 0, 1)
         IF (INDEX(J) .EQ. KVAR) GC TO 617
         JP = JP + 1
         CALL PACK1(NRNDH, JP, INDEX(J), 1)
         617 CONTINUE
         DO 618 I=1,NOVAR
         DO 619 J=1,NOVAR
         619 AII,J1 = SIMCOR(I,J)
         GO TO 605
C      620 CALL AC010
         CALL SLITET(1,LIGHT)
         GO TO 1750,630, LIGHT
630      GO TO (640,720,690) LPATH
C      640 TEST(INPNOW)=1(NOVAR,NOVAR)
         650 CALL OUTPUT

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PROGRAM MLR 74/74 OPT=1 FTN 4.2+74270 11/22/74 10.10.21.

```

CALL SLITET(12,LIGHT) PLR 6630
IF(LIGHT.EC.1) GO TO 615
NSTEP=STEP1 MLR 6631
IF(INSTEPI.GT.MAXSTP) GO TO 750
IF(NBRN0W.GE.MAREG) GO TO 750
CALL SLITET(11,LIGHT) PLR 6632
GO TO (750,660), LIGHT PLR 6633
GO TO (680,670,10), KPATH PLR 6700
673 KPATH=1 PLR 6701
C SEF IF VARIABLE CAN BE REMOVED
C 680 IFI(NBRN0W.EQ.2)ANO. JF(ANE.D) GO TO 700
CALL RMVCVE MLR 6730
IFI(MODEL.EQ.0) GO TO 700
C VARIABLE WAS REMOVED
LPATH=1 MLR 6731
690 CALL MATRIX PLR 6732
GO TO (650,740,700), LPATH PLR 6733
700 IF(NBRN0W.LT.NBRTX) GO TO 610 PLR 6734
    WRITE(6,1020) PLR 6735
    GO TO 750 PLR 6736
710 KPATH=2 PLR 6737
720 L=1 PLR 6738
    LPATH=2 PLR 6739
730 CONTINUE PLR 6800
    CALL PACK(NBRN0W,L,KVAR,2) PLR 6801
    GO TO 690 PLR 6802
C ***** TRY ADJUNCTION *****
740 L=L+1 PLR 6803
    IF(L.LE.NPRN0W) GO TO 730 PLR 6804
    GO TO (700,640,740), KPATH PLR 6805
C SUMMARY PLR 6806
C 750 WRITE(6,1040) ALPHA,NPROB PLR 6807
    IF(IFBACK.EC.991) IFBACK=INSTEP PLR 6808
    IF(IFBACK.EQ.999) IFBACK=1 PLR 6809
    IF(PAXSTP.EC.999) MAXSTP=999 PLR 6810
    00 757 J=1,0BRN0D PLR 6811
    DO 755 L=1,J PLR 6812
    755 CALL PACK(J,L,INDEX(L),2) PLR 6813
    IF(INDEX(1).LE.0) GO TO 757 PLR 6814
    WRITE(6,1070) J,STEP(J),STERR(J),CORSCR(J),INDEX(L),L=1,J)
    757 CONTINUE PLR 6815
    KPATH=3 PLR 6816
    NBRN0W=MIVAR PLR 6817
    NSUQRY=1 PLR 6818
    760 IF(NBRN0W.GT.MAXVAR) GO TO 795 PLR 6819
    CALL PACK(NBRN0W,1,J,2) PLR 6820
    IF(J.LE.0) GO TO 793 PLR 6821
    DO 770 I=1,NOVAR PLR 6822
    DO 770 J=1,NOVAR PLR 6823
    770 AI,I,J=SIMCOR(I,J) PLR 6824
    GO TO 720 PLR 6825
    780 CALL OUTPUT PLR 6826
    791 NBRN0W=NPRN0M+1 PLR 6827
    GO TO 760 PLR 6828
C 795 PEND = SECONDL PEND , PLR 6829

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PROGRAM MLR	74/74	OPT=1	FTN 4.2+74278	11/22/74 18+10.21.
110J FORMAT(1H1,F10.3/20/1H-+602H*) 110S FORMAT(6H POINT 14.26H DELETED DUE TO BLANK DATA ) 1110 FORMAT(25HCROSS PRODUCTS GENERATED ) 1120 FORMAT(14.16HPOWERS GENERATED FOR POLYNOMIAL OF ORDER 13) 1130 FORMAT(53HONLY ONE INDEPENDENT VARIABLE ALLOWED FOR POLYACPIAL) C 1140 FORMAT(4H ** *12) ENC			PLR 7730	

860

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SUBROUTINE OUTPUT      7474   OPT=1          FTN 4.2+74278      11/22/74  10.30.37.

      SUBROUTINE OUTPUT
      COMMON SIGMA(60),A(52,52),SIMCOR(52,52),AVG(60),TEST(60)
      COMMON PCINT(60),STRING(3000),INDPAC(30,30),INDEXP(60)
      COMMON INDEX(60),NOUT(60),MSTEP(60),ALPHAB(16),YMEAN,IDEN,IFAVE
      COMMON MASTP,IFPNC,NINSTRY,MSP,NAPE,NEW
      COMMON NINCR,NINROM,NOSTEP,INDATA,NINRY,NINRX,LPATH,DEFRM,KVAR
      COMMON IFEACK,IFCNSI,IFCNR,NPROM,NRPVR,NDDEL,JF
      COMMON INDEXY,LBAD,NOGOOD(128)
      COMMON IFR,YCONS,INTRAV(12,51),YTRA(21)
      COMMON STORER(50),CORSOR(50)
      REAL      A,SINCOR,SIGMA,AVG,TFST
      C
      C      DIMENSION CDEFF(61),AAC(5)
      C      CDEFF,CONST
      REAL      CUFF,CONST
      REAL      SURSQ,TSS,SIGN2,SIGY
      REAL      YPREO,YOBS,DEV,RSQREG,SDREG2
      REAL      DEVSQ,CHISQ,SUMSQU,CMSEQ,DEVSQ,VD,VG
      DATA BLANK/1M /*VOID/6HVOIDED/6HREVIEW/
      DATA ACTUAL/6HACTUAL/
      NPNON = NUMBER OF COEFFICIENTS FOR PRESENT EQUATION
      INDEX = INDEX OF PRESENT EQUATION
      NSUMRY = 0 FOR BUILDING PHASE. = 1 FOR SUMMARY PHASE.
      IF(NRNDNAEQ.0.) JF = 1
      KPATH=1
      IF (INSKIP.NE.0) KPATH=2
      NMT=10
      TSS = SIGMA(INOVAR)*SIGMA(INOVAR)
      CALL SLITET(1,LIGHT)
      GC TO (10,20), LIGHT
      10 CALL SLITE(1)
      GO TO 30
      20 NOSTEP=NSTEP+1
      IF (INSLMTY.EC.1) NOSTEP=KSTEP+INPNON
      30 00 40 J=1,60
      40 NOUT(J)=0
      ON 50 J=1,NROMN
      CALL PACK (NBRROW,J,1,2)
      INDEX(IJ)=I
      NCUT(IJ)=1
      C      50 CNEFF(IJ)=A(IJ,NOVAR)*SIGMA(INOVAR)/SIGMA(IJ)
      IF (IFCNSI.EQ.0) GO TO 60
      CONST=0
      CONST=0
      GO TO 40
      60 CONST=AVG(INOVAR)
      00 70 I=1,NROMN
      INDEX(IJ)
      70 CONST=CONST-(COEFF(IJ)*AVG(IJ))
      61 SUMSD = (INOVAR,NOVAR) * TSS
      XVARINROMN
      DEF=DEFRM-KVAR
      NODEF=CEFR
      NODEFRM=DEFRM
      C.....*
      55 IF (A(INOVAR,NOVAR).LT.0.0) A(INOVAR,NOVAR)=0.000
      CC
      CC      SUHS3=CAES(SUHS2)
      AUTP1460
      AUTP1476

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SUBROUTINE OUTPUT    74/74   OP1+1

60      SUMSQ= ABS(SUMSQ)
        IF (NOTF,G1,0) J0 TO 240
        WRITE (6,370) MOSTFP
        SIGY=0.0
        S1PCT=0.0
        REFCO=0.0
        CALL SL1FE (1)
        S1DER (INBRNDM) = 0.0
        COSBR (INBRNDM) = 0.0
        IF (MAXSTP.EQ.999) GO TO 999
        GC TO 260
        CONTINUE
240      SIGY2=SUMSQ/DEFR
        CC      SIGY = DSORT(SIGY2)
        SIGY = SCRT ( SIGY2 )
        RSI=1.00D-1*INVAR*NOVARI
        IF (I NCSTEP .NE. KSTEP (INBRNDM)) GO TO 245
        S1DER (INBRNDM) = SIGY
        COSBR (INBRNDM) = RSQ
245      CONTINUE
        IF (MAXSTP.EQ.999) GO TO 999
        AY=AINVAR*NOVARI,
        TR=1.0
        PEGRCO=SORT (RSQ)
        R2=RSG/AY*AY
        V2A=INVAR*NOVARI/DEFR
        FTEST=P2/Y2
        S0REG=FS*Y2SS
        S0REG2=S0REG*AY*AY
        DATA CORREC, TED/6HCORREC,3INTED/,ORI,GIN/6H ORIGI,1NN/
        BASE1=COREC
        BASE2=TED
        BASE1=ORI
        BASE2=GYH
250      WRITE (6,440) MOSTEP,ALPHA,MPCB,BASE1,EASE2,NCEFPN,155,1P,4ERRACH,
        1S0REG,S0REG2*RSQ*2*FTEST,DEFR,SUMSO,SIGY2,AY,V2
        SIGPCT=AES(SIGY/YMEAN*100.0)
C      260      WRITE (6,340) SIGY,V11,NOVARI,V12,NOVARI,SFACT,TEGRCC
        DO 290 J1=1,NBRX
        CC      IF (I IFPCH .NE. 0 .AND. NSUMRY .EQ. 0) WRITE (7,400) KK,CCAST,AFACE,APPALT
        CCP71 POINT(J1,0).99922
        CC      GO TO 293
        CC      POINT(J1)=1(J,NOVARI*A(NOVARI,J)/A(J,J))
299      CONTINUE
C      299      WRITE (6,440)
        K1=0
        IF (I IFPCH .NE. 0 .AND. NSUMRY .EQ. 0) WRITE (7,400) KK,CCAST,AFACE,APPALT
        10M,INDEXY,NOSTEP,IIFPCH,SIGPCT,REGPC
        IF (IFCNS1.EQ.0) WRITE (6,390) CONST
C      *** LIST COEFFICIENTS
        DO 310 J1=1,NBRNOM
        I=INCE(XIJ)
        DO 310 J1=1,NBRNOM
        IF (I INCE(XIJ)
        100      C
        110      C

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SUBROUTINE OUTPUT 74/74 OPT=1          FTN 4.2+7278   11-16-79
115      WRITE(17,400) I,COEFF(J),NPROM,NBROB,INOREX,MSTEP,IIFPC1,LIP230
        1 REGRO
        300 SEB=SIGN(ABS(A(I,I)))*SIGY/SIGMA(I)
        CT=COEFF(IJ)/SEB
        IF( SEB .EQ. 0.0 ) CT = 0.0
        F = CT *
        MP1=I,411,V1,I,Y(2,I),I,COEFF(J)+SEB,C1,F,A(I,NOVAR),POINT(I)
        IF(I POINT(I)).EQ.0.0 ) GO TO 310
        WRITE(16,50)
        CALL SLITE(2)
        KVAR = 1
        310 CONTINUE
C       IF (NBROW>EO-NBRX) GO TO 360
        IF (NOEFLR>0) GO TO 360
        WRITE(16,420)
        NP=0
        CT = NOEFLR - 1
        00 350 IM=1,NBRX
        IF( IM>POINT(I) ) 350, 315, 350
        IF( CT + POINT(I) ) / ( AVY - POINT(I) )
        PAP(POINT(I))/AVY
        IF( A(I,1).LE.TOL ) POINT(I) = 3.333333E33
        320 IF( NP) 340,330,340
        330 IM=1
        FH=F
        RPAR=RAR
        SSN=(I/I)
        DELTPC(I)
        NP=1
        340 WRITE(16,430) IM,RPAR,SSN,DELT,FH, I,PAR,A(I,I),POINT(I),F
        350 CONTINUE
        IF( NP.NE.0 ) WRITE(16,430) IM,RPAR,SSN,CEL,I,FH
        360 CCNTINUE
C.....*
C     *** COMPUTE BACK SOLUTION
C     *F (MAXSTP,EO,999) GO TO 999
        IF (IFBACK.EQ.0.OR.IFBACK.GT.MSTEP) GO TO 999
C
        NABC = 1
        IF( IFWT.NE.0 ) NACC = 2
        IF( INTRNE,NE,0) NABC = 5
        SUMSD=0.000
        CHISO = 0.000
        SUMSOU = 0.000
        CHISOU = 0.000
        NONO = 0
        NDROP=0
        LINE50
155      C
        160      165
C
        170

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SUBROUTINE OUTPUT 74/74 OPT=1 FTN 4.2+74278 11/22/74 10-10-37.

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      GO TO 113
      90 JJ=NURXYH*(N-1)
      DO 100 L=1,NVAR
      KK=JJL
      100 POINT(L)=STRING(KK)
      JJ=MURXYH
      KHT=STRING(4JJ)
      C 110 YPREDEFCONST
      DO 120 I=1,NBPMON
      J=INODE(I)
      120 YPREDEF*YPREDCOFF(I)+POINT(J)

      YOBS=POINT(MCVARD)
      DEV=YRED-YOBS
      DEVN=DEV/SIGN
      IF(1 NEFR .LE. 0 ) DEVN = 0.1
      PCT = (DEV*100.0)/YOBS
      GOOD = BLANK
      IF (A8ICEVH*GT.3*5) GOOD=CHECK
      GO TO (150,130), KPATH
      130 DO 140 J=1,BAD*2
      IF (BLT.NGCCD(IJ).OR.NGT.NGODD(IJ+1)) GO TO 140
      GDD=DID
      BAD=BAD+1
      IF (BAD.ED.NSKIP) KPATH=1
      GO TO 160
      140 CONTINUE
      150 DEVSQ = (DEV*DEV)*WHT
      SUMSQ = SUMSQ + DEVSQ
      CHISQ = CHISQ + DEVSQ/YPRED
      160 LINE=LINE+1
      N = NEC

      ARC(1) = GGOOD
      ABC(12) = WHT
      170 IF (NTRAI.170.*190.*1A0
      170 IF (YOBS.GT.15.*.OR.YPRED.GT.15.) GO TO 185
      YO = EXP ( YOBS ) - YCONST
      VC = EXP ( YPRED ) - YCONST
      GO TO 200
      180 IF (YOBS.GT.8..OR.YPRED.GT.8.) GO TO 185
      YO = 10.000*YOBS - YCONST
      VC = 10.000*YPRED - YCONST
      200 CONTINUE
      DEWU = YC - YD
      ABC(3) = YD
      ABC(4) = YC
      ABC(5) = DEWU
      IF (GOOD.EQ.VOID) GO TO 190
      DEVSQ = (DEWU*DEWU) * WHT
      SUMSQ = SUMSQ + DEVSQ
      CHISQ = CHISQ + DEVSQ
      GO TO 190
      195 ADNO = 1
      M = 2
      190 CONTINUE
      IF (LINE.LE.50) GO TO 210
      
```



```

SUBROUTINE PRSUM   7474  OPT=1          FTN 4.2+74278    11/22/74  10-10.A6.

      C SUBROUTINE PRSUM
      C   S/R TO PRINT SUMS OF CROSS PRODUCTS
      C   COMMON SIGMA(60),A(52,52),SIMCOR(52,52),AVG(60),TEST(60)
      C   COMMON PCNT(60),STRING(3000),INOPAC(30,30),INDEX(61)
      C   COMMON INDEX(60),MOUT(60),KSTEP(60),ALPHA(10),VMEAN,IOEN,IFAVE
      C   COMMON MAXTP,TFCNCH,NSUMR,NSKP,NTAPF,NEW
      C   COMMON NOVAR,NOBNOR,NOSTEP,NOATA,NBRXYN,NBRY,LPATH,OFFER,KVAB
      C   COMMON IPBACK,IFCNS,IFCGR,NPROB,TCL,MODEL,JF
      C   COMMON INDEXY,LBAO,NOG000128
      C   COMMON IENT,ICONST,MYTRA,Y12,S11,YTRA12
      C   COMMON STOERR150,COSQR150
      C   REAL A,SIMCOR,SIGMA,AVG,TEST

      15   C DATA JB/1M /
      C       WRITE (6,150) ((J3*I*A(I,NBRXYN)),I=1,NBRXY)
      C       WRITE (6,160) ((J3*I*A(I,NBRXYN)),I=1,NBRXY)
      C       WRITE (6,170) A(NOVAR,NOVAR)
      C       WRITE (6,180)
      C       WRITE (6,190) ((JB*I,J,A(I,J)),J=1,NBRXY),I=1,NBRXY
      C       WRITE (6,200) ((JB*I,A(I,NVAR)),I=1,NBRXY)
      C       WRITE (6,210) A(NOVAR,NOVAR)
      C       RETURN
      C
      C *****S/R TO CALCULATE AND PRINT THE RESIDUAL SLM CF SQUARES AND C.F.*****
      C
      C ENTRY TO STOP
      C IF (A(INCRNM,NBRXYN)) 10,10,23
      10  MWRITE (6,220) NPROB
      C CALL EXIT
      20  Q 1=1,NOVAR
      30  Q 1=1,NOVAR
      30  A(I,J)=A(I,J)-(A(I,NBRXYN)*A(J,NBRXY))/A(NBRXYN,NBRXYN)
      40  AVG(I)=A(I,NBRXYN)/A(NBRXYN,NBRXYN)
      IF (IAVE,NE,0) GO TO 50
      MWRITE (6,230)
      WRITE (6,190) ((JB,I,J,A(I,J)),J=1,NBRXY),I=1,NBRXY
      MWRITE (6,200) ((JB*I,A(I,NVAR)),I=1,NBRXY)
      MWRITE (6,210) A(NOVAR,NOVAR)
      51  RETURN
      C
      C *****S/R TO CALCULATE AND PRINT THE SIMPLE CORRELATION COEFFICIENTS*****
      C
      C ENTRY CORREL
      45  Q 1=1,NOVAR
      00  Q 1=1,NOVAR
      IF (A(I,I)) 60,60,80
      60  MWRITE (6,240)
      IF (I.EQ.,NOVAR) CALL SLITE(I)
      SIGMA(I)=0.0
      00  Q 1=1,NOVAR
      70  J=1,NOVAR
      A(I,J)=0.0
      70  A(I,J)=0.0
      GO TO 90
      CC 80  SIGMA(I)=OSORTIA(I,I,I)
      80  SIGMA(I) = SQRT ( A(I,I) )
      90  A(I,I)=1.000
      00  100  I=1,NBRXY
      55

```

```

SUBROUTINE PRTSUM      76/74      OPT=1      FTN 4.0-74278      11/22/74  10:10:42.

      IP1=I+1
      DO 100 J=IP1,N0VAR
     100 A(I,J)=((I,J)/(SIGMA(I)*SIGMA(J)))
      DO 110 J=1,N0VAR
     110 A(I,J)=A(I,J)
      DO 110 K=1,N0VAR
     110 SIGCOR(I,J,K)=A(I,J,K)
      IF (IFCRST.NE.0) GO TO 140
      IF (IFCORR) I40=120,I40=140
     120 WRITE (6,250)
      IF (NBRX.L.E.1) GO TO 135
      N0H2=NBRX-1
      130 I=1,N0VAR2
      131 IP1=I+1
      132 WRITE (6,260) (J3,I,J,A(I,J),J=IP1,N0RX)
      135 WRITE (6,270) (JB,I,A(I,N0VAR),I=1,NBRX)
     140 RETURN

      C
      C
      150 FORMAT (1H0,40X,16HSUM OF VARIABLES/ )
      160 FORMAT (4(1A11H
     161          SUM X(12,3H) =F14.4))
      170 FORMAT (6X,1HSUM
     171          Y =F14.4)
      180 FORMAT (1H07H
     181          )
      190 FORMAT (3(1A16H
     191          X(12,7H) VS X(12,3H) =F17.6)
     192          X(12,12H) VS Y =F17.6)
      200 FORMAT (5X,1HY
     201          VS Y =F17.6)
      220 FORMAT (132H0 ZERO NUMBER OF DATA, PROBLEM 16,/ )
      230 FORMAT (1H022X56HSUMS OF SQUARES AND CROSS PRODUCTS
     231          MEAN,/ )
      240 FORMAT (10H0 VARIABLE15,13H IS CONSTANT //,
     241          CORRELATION COEFFICIENTS,/ )
      250 FORMAT (1H0,3X,3SHSIMPLE
     251          )
      260 FORMAT (3(1A16H
     261          X(12,7H) VS X(12,3H) =F12.8*5X),
     270 FORMAT (3(1A16H
     271          X(12,12H) VS Y =F12.8*5X),
     272          )
      END

```

```

SUBROUTINE A0010      74/74      OPT=1          FTN 4-2+74270    11/22/74   18-18-52.

      C      SUBROUTINE A0010
      C      S/R TC A00 A VARIABLE
      C      COMMON SIGN(160),A(52-52),SIMCOR(52-52),AVG(68),TEST(68)
      C      COMMON POINT(160),STRNG(13000),INOPAC(130-130),IMEX(61)
      C      COMMON INDX(160),MOUT(160),NSTEP(160),ALPHA(16-16),YMEAN,IDEF
      C      COMMON MAXSTP,MSPATP,NTAPE9,LEN
      C      COMMON IFCNM,MSUNRY,MSKTP,LIPATN,DEFRM,KVAS
      C      COMMON IBACK,IFCMST,IFCRR,MPROB,MBPRV,TOL,MODEL,JF
      C      COMMON INDXY,LBD,NODG0(28)
      C      COMMON IFNT,YCONST,NYRA,V(2,51),YTRA(21)
      C      COMMON STORE(59),COSQR(50)
      C      EQUIVALENCE (KVAR,KI)
      REAL      A,SIMCOR,SIGMA,AVG,TEST

      C      REAL      DA,VAR,VMIN,VMAX
      C      C      NBRMON = NUMBER OF COEFFICIENTS FOR PRESENT EQUATION
      C      C      NBRPRV = NUMBER OF COEFFICIENTS FOR PREVIOUS EQUATION
      C      C      INDX = INDEX OF PRESENT EQUATION
      C      C      INDXP = INDEX OF PREVIOUS EQUATION
      C      C      00 16 J=1,NBRMON
      C      16 CALL PCK(NBRMON,J,INDEX(J),2)
      C      ON 20 J=1,NBRX
      C      20 MOUT(J)=0
      C      IF (LBRANCH) 50,50,30
      C      30 NO 40 J=1,NBRMON
      C      NO UMMY=INDEP(JJ)
      C      NO UMMY=INDEP(JJ)
      C      NO UMMU(NUMMY)=1
      C      50 VMAX=-1.0
      C      FINO LARGEST DELTA      (VAF = CEL1A)
      C      00 70 I=1,NBRX      BYPASS IF ALREADY IN EQUATION
      C      IF (NOUT(I).NE.0) GO TO 70
      C      IF (AT(I,I).GE.TOL) GO TO 68
      C      WRITE (6,510) AT(I,I),I,(INDEX(J),J=1,NBRANCH)
      C      GO TO 70
      C      60 VAR=AT(I,NOVAR)*(NOVAR,I)/AT(I,I)
      C      IF (VAF.LE.VMAX) GO TO 70
      C      VMAX=VAR
      C      K=I
      C      70 CONTINUE
      C      C      HAVE FOUND OPTIMAL VARIABLE
      C      NSTEP=NSTEP+1
      C      IF (VMAXI .GE. 90,90,90
      C      90 WRITE (6,520) JMAX
      C      CALL SLITE(1);
      C      GO TO 260
      C      90 MBPRV=NBRMON
      C      NBRMON=NBRMON+1
      C      IF ((TEST(1NBNOW)-A(1NOVAR*NOVAR)+VMAX) 100,100,120
      C      100 NWRITE (6,530) K,NBRMON,NSTEP
      C      00 110 I=1,NOVAR
      C      DO 110 J=1,NOVAR
      C      110 AT(I,J)=SIMCOR(I,J)
      C      LPATH=2
      C      GO TO 260
      C      ADD VARIABLE TO INDX

```

```

SUPERJUINF AJJO 74/74 OPT=1          FTN 4.2+74270      11/22/74 18.18.52.

120 CONTINUE
      TF (NBRPVR) 230+230+130
      130 99 140 J=1,NBRPVR
      140 CALL PACK (NBRPVR,J,INDEXP(J),2)
      150 DO 150 J=1,NBRNOW
      150 CALL PACK (NBRNOW,J,INDEXP(J),2)
      160 IF (INDEXP(J)=K) 180,160,170
      160 CALL SLICE (J)
      170 WRITE (6,400)
      170 GO TO 260
      171 JJ=J
      180 GO TO 193
      181 CONTINUE
      INDEXP(NBRNOW)=K
      GO TO 210
      190 L=NBRNOW-JJ
      200 TO 200 J=1,L
      NP=NBRNOW+1-J
      NS=NBRNOW-J
      210 INCE P(K2)=INDEXPNSI
      INDEXP(JJJ)=K
      C     CHECK TO SEE IF SET HAS ALREADY BEEN COMPUTED
      210 30 220 JE1,NBRNOW
      CALL PACK (NBRNOW,J,1,2)
      IF (INDEXP(J)=NE,J) GO TO 240
      220 CONTINUE
      K=ITE (16,450) NOSTFP,K,NBRACH,(INDEX(J),J=1,NBRNCH)
      LPATH=3
      GO TO 260
      C
      230 INDEXP(1)=K
      240 TEST(INPNC)=A(NBVAR,NBVAR)-VMAX
      C     NEW SET - PUT INDEXES IN MATRIX
      250 DO 250 J1,NBRNOW
      250 CALL PACK (NBRNOW,J,INDEXP(J),1)
      LFATH=1
      IF (MAXSIP.EQ.999) GO TO 255
      IF (INSTP.EC.1) GJ 10 256
      255 WRITE(6,500) NSTEP,K, V(1,K),V(2,K)
      256 KSTEP=NBRNCH=NSTEP
      260 RETURN
      100 C
      C     ADJUST CORRELATION MATRIX FOR FNTPANCE OF VARIABLE K
      C     ENTRY MATRIX
      105 C
      DA=1.0/A(K,K)
      DO 310 I=1,NBVAR
      310 IF (I-K).GT.0,300,270
      270 ON 290 J=1,NBVAR
      IF (J-K).LT.0,290,210
      280 AII,J=AI(I,J)-AI(I,K)*A(I,J)*DA
      290 CONTINUE
      AI,I,K=-AI(I,K)*DA
      300 CONTINUE

```

```

SUBROUTINE ADD10      74/74   OPT=1          FTH 4.2+74278    11/22/74  18.10.92.

115      DO 120 J=1,NCVAR
        IF (J-K) 310,320,310
310      A(K,J)=A(K,J)*OA
320      CONTINUE
        A(K,K)=OA
C       IF (NSTEP.LE.1) NCUMB=MIN(1000,12.***INCVAR-1)
C       IF (NSTEP.LE.NCMOD) GC TC 33)
        WRITE (6,540) NPMOD
540      CALL SLITF (1)
330      RETURN
C       *****S/R TC DELETE A VARIABLE
C       ENTRY RMVNE
C
120      NODEL = 0
        IF(I .NE. NORMALE-1) GO TO 470
        DO 340 J=1,NBRNOW
340      CALL PACK (NBRNOW,J,INCEX(J),2)
K=0
        NSTEP=NSTEP+1
        VMIN=-2.0E+30
        FINO SMALLEST DELTA
        DO 160 J=1,NBRNOW
160      I=INDEX(J)
        IF (I(A,I,I)-TCL) 360,369,350
350      VAR=A(I,NOVARI)*ANOVAR(I)/AIL,I)
        IF (IVAR.LE.VMIN) GC TC 360
        VMIN=AP
        K=I
360      CONTINUE
        IF (TEST(NBRNOW-1)+V1N-ANOVAR,NOVARI) 370,370,380
370      CONTINUE
        GO TO 470
C       REMOVE VARIABLE TO BE DELETED FROM INDEX
C
150      JP = 0
        CO 390 J=1,NBRNOW
        IF(I .NE. INDEX(J).EQ.K) GC TO 390
        JP = JP + 1
        INCEX(J) = INDEX(J)
        NRPUP = NBRNOW - 1
        CONTINU
C       CHECK TO SEE IF SET HAS ALREADY BEEN COMPUTED
390      IF (I(.NE.I-TPIX)) GO TO 450
        DO 440 J=1,NBRPNR
        CALL PACK (NBRPNR,J,1,2)
        IF (I(.NE.I-TPIX)) GO TO 450
440      CONTINUE
        WRITE (6,550) K
        GO TO 470
C       NEW SET - PUT INDEXES IN VAPIX
165      NODEL = 1
        NBRNOW = NBRNOW - 1
        TEST(NBRNOW)--VMIN+ANOVAR,NOVARI)
        DO 460 J=1,NBRNOW
460      CALL PACK (NBRNOW,J,1,NOEXP(J),1)
        WRITE (6,560) NSTEP,K
        KSTEP(NBRNOW)=NSTEP
        GO TO 170
C
170      END

```





```

SUBROUTINE CHANGE 74/74 OPT1=1
C SUBROUTINE CHANGE (PCINT, NRTRIA, CONST, NCTAL)
C INFORMATION OF DATA
C DEFINITION POINTS), NRTRIA(60), CONST(60)
5   C
      DO 220 I=1,NTOTAL
        ITRA=NRTRIA(I)
        IF ((ITRA.LE.0) GO TO 220
        IF ((ITRA.LT.-17) GO TO 10
        5  NRTRIA(60+30)=ITRA
        GO TO 220
      C
      1 J  C=PCINT(I)
      C=CONST(I)
      IF ((ITRA.GT.0) GO TO 120
      2 G  TG (TG (20+30+40+50+60+80+100+110)+ ITRA
      2 G  D=J+6
      GO IJ 210
      3 J  C=D*C
      GO T5 213
      4 J  C=D/C
      GO TC 210
      5 J  C=C/C
      GC TC 211
      6 J  IF ((C.GT.J-3)) GO TO 76
      C=D*C
      GO T5 212
      7 0  C=1.D+0+C
      GO TC 211
      A1  IF ((D.LT.0.0)) GO TO 30
      C=C*D
      GO TC 213
      8 R  C=1./C*(1.-R)
      GO TC 211
      100 D=ALOG(D0+C)
      GC TC 211
      11 J  D=ALOG10(J+C)
      GO T5 210
      C
      12 0  ITRA=ITRA-6
      IF ((C.GT.C-J,0)) C=1.
      GO TG (130+140+150+160+170+180+190+200)+ ITRA
      13 0  C=EPIC(C)
      GO TG (210
      14 0  C=EXP(C/J)
      GO TG 210
      15 0  D=SIN(C)
      GO T5 210
      16 0  C=COS(C)
      GO T5 210
      17 0  D=SEN(C)/COS(C)
      CC14) C=SINH(C)
      CC  GG TG 210
      18 0  C=COSH(C)
      CC190 C=COSH(C)

```

SUBROUTINE	CHARGE	7474	OPT=1	FTN 4.2+74276	11722/74 10.2E95.
CC	CC T1 21)				CHAN 580
19)	CC T2 5				CHAN 570
20)	C=1AHC(G*)				CHAN 560
21)	PCINT(I)=0				CHAN 550
22)	CGNTNL				CHAN 540
	ORTNLSH				CHAN 530
C					CHAN 610
C					CHAN 620
65					CHAN 630
					CHAN 640
					CHAN 650
					CHAN 660

23) FORWARD (/1X16-WTRANSFORMATICN). ITEM IS NOT IN TABLES. SET IT WILL FCBN  
 1E SFT TO /FRO AND IGNORED. //  
 FNC

```

SUBROUTINE NRCFX      74/74   OPT=1          FTN 6.2+74272   11/22/74  16.25.C1.

SUBROUTINE NRCFX ( IFSUN, POINT )
C   DUMMY SUBROUTINE TO MAKE UNUSUAL TYPES OF TRANSFORMATIONS.
C   THIS SUBROUTINE IS REPLACED AT OBJECT TIME WITH ONE WHICH
C   PERFORMS THE DESIRED TRANSFORMATIONS.
5    POINT = DATA OBSERVATION WHICH WAS READ IN ON CARDS OR TAPE
C   IFSUN = A NUMBER GREATER THAN ZERO WHICH CALLS SUBROUTINE FCT
C   NRCFX.  MAY BE USED AS A BRANCH INDICATOR.
C   NXV = NEW NUMBER OF INDEPENDENT VARIABLES
C   MAKE SURE DEPENDENT VARIABLE MATCHES NXV + INDEX.
10   C/P FQALF IS USED TO ADD OR CHANGE DATA OBSERVATIONS
C
C   S/R NRCFX IS USED TO CHANGE THE VALUE OF NXV.
C   AND MUST BE USEC.
15   ****
C
EXAMPLE - POLYNOMIAL EQUATION
IFSLB = POWER CF X
Y = A0 + A1*X + A2*X**2 + A3*X**3 + ... + A(IFSLB)*X^IFSLB
C
C
20   DIMENSION POINT(52)
C EQUIVALENCE ( XV,NXV )
PRINT 401, IFSUB
401 FORMAT(1X, SUBROUTINE EQUAT WAS CALLED WITH IFSLB = I3)
NXV = IFSUE
POINT(1) = XV
25
NXV = NXV+1
NX=NXV
RETURN
50
C
FNTRY EQUAT
C   STORE Y
K=54-NX
L=53
D0 1 J=N+52
K=K-1
L=L-1
1) PCINT(L)=PCINT(K)
C
40   IF( NX.EC.1 ) RETURN
C   STORE POWERS OF X
D0 2 J=2,NX
20 POINT(J)=POINT(J-1)*POINT(1)
PFTURN
END
45

```





```

PROGRAM UTANAL    74/74   001+1           FIN 4.2+74278   11/21/74  10:52:100.

115      WALL LAGR01 A, 4, 76, RIGHT,
C
C      THICKNESS
C      0.246 MIG AL+S
TC = MIG
120      SEL, IC, GE, A, A, 60, 10, 200
C
C      6.4. NBLA
J0 230 JP=1,101
325      C COMPUTE DYNAMIC CUSHIONING MODEL VARIABLES
C      60,10,200
222      CONTINUE
Y10P1 = CONST
DO 225 J=1, NV
1-100,1
225 Y10P1 = Y10P1 + COEFF(J) * V10
        100,1
C
C      INSERT REVERSE-YAW TRANSFORMATION-METHOD
C      230 A,X = X,A - OX
C
C      CALL RICARDIA-A, MECHANICAL-360-X-1
140      CALL RICARDIA-A, MECHANICAL-360-X-1
C
C      240 CONTINUE
C      CALL PRINTPL A, 66, OUTPUT
C      660 CONTINUE
260      CONTINUE
C      60,10,200
C
C      640-CONTINUE
C      640- DYNAMIC CUSHIONING MODEL
C
150      SS = SS + 100.
AL = ALD, SS
AL2 = AL * AL
SSH = SS-100
1-100,1
ICSH = IC ** (-3.5)
IR2 = IR * IR
IRd = IR * IRd
IRb = IRd * IR
C
160      TCOM = TC ** (-0.5)
I-1, N = IC ** (-1.5)
ICINV = IC ** (-2.5)
C
V10P1 = IR * TCOM * 1.0
410,21,1, IR * TCOM * 1.0 * AL
V10P1 = IR * TCOM * 1.0 * AL2
410,41,1, IR * TCOM * SRUM * AL
V10P1 = IR * TCOM * SRUM * AL2
410,61,1, IR * TCOM * SRDM * AL2
V10P1 = IR * TCOM * SRDM * AL2
410,81,1, IR * TCOM * SRDM * AL2

```





SUBROUTINE PRINTPL 74/74 OPT=1

FTN 4.2+ REL 07/25/74 14.19.05.

```

60      JL = IABS( N1 )
        IF( JL .EQ. 0 ) GO TO 90
        GVY = 10H
        GMH = 10H
        GMH = 10H
        GVY = GVY
        GH = GH
        GC = GC
        IF( N1 .GE. 0 ) GO TO 2
        GV = BLANK
        GG = GVY
        GO TO 70
    C   2  P111 = PMASK
          SET UP CODE THAT SFTGRID HAS BEEN CALLED
    C   NUMBER LINES TO GRAPH
    P(2) = JL
    75      JJ = 4
          00 4 I=1,JL
          G = GV
          GB = GVY
          JJ = JJ + 1
          IF( JJ .NE. 5 ) GO TO 3
          G = GMH
          GB = GG
          JJ = 0
          3  DO 4 K = 1,11
              P11(K) = G
              INO = 18 + (I-1)*11 + K
              P11D) = 5
              IF( K.EQ.1 .OR. K.EQ.11 ) P11(IND) = 5B
              IF( I.EQ.1 .OR. I.EQ.JL ) >(IND) = GMH
              CONTINUE
          C   COUNT OF POINTS THAT FELL OUT OF GRID
          P13) = 0.
          C   XMIN
          JK = N2
          P14) = XMIN
          YMAX
          YMAX = Z
          P15) = YMAX
          XMAX = X11)
          YMIN = Y11)
          X AND Y SCALE INCREMENTS
          SX = ( XMAX - XMIN ) / 100.
          SY = ( YMAX - YMIN ) / ( P12) - 1. )
          DX = 1.5 - 4MIN / SX
          DY = 1.5 - YMIN / SY
          P16) = SX
          P17) = SY
          P18) = OX
          P19) = OY
          C   SET TABLE ADDRESS = 0
          00 5 J=10,17
          6  P11) = 0.
          RETURN
    C

```

```

SUBROUTINE PRINTPL    74/74   O7P=1          FTN 4.2+ REL      07/25/74  14.19.06.

115      C***** CALL LAB GPO ( P, L * NC, LABEL ) *****
C      C      LABGRID PUTS LABELS ON GRID AXIS
C      C      NC = NUMBER OF CHAR. IN LABEL ARRAY = N2
C      C      LABEL = LABEL ARRAY
C      C      INTY LAB GRID
AJ = P(1)
CALL TMLABGRID
IF(I JX * NE, MASK) GO TO 98
LAB(LA = LDOF( X11 ) - LODF( P(11) ) + 1
J = N1*2 + 6
LABEL ADDRESS IN PREFERENCE TO P
PJ1) = LABELA
C      NUMBER OF CHARACTERS IN LABEL
PJ(J+1) = N2
RETURN
130      C***** CALL PLT GRID ( P, NSYM, NP, X, Y ) *****
C      C      PLTGRID ENTERS DATA INTO PLT GRID P
C      C      NSYM = PLOT SYMBOL ( EX, PSYM = 1M )
C      C      NP = NUMBER OF POINTS TO PLOT
C      C      INTY PLT GRID
CALL TMLPLTGRID
AJ = P(11)
IF(I JX * NE, MASK) GO TO 98
NP = N2
NSYM = N1
IF(I NP * LE, 0 ) RETURN
SX = P(6)
SY = P(7)
DX = P(8)
DY = P(9)
140      UO 30 L = 1, NP
IF(I LEGVAR( Y(L) ), NE, 0 ) 50 TO 20
IF(I LEGVAR( X(L) ), NE, 0 ) 60 TO 20
JL = P(2)
145      UO 30 L = 1, NP
IF(I LEGVAR( Y(L) ), NE, 0 ) 50 TO 20
IF(I LEGVAR( X(L) ), NE, 0 ) 60 TO 20
J = X(L) / SX + DX
I = Y(L) / SY + DY
IF(I JLT, 1 .02, JGT, 1.01 ) GO TO 23
IF(I ILT, 1 .02, IGT, 1.01 ) GO TO 23
C      INSERT PLOT CHARACTER
JW0>D = ( J-1 )/10 + 1
JPOS = J - ( JW0>D-1 )*10
IN0 = 16 + ( I-1 )*11 + JW0>D
DECODER( IN0, PIND ) ( JWORK(J), J=1,10 )
JWORK( JPOS ) = NSYM
150      305 FOR MAT10A1)
GO TO 30
20 P(31) = 2(3) + 1.0
30 CONTINUE
160      C
C      RETURN
C***** P, N1
C***** CALL PRINT PL ( P, NFILE ) *****
C      PRINTPL PRINTS GRAPH STORED IN ARRAY P ON OUTPUT FILE NFILE.
C      IT DOES NOT CLEAR THE GRID.
165      C

```

```

SUBROUTINE PRINTPL    74/74   OPT=1           FTN 6.2+ REL      07/25/74  14.19.06.

15 XJ = P(1)
      CALL = ?PRINTPL
      IF( JX .NE. MASK ) GO TO 90
175  INFILE = N1
      IF( N1 .EQ. 0 ) NFILE = 6 OUTPUT
      WRITE(INFILE,902)                               PRINT1600
      312 FORMAT(0F0)
C       PRINT TOP LABEL
      L1 = P(14)
      IF( L1 .EQ. 0 ) L1 = LOCFL( BLANK ) - LOCFL( P(1) ) + 1
      L2 = P(14) + P(15)/10.**05
      WRITE(INFILE,903) ( P(L), L=L1,L2 )
      306 FORMAT(1H1,(30X,10A10))
      WRITE(INFILE,907)
      307 FORMAT(8X *FR04  T0*)
      L1 = P(12)
      L2 = P(13)
      L3 = P(16)
      L4 = P(17)
      YMAX = 2.5
      SY = P(17)

190  C       Y1 = YM4X - 0.5*SY
      L = -6
      JL = P(2)
      JLPI = JL + 1
      DO 50 I=1,A
      L = L + 6
      IF( L .LT. 50 ) GO TO 35
      L1 = L1 + 1
      L3 = L3 + 1
      L = 0
      35 I02 = 1H
      I04 = 1H
      IF( L2 .EQ. 0 ) GO TO 40
      IF( I .GT. L2 ) GO TO 40
      I02 = SHIFT( P(L1), L )
      40 IF( L4 .EQ. 0 ) GO TO 45
      IF( I .GT. L4 ) GO TO 45
      I04 = SHIFT( P(L3), L )
      C       45 K = JLPI - 1
      Y2 = Y1 + SY
      CC: IN01= 18 +(K-1)*11
      IN01= 19 +(K-1)*11
      IN02= IN01+10
      WRITE(INFILE,908) I02,Y1*Y2*, ( P(J), J=IN01,IN02 ), 104
      308 FORMAT(1X4,1XF10.3,I0-3,I0-3, 1X,10A10,A1, 1YA1) PRINT1740
      220  Y1 = Y1 - SY
      50 CONTINUE
      C       SX = P(6)
      SXX = SX + 10.
      WORK(1) = P(4)
      00 60 J=2,11
      66 WORK(J) = WORK(J-1) + SXX
      WRITE(INFILE,910) ( WORK(J), J=1,11 )

```

```

SUBROUTINE PRINTPL    7/27/74   OPT=1           FTN 4.2+ REL      07/25/74  16.19.06.

      310 FORMAT(5X,11F10.2)
      C   310 FORMAT(LABEL
      C   L1 = P(10)
      C   L2 = P(10) + P(11)/10. - .05
      C   IF(L1.EQ.0.) GO TO 65
      C   ARITE(INFILE,915) 1 P(1), L=L1,L2 1
      315 FORMAT(10X,10A1)
      65 L = P(13)
      IF(L.LE.0.) GO TO 67
      WRITE(INFILE,982) L
      102 FORMAT(IX,17, 20H FJNTS OUTSIDE GRID 1
      GO TO 69
      57 WRITE(INFILE,916) SX, SY
      68 WRITE(INFILE,916) BLANK
      68 FORMAT(*'',35X,"DX =",F10.3)
      316 FORMAT(*'',35X,"DY =",F10.3)
      WRITE(INFILE,914)
      314 FORMAT(*$)
      RETURN
      C
      C   90 L = LOCFL(P(1))
      C   WRITE(INFILE,920) CALL,L
      250 FORMAT(*D*48----- S/R SETGRID NOT CALLED FOR P ARRAY AT LOC *06)
      LERR = LERR + 1
      IF(LERR .GT. 100 ) STOP 111
      RETURN
      END

```

**Appendix C**

**MLRD PLOTS OF DYNAMIC CUSHIONING CURVES OF THE  
UAH BEST FITTING POLYNOMIALS**

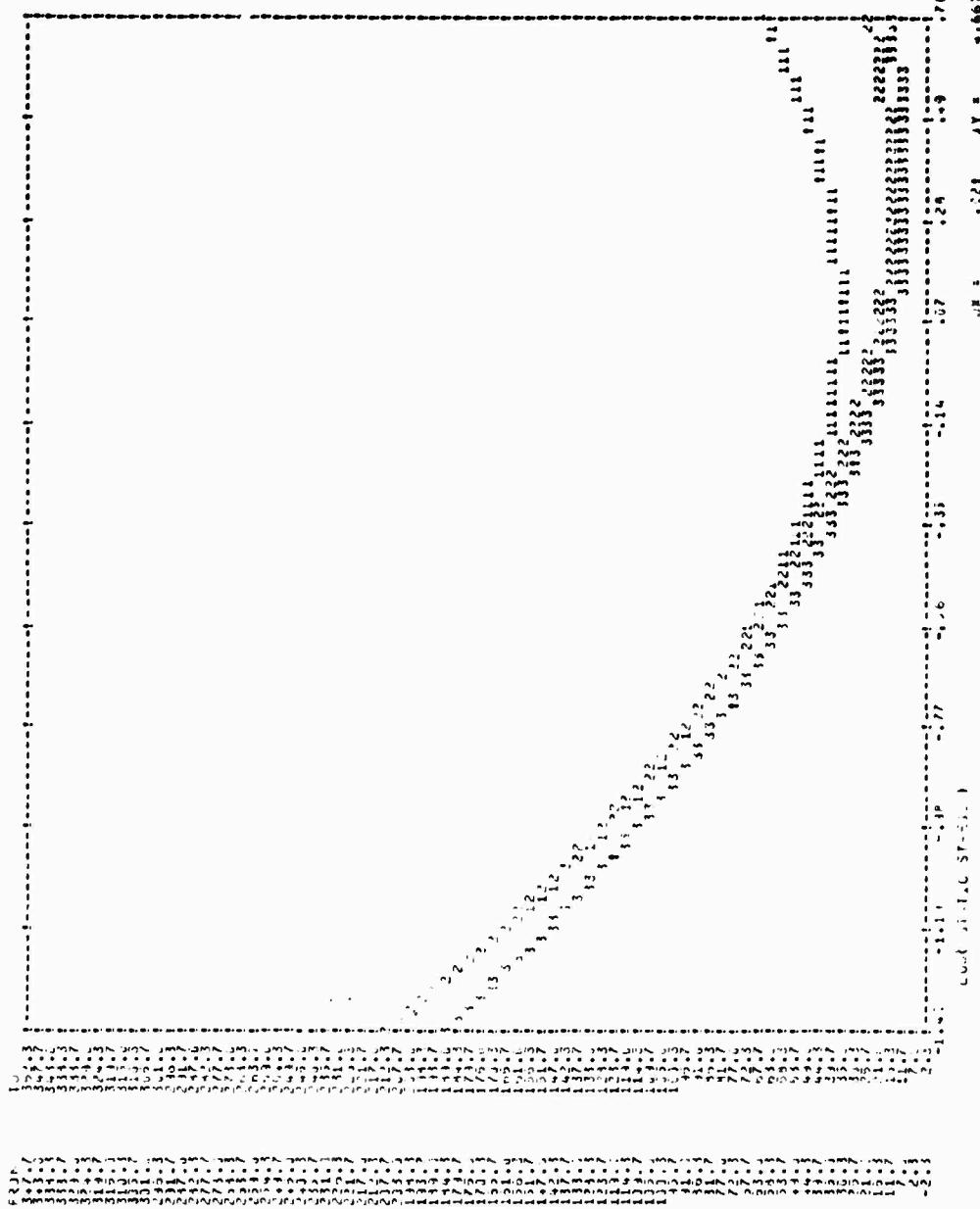


Figure C-1. -65°F, 12-inch drop height.

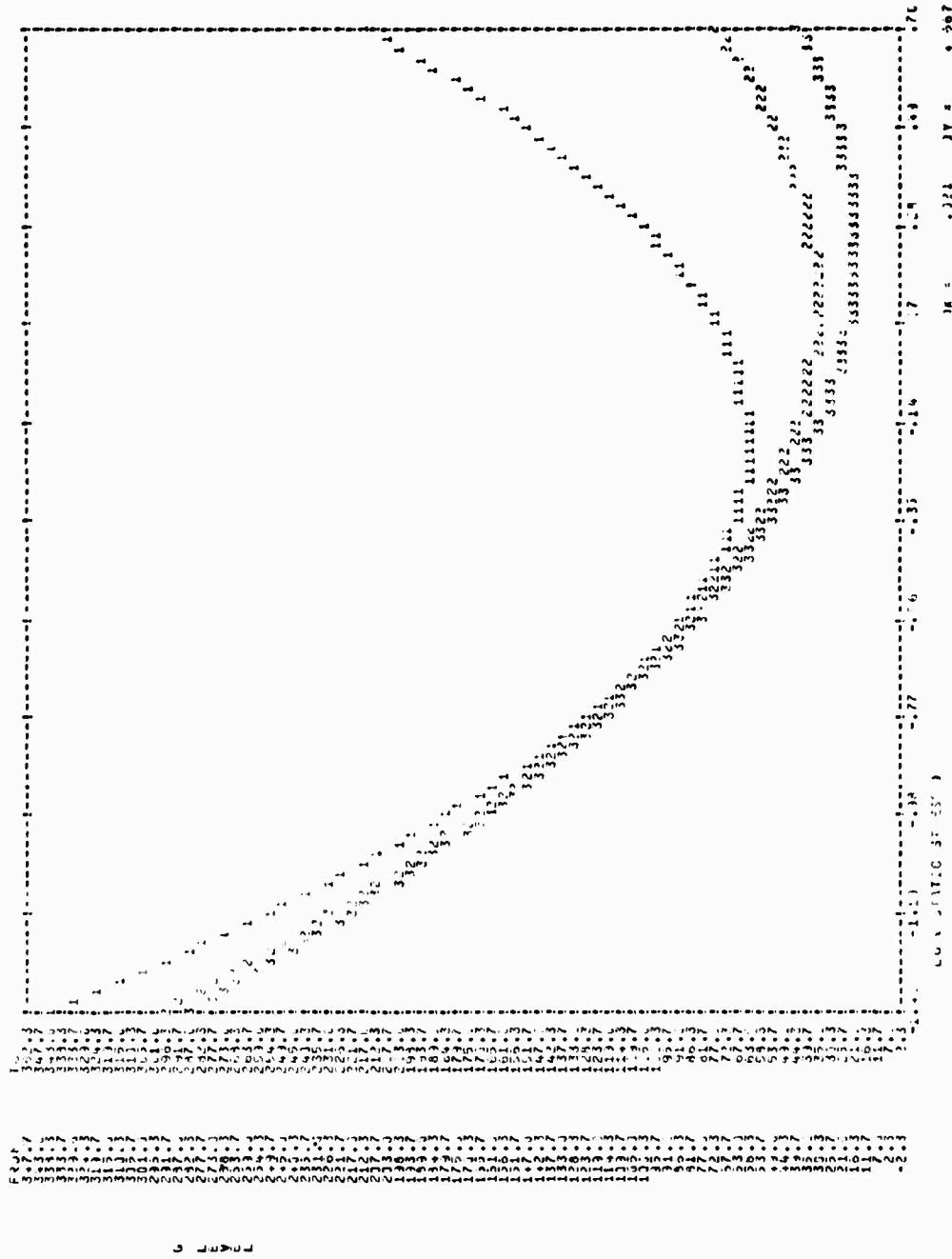


Figure C-2.  $-65^{\circ}\text{F}$ , 24-inch drop height.

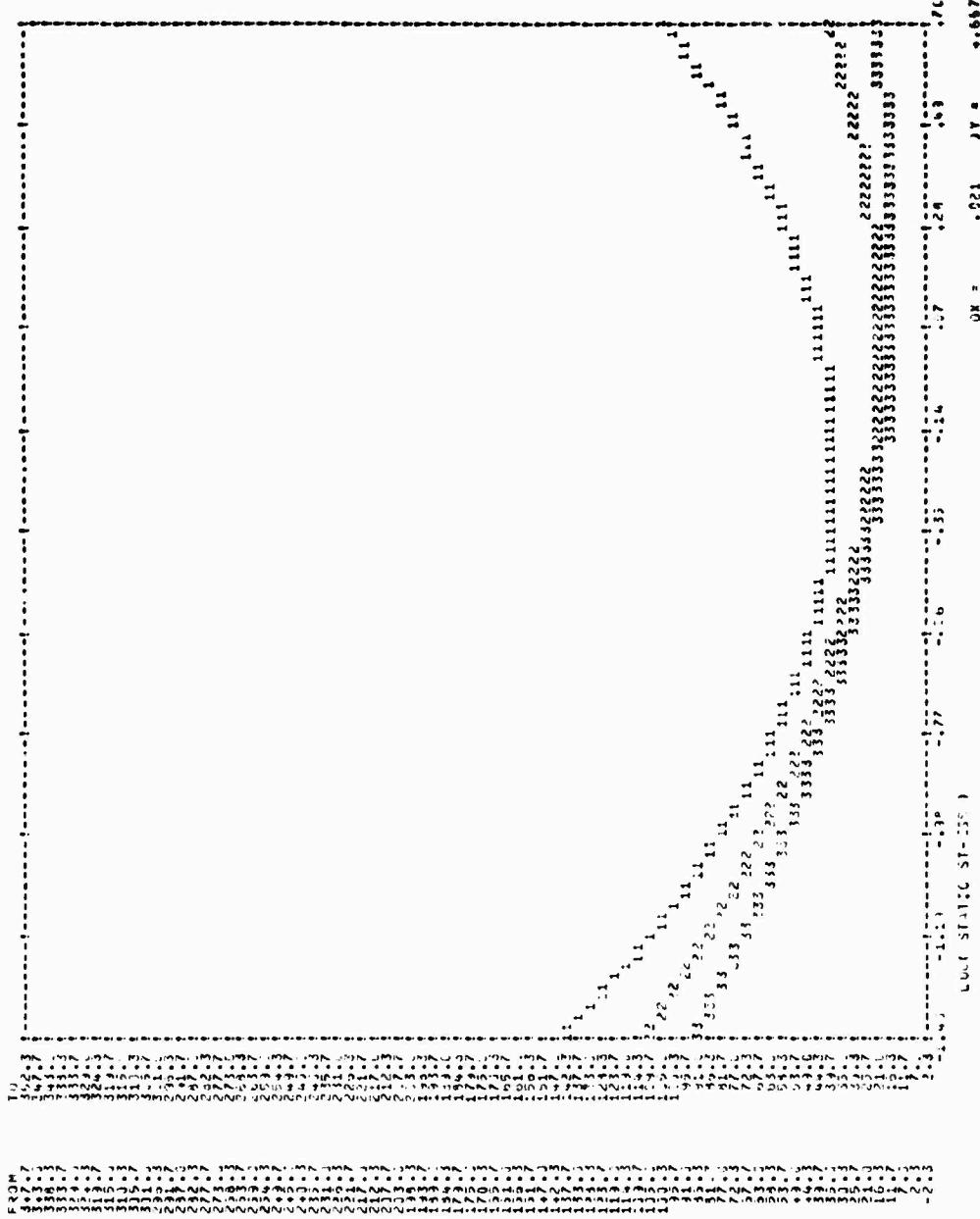


Figure C-3. 70°F, 12-inch drop height.

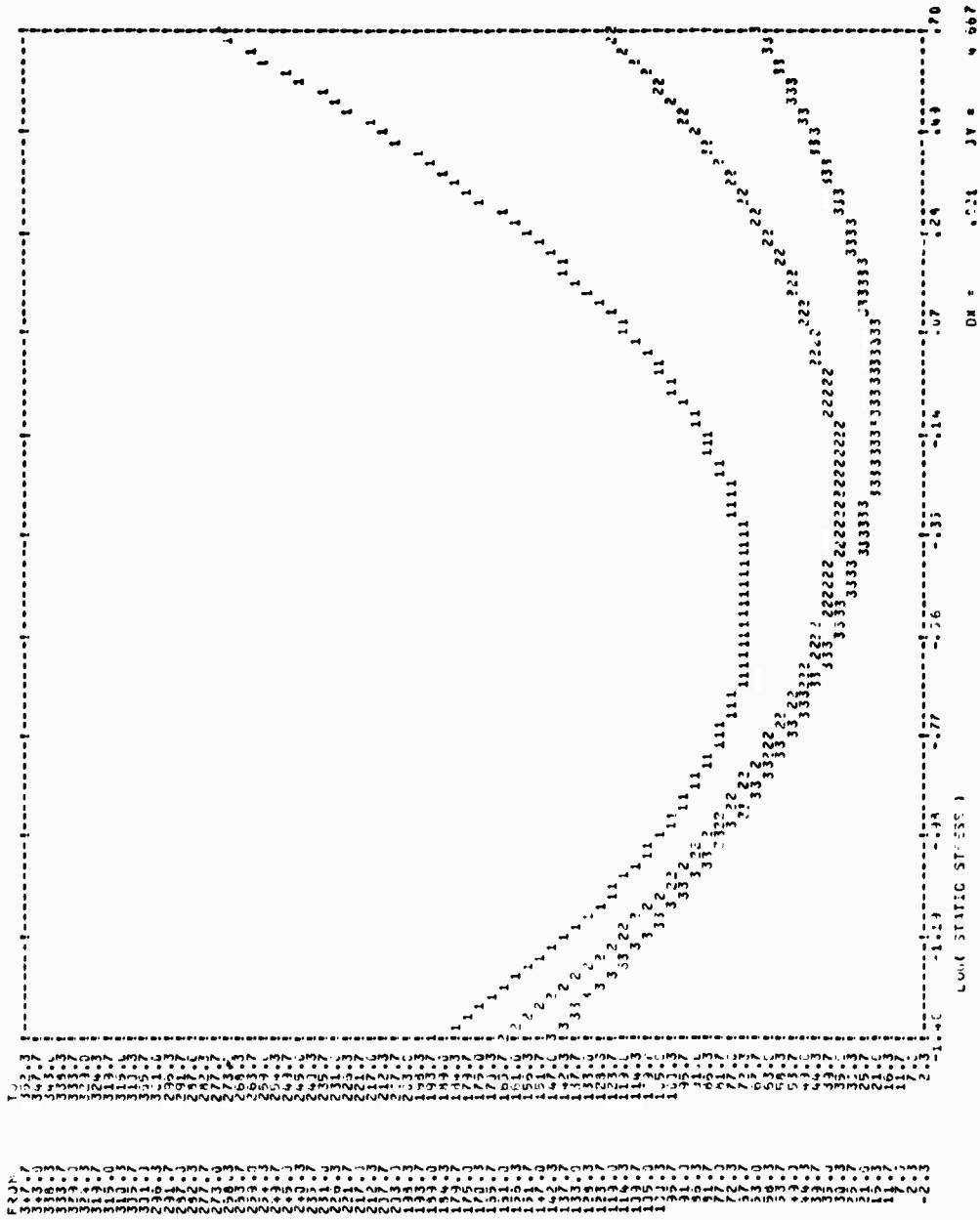


Figure C-4. 70°F, 24-inch drop height.

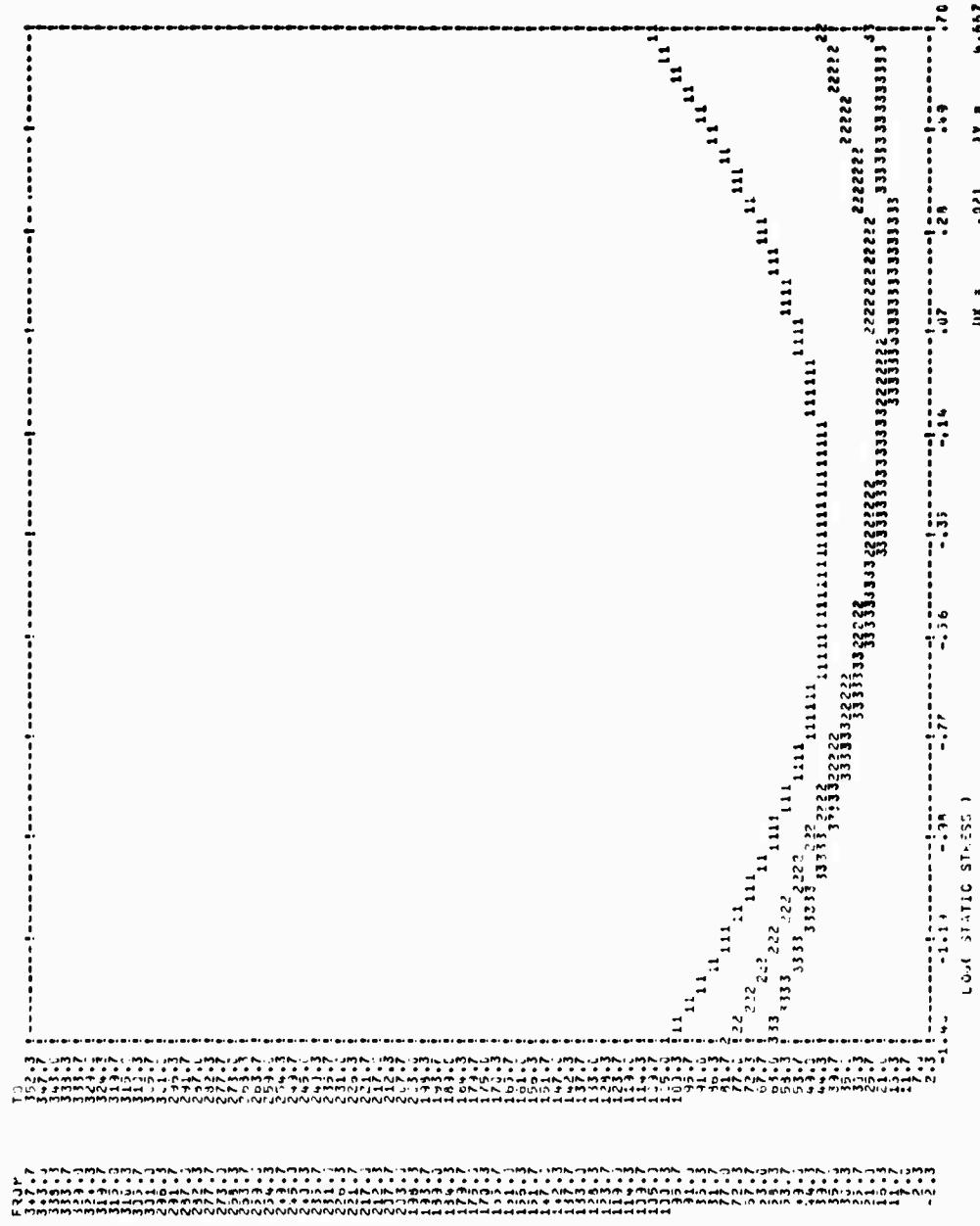


Figure C-5. 160°F, 12-inch drop height.

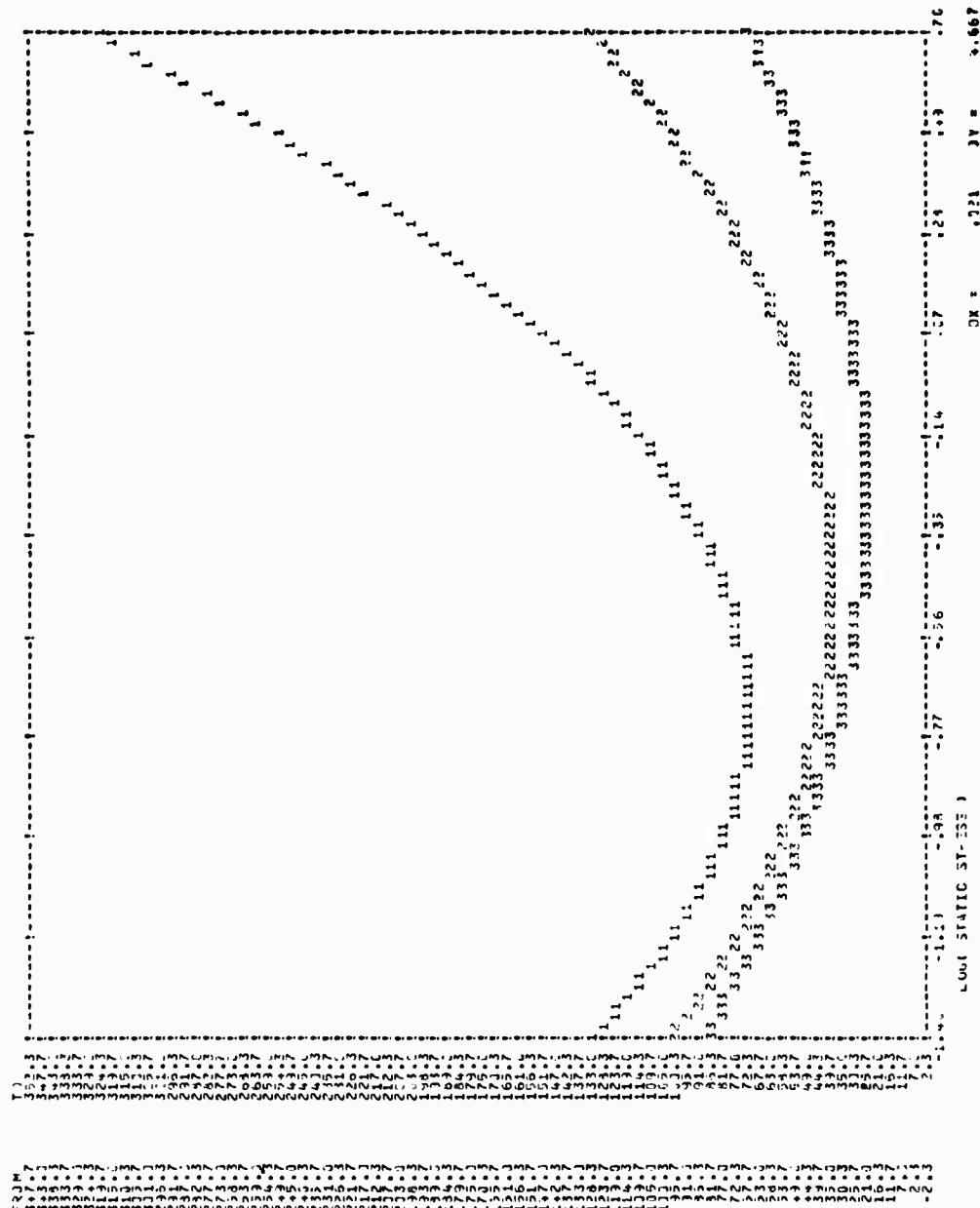


Figure C-6. 160°F, 24-inch drop height.

**Appendix D**

**MLRD PLOTS OF DYNAMIC CUSHIONING CURVES OF THE GENERAL MODEL**

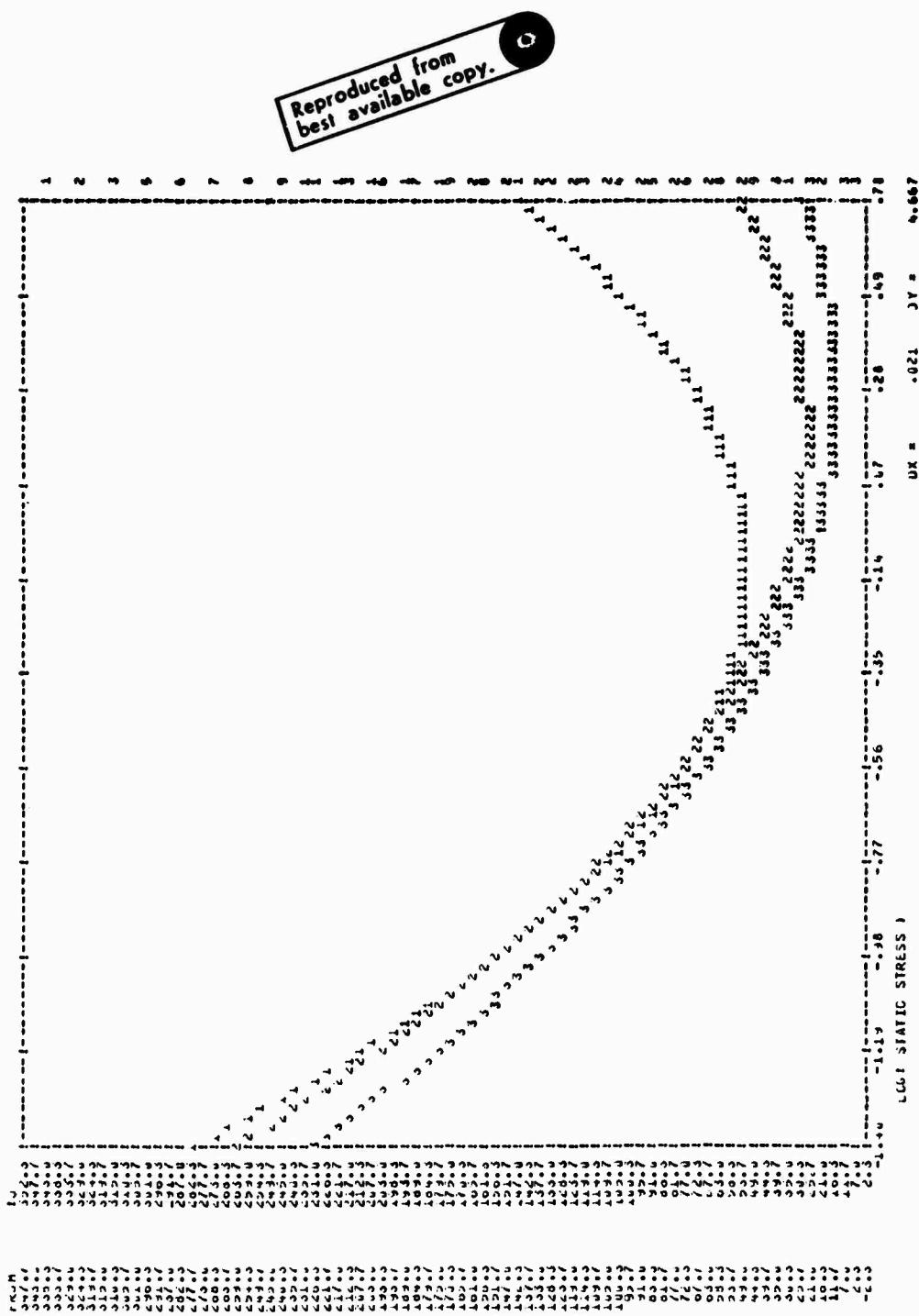
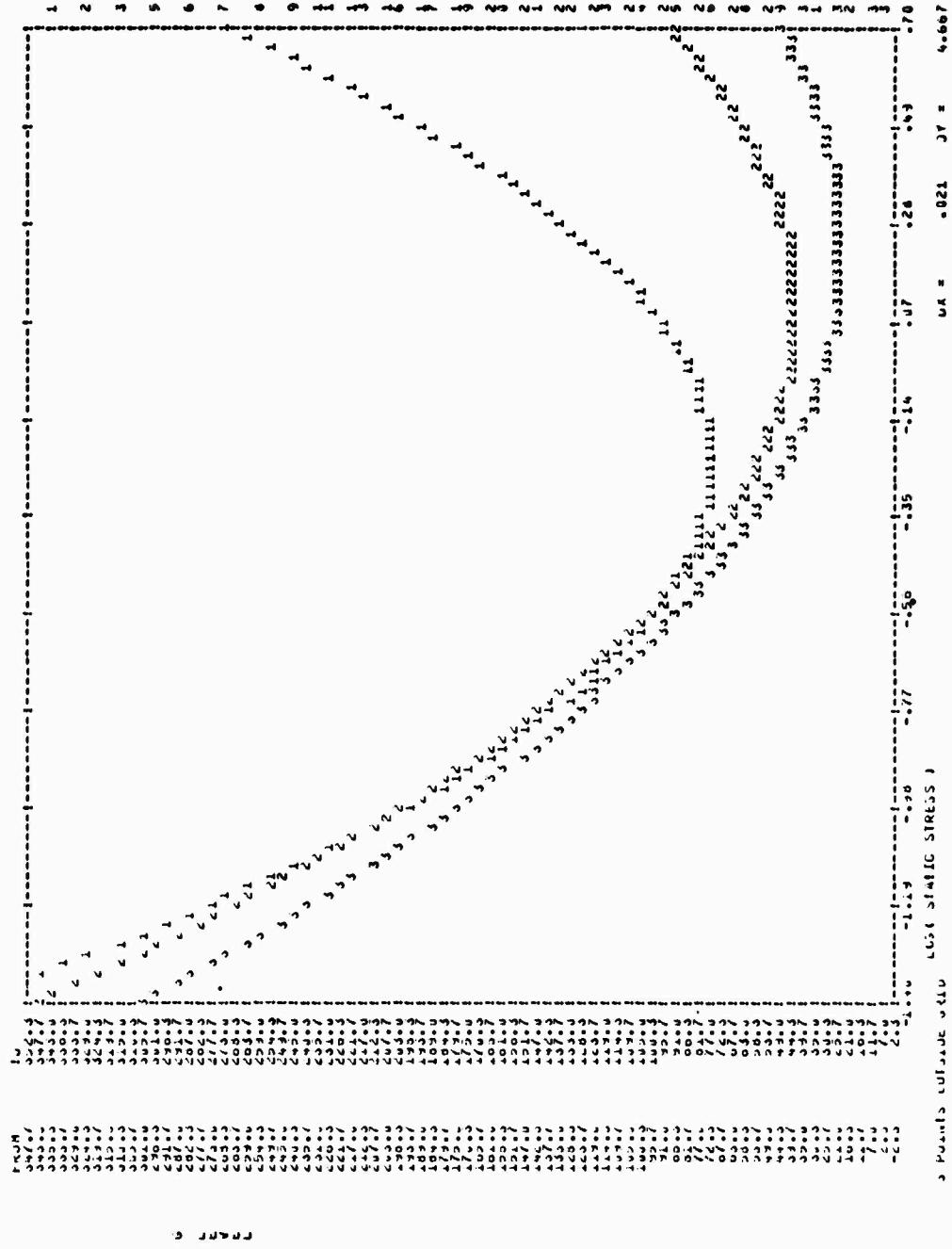


Figure D-1. -65°F, 18-inch drop height.



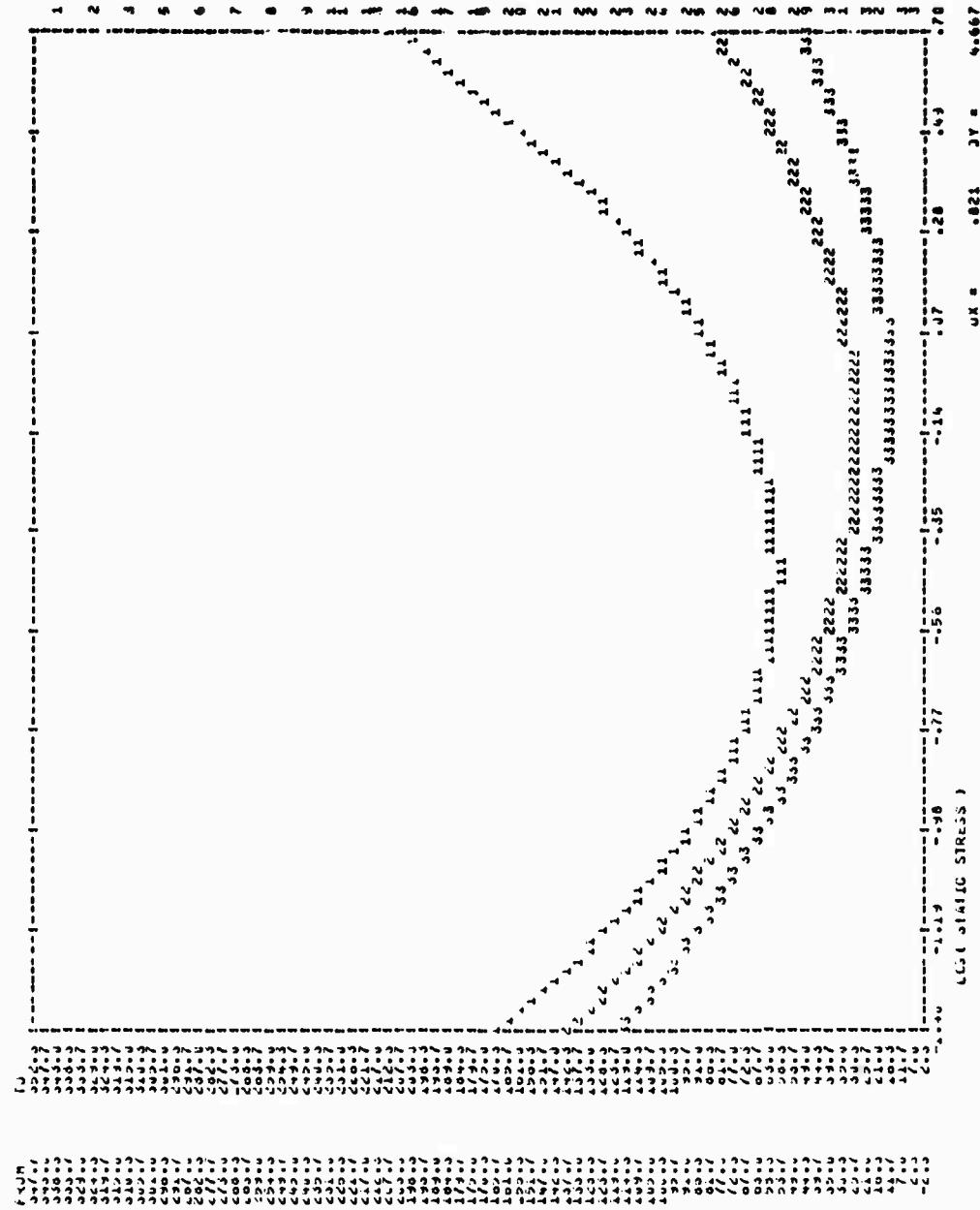
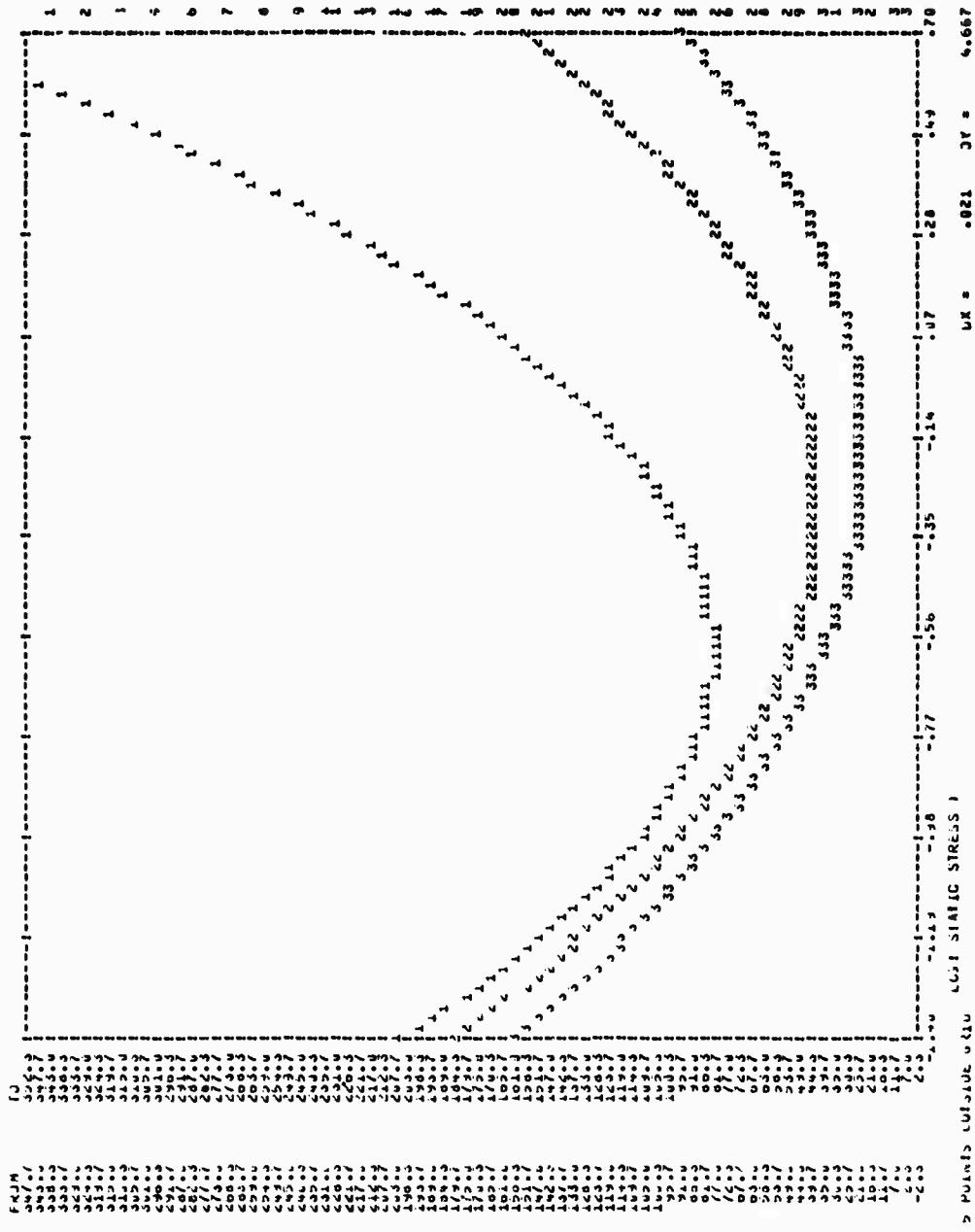


Figure D-3. 70°F, 18-inch drop height.



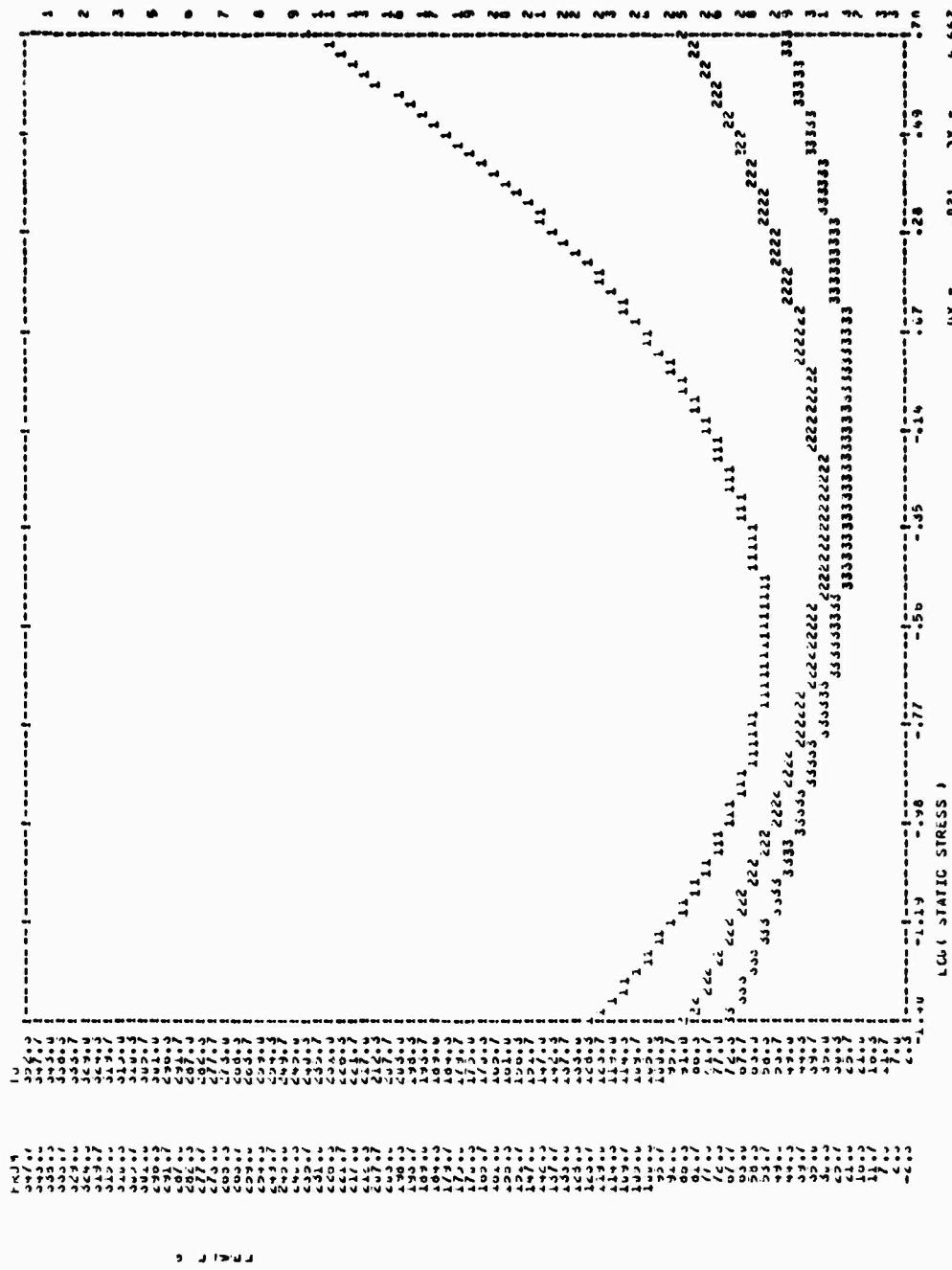
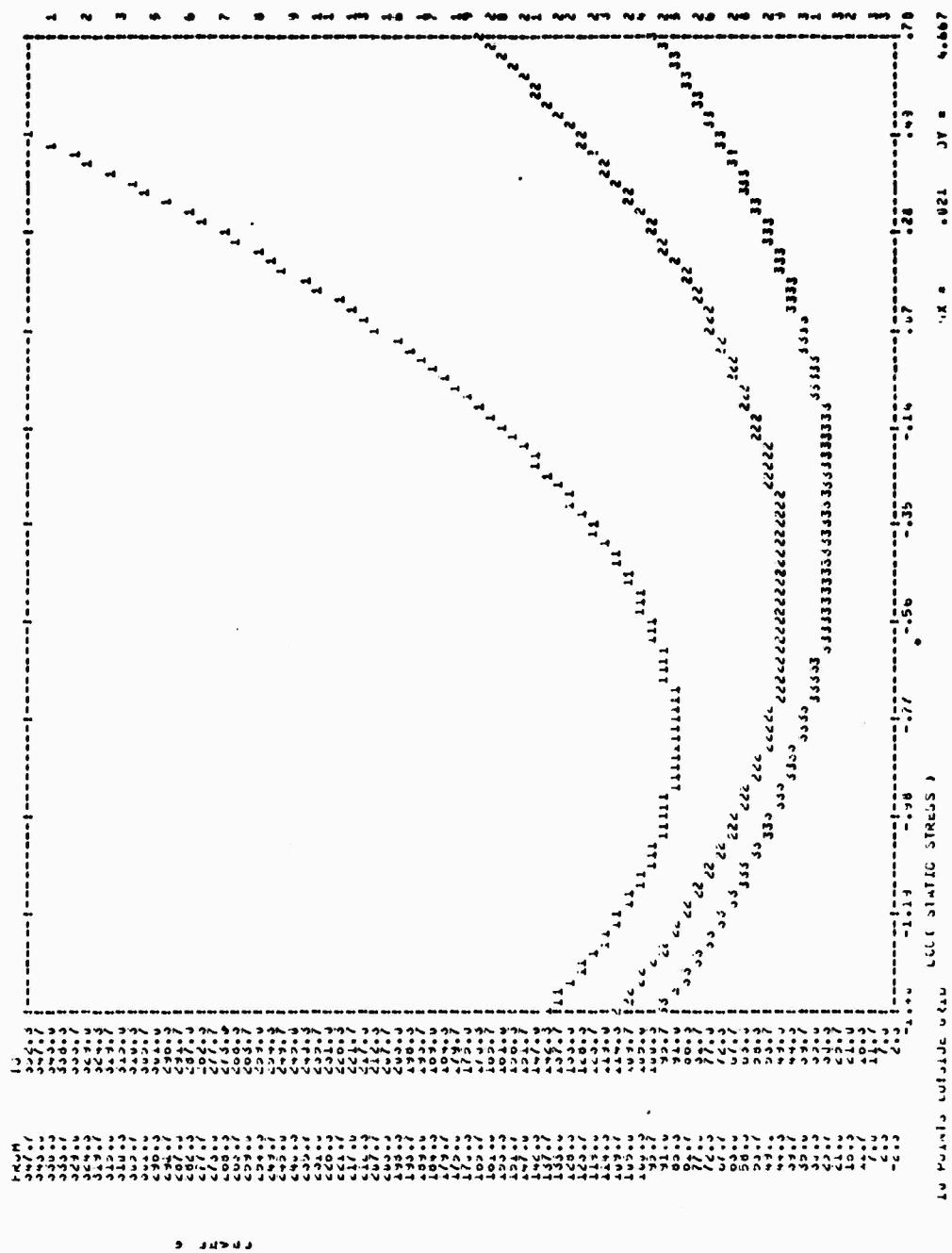


Figure D-5. 160°F, 18-inch drop height.



**Figure D-6.** 160°F, 30-inch drop height.

**Appendix E**

**MLRD PLOTS OF DYNAMIC CUSHIONING CURVES OF THE MINICEL MODEL**

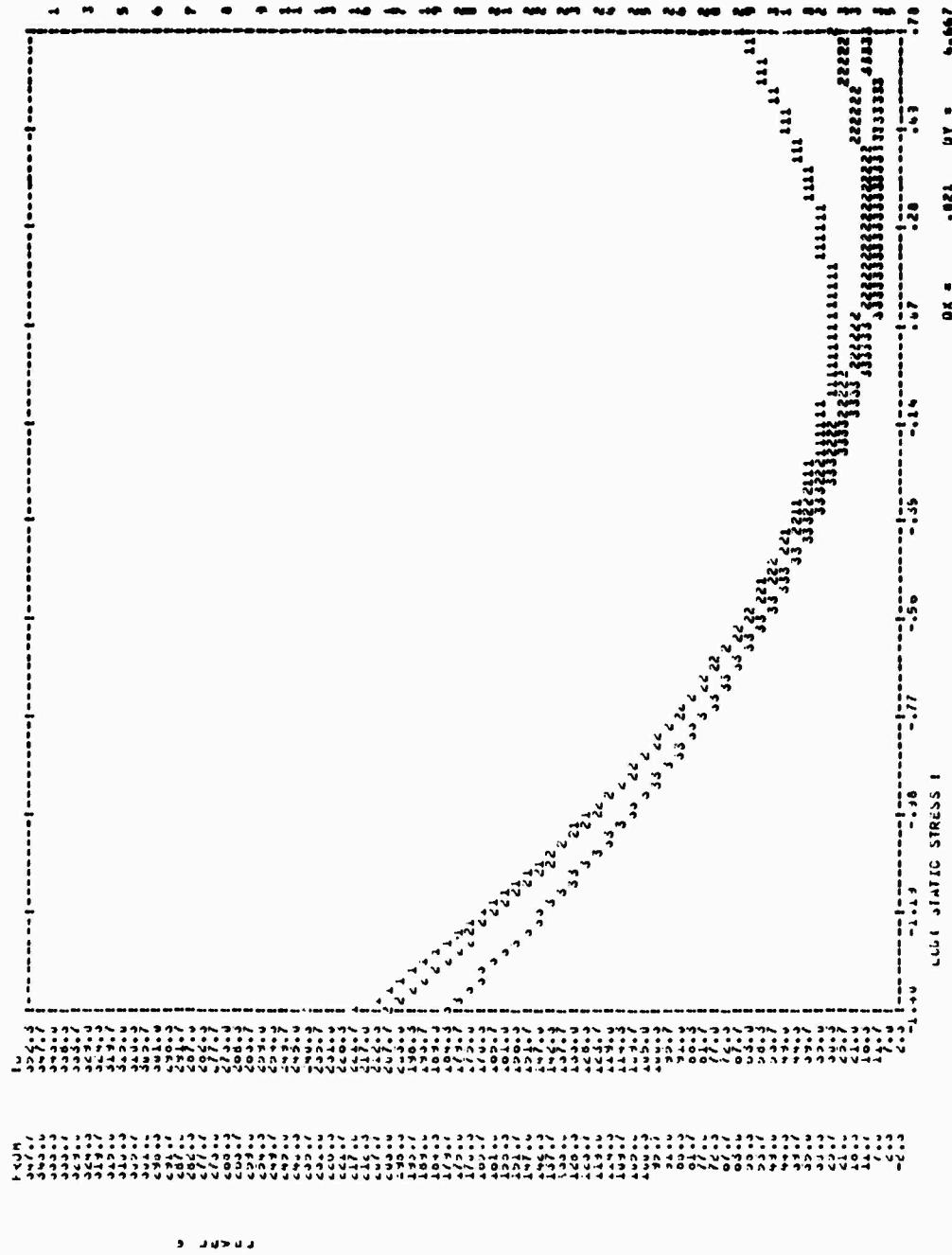


Figure E-1. -65°F, 12-inch drop height.

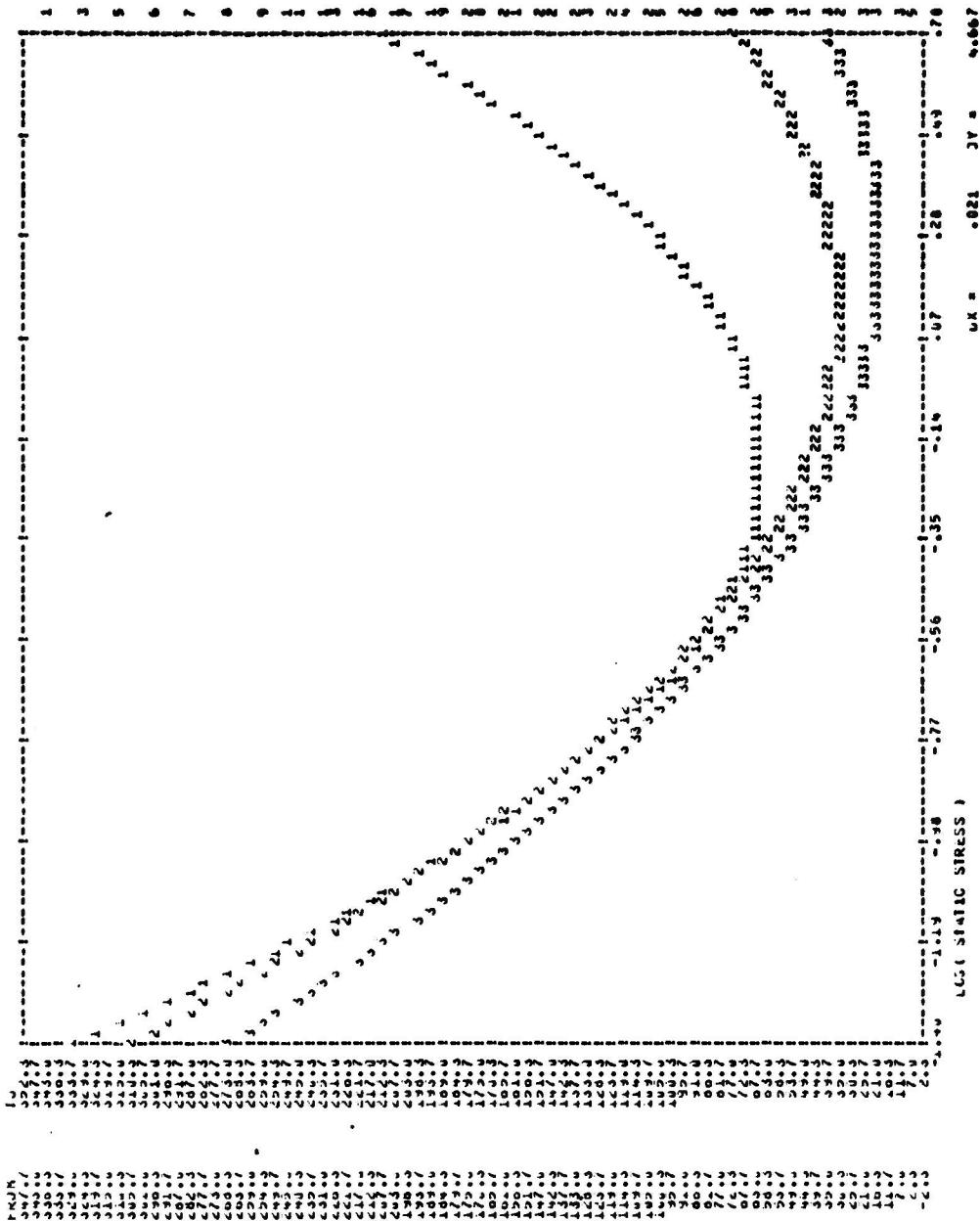


Figure E-2. -65°F, 24-inch drop height.

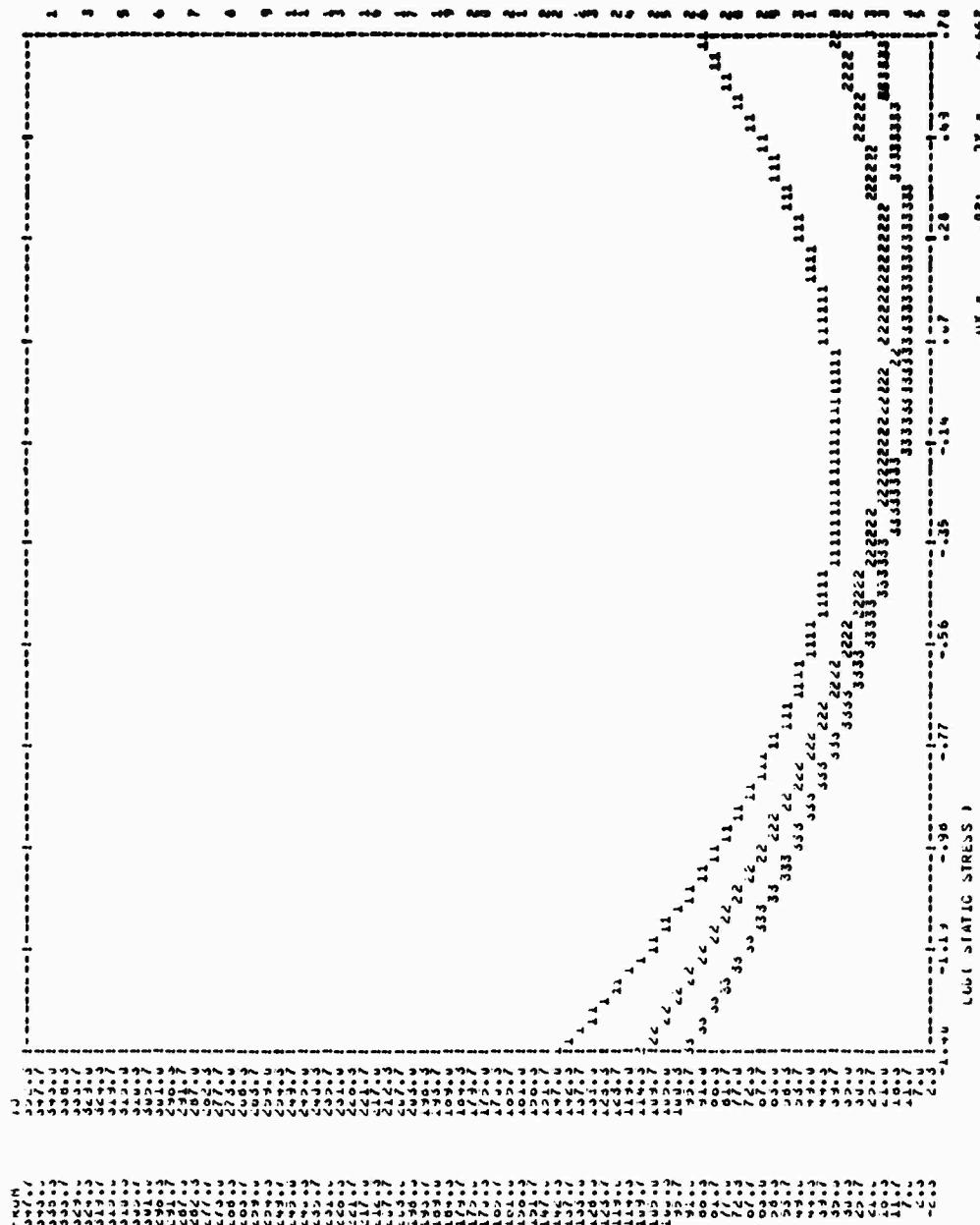


Figure E-3. 70°F, 12-inch drop height.

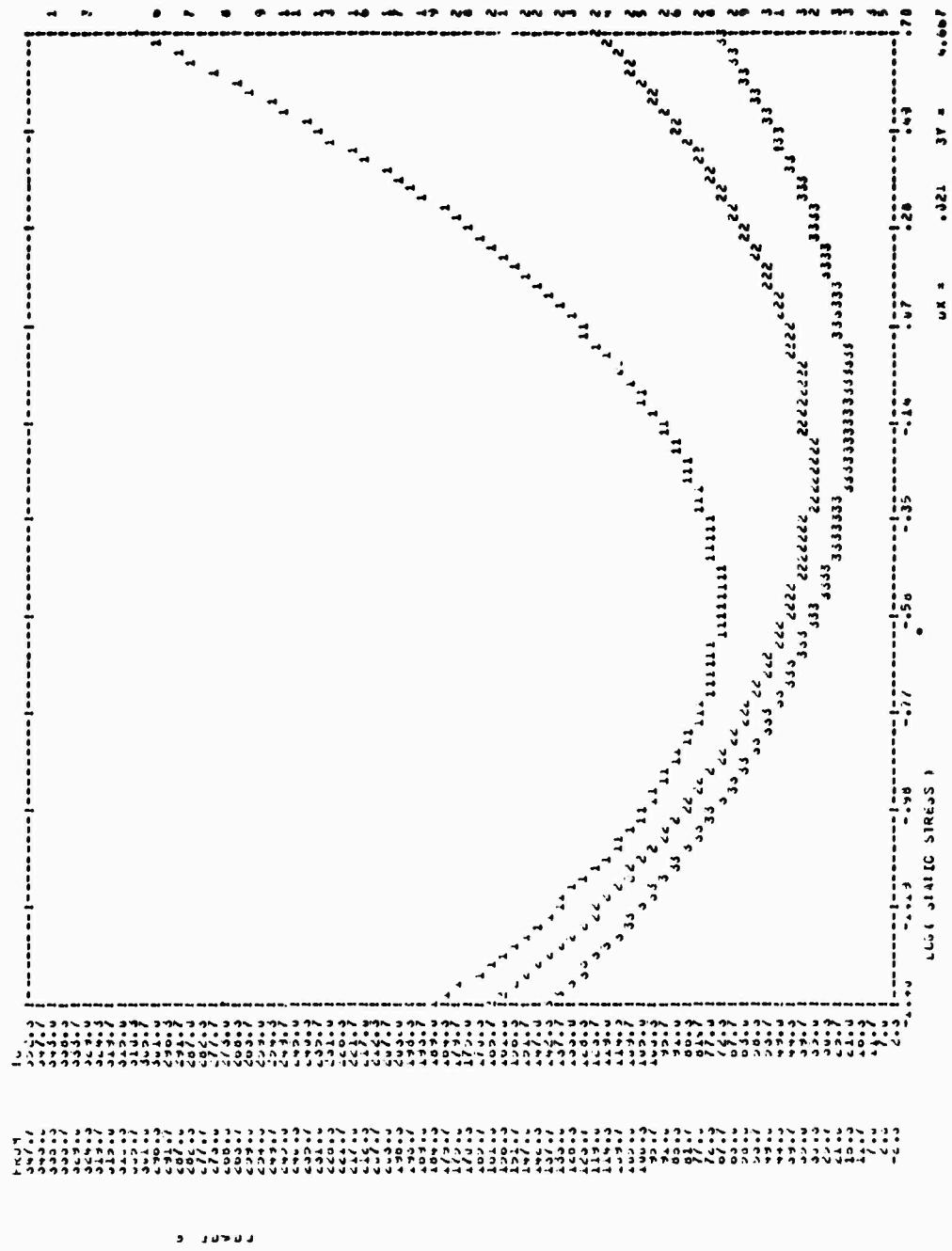


Figure E-4. 70°F, 24-inch drop height.

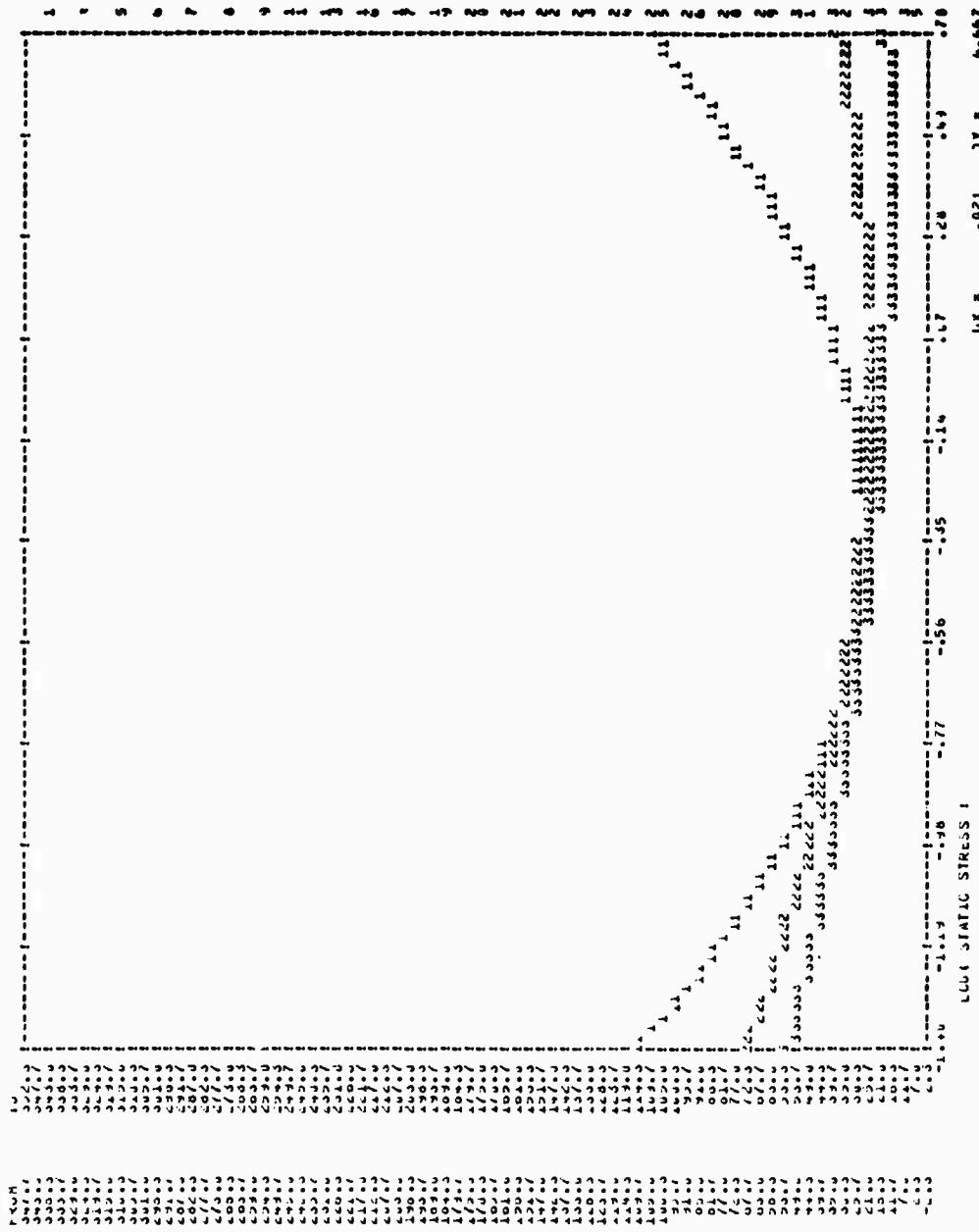


Figure E-5. 160°F, 12-inch drop height.

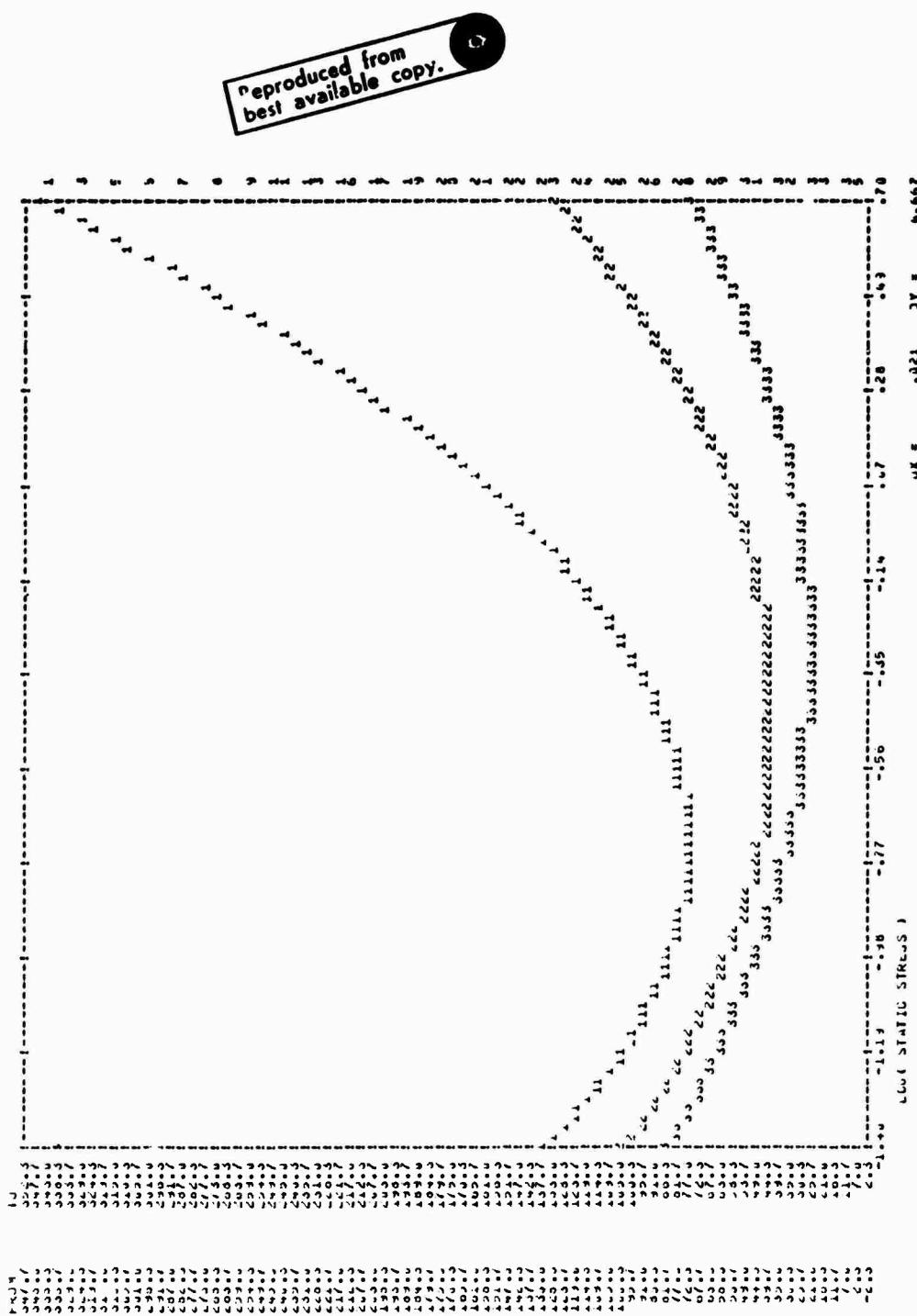


Figure E-6. 160°F, 24-inch drop height.

**Appendix F**

**TWENTY-SEVEN-INCH DROP HEIGHT MINICEL DATA,  
1, 2, AND 3 INCH THICK**

		0.04				0.08				0.10				0.20				0.80			
										Temperature (°F)											
						-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	Thickness
Replication	1	349	236	142	276	146	123	215	146	76	111	79	86	41	83	141					
	2	301	237	123	286	148	134	203	153	88	121	67	88	40	94	148	1"				
	3	383	247	105	292	142	142	207	120	85	96	68	79	40	90	149					
Replication	1	321	205	110	211	106	80	189	99	73	89	55	68	31	59	56					
	2	346	180	92	228	124	73	184	111	97	94	55	60	31	40	56	2"				
	3	370	191	120	205	93	94	138	98	88	98	55	61	35	49	52					
Replication	1	278	181	86	289	66	47	185	55	45	67	49	29	35	30	28					
	2	269	171	78	293	70	52	172	60	40	61	55	31	36	30	25	3"				
	3	264	215	86	271	68	50	177	56	53	53	53	29	36	32	23					
		1.00				1.50				1.50				2.60				3.00			
										Temperature (°F)								3.40			
						-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	Thickness
Replication	1	57	96	183	80	162	178	96	183	269	122	224	264	178	285	323	170	259	324		
	2	53	92	193	73	170	108	91	173	253	131	213	292	176	251	313	158	294	337	1"	
	3	38	99	189	85	168	184	106	172	258	132	211	296	170	278	308	167	297	284		
Replication	1	29	29	41	30	52	73	24	72	75	37	77	94	32	85	95	42	122	97		
	2	29	29	38	24	57	45	20	74	75	39	87	93	37	92	91	30	123	118	2"	
	3	17	35	47	24	56	76	25	79	77	40	76	90	40	97	91	38	120	119		
Replication	1	22	24	35	18	32	45	18	20	31	24	31	37	20	33	44	16	36	45		
	2	24	34	33	16	21	48	18	22	29	24	29	35	17	30	62	26	34	49	3"	
	3	23	20	26	17	30	48	18	22	27	25	26	38	18	31	62	15	31	45		

**Appendix G**  
**FOUR- AND FIVE-INCH THICK MINICEL DATA**

**MINICEL - 12 in. Drop Height**  
**STRESS LEVELS (PSI)**

		0.04				0.10				0.20				0.40				0.80							

## MINICEL - 12 in. Drop Height (Continued)

## STRESS LEVELS (PSI)

		STRESS LEVELS (PSI)											
		3.60			4.00			4.60			5.00		
		Temperature (°F)											
		-65	70	160	-65	70	160	-65	70	160	-65	70	160
1	9	10	13	5	12	13	9	12	13	6	13	12	Thickness
2	9	10	15	9	11	13	5	12	13	5	13	21	4"
3	8	12	12	8	13	9	6	13	14	7	13	21	
Replication	1	12	9	15	8	8	13	6	13	10	7	8	13
2	8	10	12	6	19	13	6	7	11	5	8	11	
3	7	9	10	7	9	13	7	8	13	8	10	13	5"

## MINICEL - 18 in. Drop Height

## STRESS LEVELS (PSI)

		STRESS LEVELS (PSI)														
		0.04			0.10			0.20			0.40			0.80		
		Temperature (°F)														
		-65	70	160	-65	70	160	-65	70	160	-65	70	160	Thickness		
1	235	120	61	148	58	41	80	31	26	39	22	21	24	17	19	
2	253	103	66	126	55	42	74	28	24	41	23	19	23	18	17	
3	206	91	66	139	55	35	71	30	25	37	23	19	25	20	16	
Replication	1	186	94	55	118	50	35	61	26	21	39	18	18	20	14	9
2	198	81	62	109	46	38	59	29	21	34	22	17	23	16	15	
3	178	92	55	115	47	35	72	27	22	37	19	18	23	15	14	5"

MINICEL - 18 in. Drop Height (Continued)

MINICEL - 24 in. Drop Height  
STRESS LEVELS (PSI)

RePLication		Temperature (°F)						Thickness			
		1.00		1.60		2.00		2.40		3.00	
-65	70	160	-65	70	160	-65	70	160	-65	70	160
18	21	21	18	19	25	17	20	26	19	23	28
20	19	24	18	20	23	16	20	26	18	26	22
23	22	23	16	22	24	16	26	25	17	24	26
1	21	15	18	15	15	19	14	18	21	16	21
2	23	15	20	14	15	19	14	14	20	13	19
3	21	17	17	15	14	19	13	18	21	14	16
1	21	12	18	12	16	21	12	18	24	18	21
2	23	14	18	14	16	20	13	19	22	14	18
3	21	12	17	12	15	18	14	16	20	12	16

## MINICEL - 24 in. Drop Height (Continued)

## STRESS LEVELS (PSI)

		Temperature (°F)				Thickness
		3.60	70	160	-65	
1	20	32	38	12	31	56
2	20	32	43	21	36	40
3	17	33	36	22	32	42
Replication						4"
1	14	20	24	12	19	29
2	17	18	30	15	23	25
3	12	21	28	30	25	30

## MINICEL - 30 in. Drop Height

## STRESS LEVELS (PSI)

		Temperature (°F)				Thickness
		0.04	70	160	-65	
1	274	122	75	132	56	47
2	250	126	74	151	60	48
3	233	106	83	130	57	45
Replication						4"
1	60	96	69	126	56	39
2	252	123	68	136	54	41
3	222	100	74	152	62	36

## MINICEL - 30 in. Drop Height (Continued)

## STRESS LEVELS (PSI)

		1.00		1.60		2.00		Temperature (°F)		2.40		3.00			
														Thickness	
-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	
1	25	27	29	18	27	35	15	29	35	23	36	44	24	39	42
2	23	36	30	21	38	36	15	32	38	24	33	43	24	43	43
3	23	24	30	21	31	33	18	30	32	19	32	41	25	39	42
Replication	1	30	18	25	18	16	22	22	18	20	12	19	23	15	22
2	26	21	21	17	19	26	15	18	24	12	15	25	18	24	28
3	23	13	23	19	18	20	12	25	22	16	20	26	16	20	32

Replication

		3.60		4.00				Temperature (°F)							
														Thickness	
-65	70	160	-65	70	160	-65	70	160	-65	70	160	-65	70	160	
1	23	40	55	29	51	61									
2	24	42	56	24	52	57									
3	41	42	58	32	47	57									
Replication	1	16	27	38	15	27	40								
2	15	28	39	15	32	46									
3	14	29	40	14	52	36									

Replication

**Appendix H**  
**PROGRAM LISTING OF CUSHION OPT**



```

PROGRAM SEARCH      74/74      OPT=1           FTN 4.0-2+7427e    11/25/74  15:44:34.

60          C HAS THIRD TEMPERATURE BEEN REACHED ?
C 200 IF( N .GE. 3 ) GO TO 500
C INCIDENT TEMPERATURE
N = N + 1
TP = T(N)

C INITIALIZE STATIC STRESS
C SS = SSMIN - OSS
C HAS MAXIMUM STATIC STRESS BEEN REACHED ?
100 IF( SS .GE. SMAX ) GO TO 100
C INCREMENT STATIC STRESS
SS = SS + CSS
C

65          CALL MODEL
IF( GL .GE. GLMAX ) GO TO 300
SSL(IN) = SS
TCC(IN) = TC

75          C HAS MAXIMUM STATIC STRESS BEEN REACHED ?
400 IF( SS .GE. SMAX ) GO TO 420
C INCREMENT STATIC STRESS
SS = SS + OSS
C

80          CALL MODEL
IF( GL .LE. GLMAX ) GO TO 400
420 SSL(IN) = SS
C MAKE RANGE TEST
IF( SSL(IN)-SSL(IN) .GE. 1.2 ) GO TO 200
GO TO 100

C 500 CONTINUE
PRINT 920, T(1), SSL(1), TCC(1), J=1,3
920 FORMAT(1X, TEMP, SSL, TCC, J=1,3)
IF( SSL(1) .EQ. 0.0 ) GO TO 510
TEST = SSL(1) - SSL(3)
IF( TEST .GE. 0.2 ) GO TO 523
510 PRINT 922
922 FORMAT(1X, RANGE TEST FAILOR)
IF( TC .GE. TCHAX ) PRINT 924
924 FORMAT(1X, USE OTHER MATERIAL)
IF( TC .GE. TCHAX ) GO TO 10
GO TO 101

520 CONTINUE
PRINT 926, TC, TYPEM, GLMAX, SSL(3), SSU(1)
926 FORMAT(1XF3.1*X INCHES OF *210/ * GIVES A*F5.0* G PROTECTION*/
* * USING A STATIC STRESS/* RAN OF F5.2* TO*F5.2*
C FLCT CURVES THAT PASS RANC TEST
CALL MPLOT
C

100         GO TO 10
800 CONTINUE
CALL SMXY( 0, 0 )
CALL USER ID( 25HQUICKPL TEKTRONICS TEST )
CALL TEKTEST
CALL END JOB
END

```



```

SUBROUTINE DYNAMIC_CUSHIONING
 74/74   CPT=1
 60      V(15) = TR3   * TCM
          V(36) = TR3   * TCM
          C* * * * *
          C
          C      COMPUTE DYNAMIC CUSHIONING FUNCTION
          C
          GL = CONST
          DO 100 J=1,NV
          I = INC(IJ)
          100 GL = GL + COEFF(IJ) * V(I)
          FEND

```

```

SUBROUTINE MPLCT      74/74   OPT=1           FTN 4.2+74278   11/25/74   15:42:41.

SUBROUTINE MELOT
COMMON TD, CH, TC, SS, GL, NVR, V(51), IN0151, COEFF(51), CCNST, NV
CC*4EN  CH, CA, CC, DRPH, GLMAX, SSL(3), SSU(3), TYPE(12)
DIMENSION X(101), Y(101,31), A(551), GG(1001), S(101)
C          HEAD(10), LEFT(5), EOTOM(81), RCP(31), T(3), LBE(1,8)
DATA LEFT /2*1H/, THG LEVEL, 2*1H /
DATA BOTCH/21H/10H STATIC STR, JMESS, 4*1H /
DATA HEAD(1)/25H DYNAMIC CUSHIONING CURVE /
C          T(1) = CH
C          T(2) = CA
C          T(3) = CC
C          HEAD(4) = TYPEM11
C          HEAD(5) = TYPE42
C          XMIN = ALDG10( 0.04 )
C          XMAX = ALDG10( 5.00 )
C          DX = ( XMAX - XMIN ) / 100.
C
C          XX= XMIN
C          DO 10 JP = 1, 101
C          X(JP)= XX
C          S(JP)= 10. ** XX
C          XX = XX * CX
C          GG(JP)= GLMAX
C
C          15 CONTINUE
C          00 30 K=1,3
C          TP = T(K)
C          00 29 JP = 1,101
C          SS = S(JP)
C          CALL MODEL
C          Y(JP,K) = GL
C
C          20 CONTINUE
C          31 CONTINUE
C
C          CALL SETRID( A, -76, XMIN, XMAX, 0.0, 350, )
C          CALL LARGRD( A, 1, 25H, 30, LEFT ), LOGI STATIC STRESS 1
C          CALL LARGRD( A, 2, 30, LEFT ), LOGI STATIC STRESS 1
C          CALL LARGRD( A, 3, 40, HEAD ), LOGI STATIC STRESS 1
C
C          CALL PLTRD( A, 1Hb, 101, X, GG )
C          CALL PLTRD( A, 1Hb, 101, X, Y1,1 )
C          CALL PLTRD( A, 1Hb, 101, X, Y1,2 )
C          CALL PLTRD( A, 1Hc, 101, X, Y1,3 )
C
C          ENCODE( 150, 934, LB ) CH, CC, DR04,
C          174 FORMAT( H=FS,0 DEGRES=4X, *A=FS,0 DEGREES=14), *C =FS,0,
C          1* DEGREE=314X, *DROP=HEIGHT =SF4.0* INCHES=X)
C          ENCODE( 120, 90, LAR1,5 ), TC, TYPE, GLPAK, SEL12, SEL11
C          50 90E FC, PHAT(F1,1) INCHES, JF *A1,A6, GIVES A-F5,* G PROTECTIC*5X,
C          1* USING A STATIC STRESS=9X,*RANGE OF FS,2* 10*FS,2.9X)
C          NY = 7E
C          OC 4,0 J=1,A
C          NY = NY - 1
C          CALL FEMGRD( A, 40, NY, 25, LAR1,J ) 1
C
C          40 CONTINUE

```

```

SUBROUTINE MPLOT      74/74   OPT=1           FTN 4.02+74278    11/25/74  19.42.41.

C
C     CALL PRINTPL( A, GLOUTPUT )
C
60   C     CALL SC4020
       CALL LABEL( 1, 35, BOTTOM, 5 )
       CALL LABEL( 1, 30, LEFT, 6 )
       CALL LABEL( 1, 50, HEAD, 3 )
       CALL PLOTA( S, GG, 0.0450, 0.00350, 101, 5, 4, 1, 100 )
JX = 420
JY = 900.
DO 4; J=1,8
     CALL RDOTC( JX, JY, 30, LAB(1,J) )
45   JY = JY - 25
DO 5; J = 1,3
     CALL PLOT3( S, Y11,K1, 101, -J )
     CALL NOTE( S(100), Y(098,K), 1, MCH(K) )
50   CONTINUE
     RETURN
END

```

```

SUBROUTINE TEKTEST      74774   OPT=1           FIN 4.2+74774   11/25/74  15.042.44.

SUBROUTINE TEKTEST
  DIMENSION X(10),Y(10),Z(10)
  DATA X1..* 2.0, 3.0, 4.0, 5.0, 6.0, 7.0, 8.0, 9.0, 10.0, /
        DATA Y130..*50., 70., 50., 60., 50., 70., 50., 60., 40., /
        DATA Z130..*100., 100., 100., 100., 100., 100., 100., 100., 100., /
        CALL QUICKPLT(X1..*2.0,HX-AXIS,6HTEST,X1..*TEST,(L,X1..*),
        * Y130..*Y11..*200,1)
        CALL QLICKFL(4,7,-1,0,-1)
        CALL PRINTV(4,6HXXX,500,500)
        CALL PRINTV(4,4H0000,0,0,0,0)
        CALL LINEV(4,0,500,500,500)
        CALL LINEV(4,500,0,500,500)
        CALL LINEV(4,0,600,600,500)
        CALL LINEV(4,600,0,600,600)
        RETURN
        END

```

19

15